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**FLIGHT VERIFICATION OF THE
ADVANCED FLIGHT CONTROL ACTUATION
SYSTEM (AFCAS) IN THE T-2C AIRCRAFT**



Rockwell International

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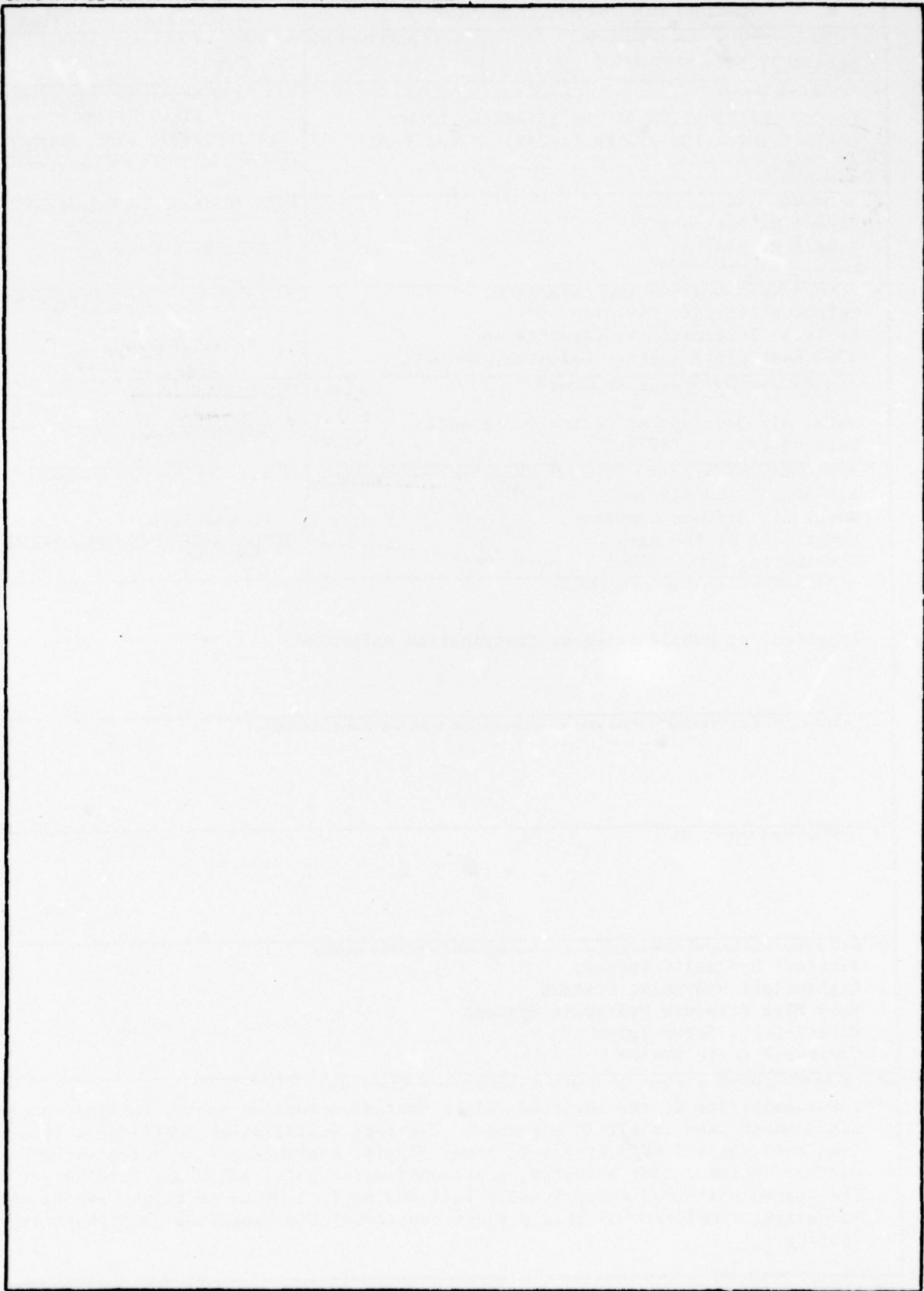
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EXECUTIVE SUMMARY

1.0 PURPOSE OF THE PROGRAM

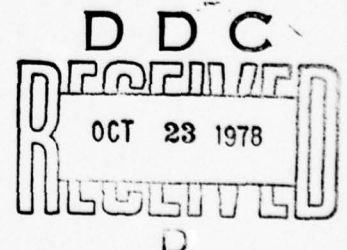
The complexity of aircraft flight control systems has increased year-by-year until present initial costs and required maintenance time are approaching prohibitive levels. This situation is due primarily to the design philosophy that improvements and refinements are best achieved by adding on accessories and/or components to proven, traditional systems. Broad new approaches and technologies involving advances in power generation, transmission, control, and actuation will be required to alleviate complexity in future Navy aircraft. The Advanced Flight Control Actuation System is a significant step in this direction.

2.0 BENEFITS TO THE NAVY

The Advanced Flight Control Actuation System (AFCAS) program introduces concepts for direct computer control of primary surface actuators. Computer processed signals are applied directly to primary actuators, eliminating traditional augmentation and secondary actuators with contaminant sensitive electrohydraulic servo valves. The actuator design employs a "building block" approach which standardizes various elements in the assembly. This modularization provides simplicity, hardware commonality, and permits the development of actuator classes. Combat survivability is improved through the use of localized hydraulic power supplies.

Adoption of AFCAS concepts will permit significant gains in several areas:

- Reduction in MMH/FH from 1.0 to 0.2
- Reduction in ground crew training
- Improved reliability and increased utilization
- Lower life cycle costs
- Improved vulnerability characteristics
- Simplified redundancy requirements



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3.0 INVESTIGATION PROCEDURE AND RESULTS

3.1 Background Information

The AFCAS program examined the feasibility of three separate concepts:

Control-By-Wire:	Computer processed signals are applied to force motor direct-driven flow control valves mounted on hydraulic actuators operating at 8000 psi (55 MPa).
Modular Actuator:	Actuators are designed with commonality features which permit the formation of "actuator classes" and the fabrication of single, parallel, or tandem actuators using modular "building blocks".
Localized Power:	Independent pump/reservoir hydraulic power packages provide 8000 psi fluid to actuators in specific, localized areas in the aircraft.

The program consisted of five phases. Phase I was a feasibility study which established that a direct-drive flow control valve, modular configured actuator, and localized power package could be readily integrated into a computer-operated, control-by-wire system. Efforts to confirm the practicality of AFCAS concepts were begun in Phase II with the design and fabrication of an engineering model control-by-wire, modular hydraulic servo actuator. Phase III involved conducting laboratory performance tests on the actuator. Major achievements accomplished in Phase III were:

- Successful operation of a direct electrical control muscle actuator for primary flight control surfaces.
- Use of building-block elements to assemble dual tandem, dual parallel, and single actuator configurations.
- Successful operation of a control-by-wire hydraulic actuator utilizing 8000 psi (55 MPa) operating pressure.
- Successful performance of a laboratory-type electronic drive unit which provided high immunity to circuitry failures.

An 8000 psi control-by-wire modular rudder actuator was designed and fabricated in Phase IV. The actuator was built for flight testing on a T-2C twin engine turbojet trainer. Phase V, reported herein, demonstrated the feasibility of AFCAS concepts by flight testing.

3.2 Flight Verification of AFCAS

3.2.1 Test Installation - The directional control system in a T-2C airplane was changed to a full-powered control-by-wire test installation containing:

- Hydraulic rudder actuator
- Electronic drive unit
- Localized hydraulic power unit
- Force transducer

The existing hydraulic system was altered to operate at two pressure levels: 3000 psi (21 MPa) and 8000 psi (55 MPa). Engine driven pumps powered the 3000 psi system in the usual manner. A localized motor/pump unit was added to power the rudder actuator. The modified system functioned like the original T-2C directional system except the rudder was hydraulically powered instead of manually operated.

The T-2C cable system between the rudder pedals and rudder was changed to incorporate the control-by-wire test installation. The rudder pedal cables were attached to a sector which operated a force transducer. Force on the pedals was converted to a proportional electric voltage from the transducer. This command signal was conditioned by an electronic drive unit which powered a torque motor on the rudder actuator. The torque motor in turn operated a single stage flow control valve on the actuator.

3.2.2 Preflight Tests - A laboratory setup integrating components to be installed on the T-2C was assembled and performance tested. Six 1-1/2 hour simulated flights were conducted on the laboratory setup to verify reliability. Following system installation on the aircraft, hangar tests were conducted using comprehensive checkout procedures. A ground demonstration test was conducted on the T-2C to simulate a one hour flight from take-off to landing, and provide a means to final check system operation and instrumentation.

3.2.3 Flight Tests - The primary objective was to verify the feasibility of AFCAS concepts by flight testing a control-by-wire, direct drive actuation system powered by a localized motor/pump unit. The flight plan was designed to determine directional control characteristics at several altitudes up to 30,000 feet (9.1 km) and various speeds up to 340 knots (174 m/s). The first two flights were dedicated to confirming satisfactory operation. Subsequent flights were scheduled to evaluate system performance while accumulating 10 flight hours. Two pilots participated in the program.

Both test pilots stated that performance of the AFCAS installation was completely satisfactory. Comments made by the pilots concerning their flights were:

- The AFCAS installation worked exactly as designed
- No malfunctions occurred
- System pressure was steady
- Hydraulic fluid temperatures were normal
- Directional control response was judged to be superior to the production T-2C
- Pilot adaptation to "force control" of the rudder was quickly and easily acquired. Reaction of the aircraft provided the clues to close the loop.

4.0 DIGITAL CONTROL OF AFCAS

The AFCAS concept is intended for application to automatic, computer operated flight control systems. The AFCAS flights did not demonstrate the full performance capabilities of the test hardware since the T-2C did not have computer operated controls. Company funded investigations at the Columbus Aircraft Division have verified the feasibility of controlling AFCAS actuators directly by a digital computer.

A laboratory setup was assembled which interfaced AFCAS direct-drive actuation circuits with a programmable digital processor. The processor was used for closed loop control of a direct-drive dual tandem actuator built early in the AFCAS program. The computer was programmed to generate the error signal in four formats: (1) dc analog, (2) pulse-width modulation, (3) bang-bang, and (4) time dwell modulation. Frequency response was determined for all four modes. Simulated failure tests confirmed that digital control signals are compatible with the automatic failure-compensation features inherent in the AFCAS electronic drive unit.

5.0 CONCLUSIONS

The feasibility of the AFCAS concept was demonstrated in a T-2C aircraft. The AFCAS installation functioned exceptionally well. Successful completion of this program confirmed the results of prior analyses and laboratory investigations. The ease with which flight testing was accomplished verified that direct-drive, control-by-wire 8000 psi actuation systems can be designed, fabricated, and maintained without special techniques or state-of-the-art advances.

6.0 RECOMMENDATIONS

The test installation was an analog control-by-wire system; the AFCAS concept is intended for application to digital control-by-wire systems. The Columbus Aircraft Division has confirmed by laboratory testing that AFCAS components are compatible with digital control-by-wire components. Therefore, it is recommended that the AFCAS test system currently installed on the T-2C be modified by the addition of a micro-processor and that additional flight testing be conducted. This will provide the Navy with an economical approach to demonstrate, in flight, advantages of the direct-drive features of AFCAS with computer control.

A second recommendation is concerned with the direct-drive control module (force motor and spool/sleeve valve). The motors and valves procured for AFCAS projects were designed for concept verification only, and were not intended to be production configurations. LHS and AFCAS technology have progressed sufficiently that effort should now be directed toward optimized designs which can be integrated into future production applications.

PREFACE

This report documents research conducted by the Columbus Aircraft Division of Rockwell International Corporation, Columbus, Ohio, under Contract N62269-76-C-0201, with the Naval Air Development Center, Warminster, Pennsylvania. Technical direction was administered by Mr. T. Jansen, Program Engineer, Aero-Mechanical Branch, Aircraft and Crew Systems, Naval Air Development Center (Code 6013), and Mr. J. Schonowski, Flight Controls Systems Unit, Mechanical Equipment Branch, Naval Air Systems Command (AIR-530311C).

This report discusses flight testing of a control-by-wire, direct-drive, 8000 psi (55 MPa) actuation system in the yaw axis of a T-2C aircraft. This work was related to tasks performed under Contracts N62269-72-C-0108, N62269-73-C-0405, and N62269-75-C-0311.

Acknowledgement is given to the following for their participation on this project:

Mr. D. Cahill	Design Engineer
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Mr. R. Dixon	Manager, Test Vehicle Maintenance
Mr. R. Cockburn	Test Pilot
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Discussions in this report of components supplied by various manufacturers shall not be construed as either an endorsement or criticism of any component. The government incurs no liability or obligation to any supplier from the information presented herein.

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1.0 INTRODUCTION

1.1 BACKGROUND INFORMATION

The development of Advanced Flight Control Actuation Systems (AFCAS) for next generation aircraft has been a joint undertaking by the Navy and Rockwell International Corporation since 1972. This report presents the results of flight testing a subsystem to verify concepts, laboratory evaluations, and component designs developed in prior phases of the AFCAS program.

The complexity of flight control systems has increased year-by-year until present initial costs and required maintenance time are approaching prohibitive levels. This situation is due primarily to the design philosophy that improvements and refinements are best achieved by adding on accessories and/or components to proven, traditional systems. Broad new approaches and technologies involving advances in power generation, transmission, control, and actuation will be required to alleviate complexity in future Navy aircraft. The Advanced Flight Control Actuation System is a significant step in this direction.

Phase I of the AFCAS program was a study which examined the feasibility of three separate concepts:

Control-By-Wire:	Computer processed signals are applied to force motor direct-driven flow control valves mounted on hydraulic actuators operating at 8000 psi (55 MPa).*
Building-Block Actuator:	Actuators are designed with commonality features which permit the formation of "actuator classes" and the fabrication of single, parallel, or tandem actuators using modular "building blocks".
Localized Power:	Independent pump/reservoir hydraulic power packages provide 8000 psi (55 MPa) fluid to actuators in specific, localized areas on the aircraft.

*The metric equivalent of 8000 psi is 55,158,000 pascals or 55 megapascals (55 MPa). This newly adopted method of defining pressure is based on the International System of Units (SI) and has been approved by DOD, reference ASTM E 380-76, Standard for Metric Practice, dated 19 January 1976. All dimensions in this report have metric equivalents given in parentheses except where unnecessary repetition would occur or when a general note is used on illustrations. A summary of metric conversions is given on page 106.

Phase I established that a direct-drive flow control valve, modular configured actuator, and localized power package could be readily integrated into a computer-operated, control-by-wire system. Adoption of AFCAS concepts should enhance flight control system maintainability, reliability, combat survivability, and lower initial costs, Reference 1.

Efforts to confirm the practicality of Phase I concepts were begun in Phase II with the design and fabrication of an engineering model, 8000 psi (55 MPa), control-by-wire, modular configured aircraft type hydraulic servo actuator, Reference 2. Electrical inputs were applied to force (torque) motors employing cobalt samarium permanent magnets. Motor output was connected directly to single stage spool/sleeve type flow control valves. The force motors and flow control valves could be integrated into dual tandem, dual parallel, or single actuator configurations.

Phase III involved conducting laboratory performance tests on the engineering model actuator(s) built in Phase II, Reference 3. Static and dynamic tests were conducted on the force motors, motor/valve subassemblies, electronic drive unit, and actuator assemblies including dual system tandem, dual system parallel, and single system configurations. The dual tandem actuator was tested under load. Major achievements accomplished in Phase III were:

- Successful operation of a direct electrical control "muscle" actuator for primary flight control surfaces.
- Use of building-block elements to assemble dual tandem, dual parallel, and single actuator configurations.
- Successful operation of a control-by-wire hydraulic actuator utilizing 8000 psi operating pressure.
- Successful performance of a laboratory type electronic drive unit which provided high immunity to circuitry failures.

In Phase IV, an 8000 psi (55 MPa) control-by-wire, modular rudder actuator was designed and fabricated for future flight testing on a T-2C airplane, Reference 4. Actuator design criteria were based on T-2C aerodynamic considerations, envelope constraints, and single system hydraulics. Actuator output was commanded by a single stage spool/sleeve valve driven directly by a permanent magnet force motor. The force motor was to be powered by an electronic drive unit which received inputs from a force transducer in the rudder system and position transducers on the actuator. A localized hydraulic power unit was planned to supply 8000 psi (55 MPa) pressure for the rudder actuator.

1.2 OBJECTIVE

The objective of Phase V was to design, fabricate, and test a subsystem to verify the feasibility of the Advanced Flight Control Actuation System - Building Block concept. The test system was to be installed in a T-2C twin engine turbojet trainer.

1.3 TECHNICAL APPROACH

The directional control system in a T-2C airplane was changed to a full-powered control-by-wire test installation containing:

- Hydraulic rudder actuator
- Electronic drive unit
- Localized hydraulic power unit
- Force transducer

The existing hydraulic system was altered to operate at two pressure levels: 3000 psi (21 MPa) and 8000 psi (55 MPa). Engine driven pumps powered the 3000 psi system in the usual manner. A localized motor/pump unit was added to power the rudder system which was formerly operated manually by the pilot. The original 3000 psi and newly added 8000 psi systems shared the existing reservoir and return lines. The T-2C electrical system was altered to power the localized motor/pump unit and electronic drive unit. The modified system functioned the same as the basic T-2C system except the rudder was hydraulically powered instead of manually operated.

The original cable system between the rudder pedals and rudder was changed to incorporate the control-by-wire test installation. The rudder pedal cables were attached to a sector which was prevented from rotating by a force transducer. Force on the pedals was converted to a proportional electric voltage from the transducer. This command signal was conditioned by an electronic drive unit which powered a torque motor on the rudder actuator. The torque motor in turn operated a single stage flow control valve on the actuator.

The direct-drive, 8000 psi rudder actuator designed and fabricated in Phase IV was modified to incorporate a bypass valve. This device allowed the rudder to seek the trail position if system pressure were lost. In the event of a "hard-over" type electronic failure, the pilot could permit the rudder to trail by turning the 8000 psi motor/pump unit "off".

The electronic drive unit was designed, fabricated, and packaged to be a flightworthy assembly. The unit had dual channels with sub-circuits which were dualized. The circuitry was designed with redundancy features which provided high immunity to component failures.

Requirements were established for an 8000 psi localized hydraulic power supply. The pump used was the same unit employed for flight testing in the Lightweight Hydraulic System (LHS) development program, References 5 through 13, except delivery was reduced to match rudder actuator flow rates and to lower input power requirements. The pump was mated to an off-the-shelf, aircraft type 28 volt DC motor.

The force transducer incorporated in the test system was designed specifically for this application. The transducer utilized two linear variable differential transformers mounted in series.

All major components in the test installation were assembled in the laboratory for integration testing. Investigations were made to determine if detrimental pressure oscillations or surges were present. Motor current and system heat rejection were measured. Frequency response tests were conducted on the actuator/system. Nine hours of simulated flight testing were performed to evaluate the endurance capability of system components.

The test system was installed in a bailed T-2C with instrumentation for monitoring pressures, flows, temperatures, etc. Standard parameters such as air speed, altitude, engine RPM, etc., were also instrumented. Flight data were collected by photorecorder and telemetry systems.

Procedures were established for system checkout, ground demonstration, and flight testing. Approximately ten hours of flight time were logged on the test system at various altitudes and airspeeds. Pilot observations and instrumentation data were used as a basis for evaluating the AFCAS installation.

2.0 T-2C AIRPLANE

2.1 GENERAL DESCRIPTION

The T-2C "Buckeye" is built by the Columbus Aircraft Division of Rockwell International Corporation. The Buckeye is a two-place, subsonic trainer powered by twin turbojet engines. The aircraft is designed for both land and carrier based operations. Distinguishing features include wide-track tricycle landing gear, straight tapered wings, and low slung intake ducts, Figure 1.

The T-2C is used as a basic trainer for military pilots, and is equipped for cross-country flight, night flying, and low altitude, high speed navigation exercises. Maximum level flight speed of the Buckeye is 465 knots (239 m/s) at 15,000 feet (4.6 km); the service ceiling is 45,000 feet (13.7 km). Take-off and landing speeds are in the range of 95 to 110 knots (49 to 57 m/s). A typical take-off gross weight is 13,000 pounds (5900 kg).

Dual power sources are provided for the electrical, hydraulic, and air conditioning systems. The flight control system includes hydraulic full-powered ailerons, a boosted elevator, and an electric trim system; rudder operation is manual. The aileron and elevator actuators are part of mechanical linkage connecting the pilot's stick to the control surfaces. Thus, in the event of a hydraulic system malfunction, control of the aircraft can be accomplished manually.

2.2 HYDRAULIC SYSTEM

The T-2C has a 3000 psi (21 MPa), Type II (-65 to +275°F) (-54 to +135°C) single hydraulic system. Two pumps, one on each engine, provide power to operate the landing gear, speed brakes, arresting hook, aileron actuator, and elevator boost package. The pumps are constant pressure, variable delivery, axial piston designs. Each pump is capable of delivering 4.9 gpm (18.5 L/m) at 7800 rpm. Hydraulic fluid (MIL-H-5606) is supplied to the pumps by an air/oil type reservoir pressurized by engine bleed air. Fluid cleanliness is maintained by 5 micron absolute filters.

One pump can adequately handle all flow demands. However, if supply pressure should drop below 1800 psi (12 MPa), a priority valve is used to insure operation of the aileron and elevator actuators. A cockpit controlled shutoff valve is installed in the aileron/elevator subsystem to permit simulating loss of power for training purposes. The landing gear and arresting hook can be lowered and locked by gravity, if desired. The wheel brakes have an independent hydraulic system.



FIGURE 1 THE T-2C "BUCKEYE" TRAINER

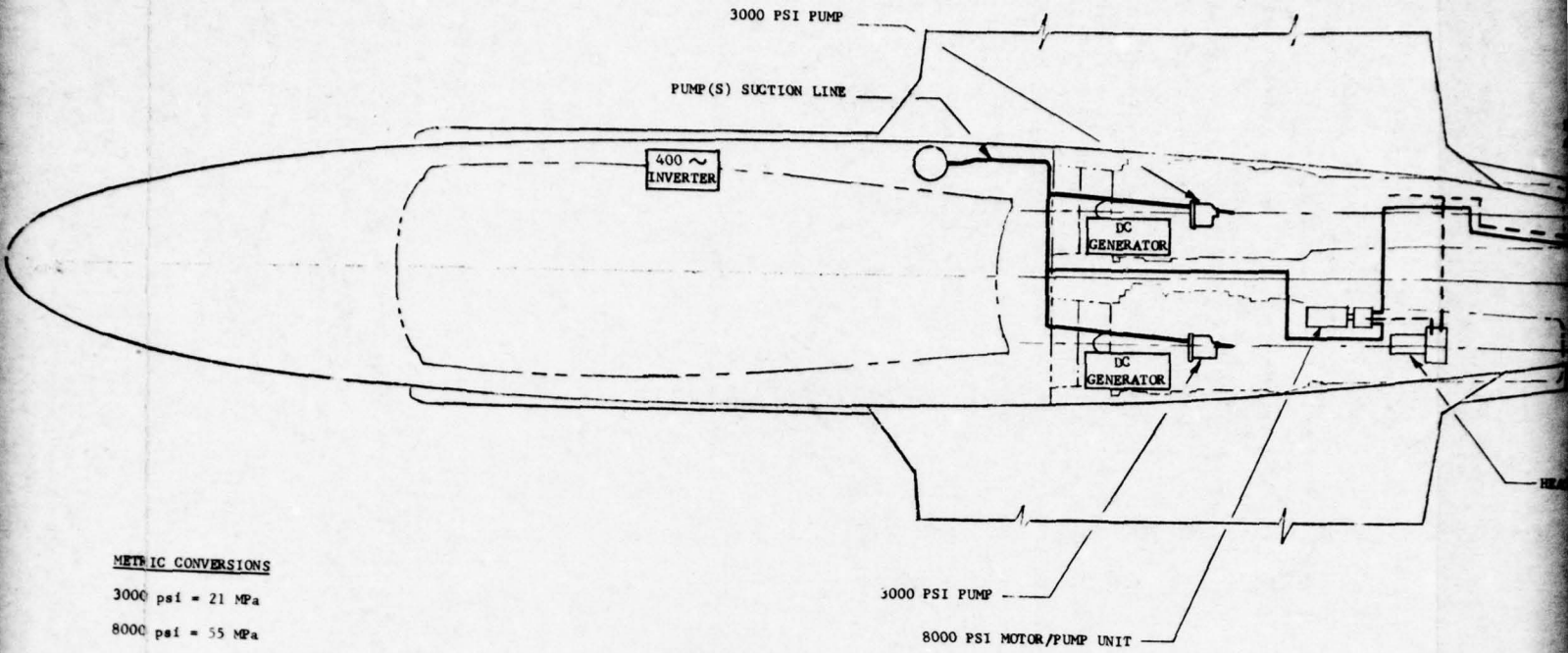
2.3

ELECTRICAL SYSTEM

Electrical power is supplied by two 28 volt DC 300 ampere starter-generators, one mounted on each engine. The generators are connected for parallel operation and power the primary bus. Output voltages are regulated for varying loads and engine speeds.

Two nickel-cadmium 24 volt re-chargeable batteries are used for engine starting and emergency DC power. The batteries are normally connected in parallel, but are used in series for engine starting.

A portion of the 28 volt DC power is converted to 115 volt 400 Hz AC power by two rotary inverters. Inverter No. 1 produces 500 volt-amperes for instruments; inverter No. 2 generates 1500 volt-amperes for avionics and serves as a backup source for instrument power.



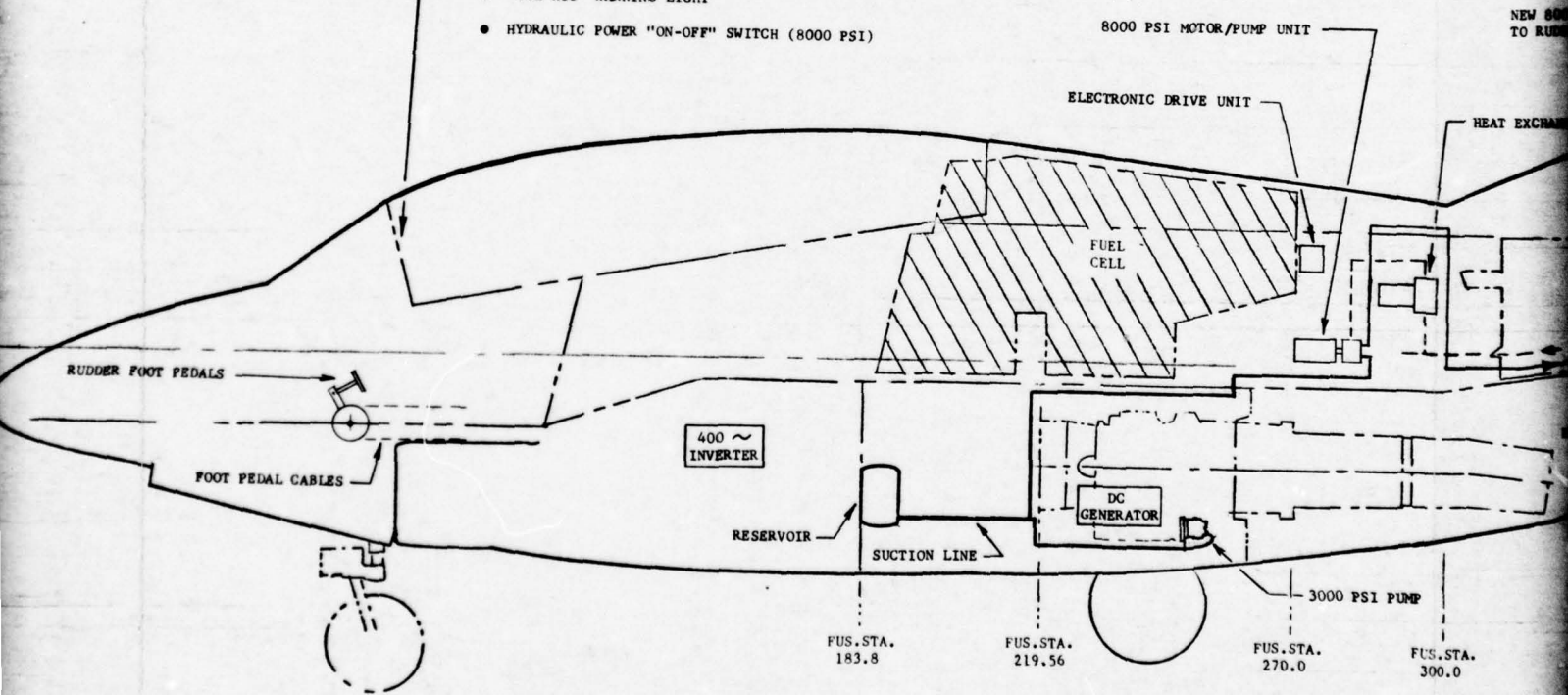
METRIC CONVERSIONS

3000 psi = 21 MPa

8000 psi = 55 MPa

NEW COCKPIT INSTRUMENTATION

- HYDRAULIC PRESSURE INDICATOR (8000 PSI)
- "OIL HOT" WARNING LIGHT
- HYDRAULIC POWER "ON-OFF" SWITCH (8000 PSI)



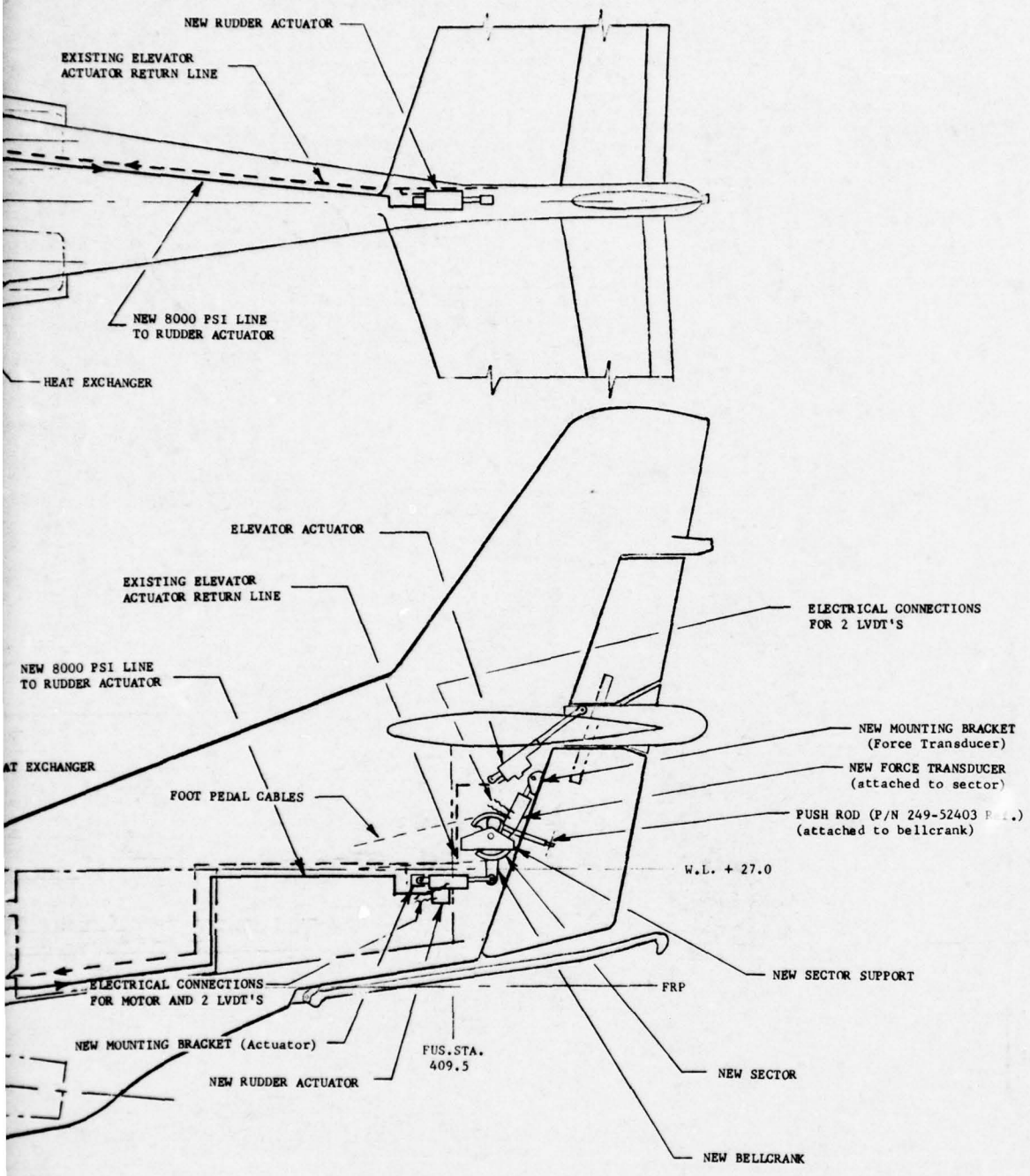


FIGURE 2 AFGAS TEST INSTALLATION

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3.0 AFCAS TEST INSTALLATION

3.1 GENERAL DESCRIPTION

The directional (rudder) system in a bailed T-2C (BuNo. 152382) was changed from a manual to a full-powered control-by-wire system for the AFCAS program. Principal components in the test installation were:

- Hydraulic rudder actuator
- Electronic drive unit
- Localized hydraulic power unit
- Force transducer

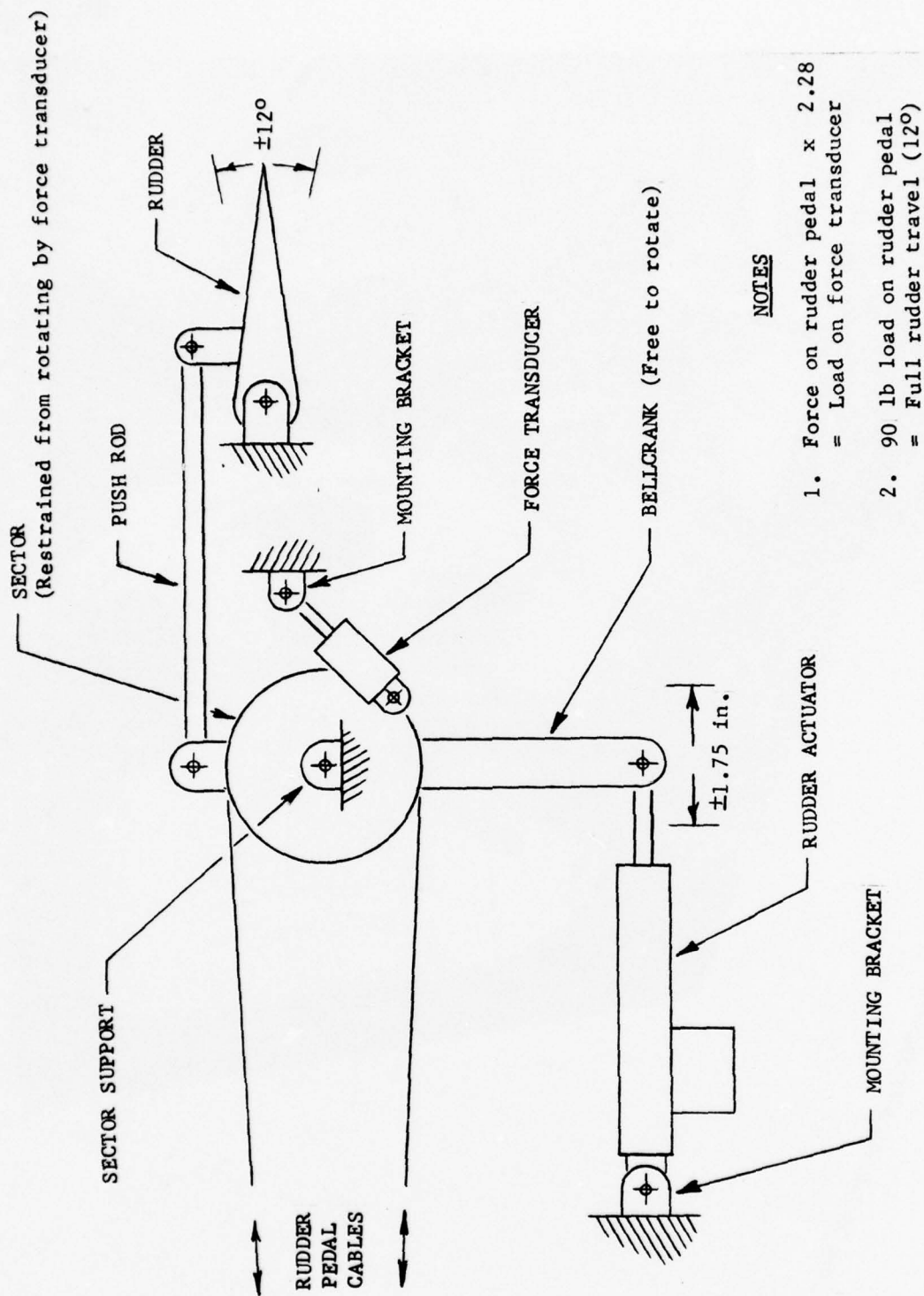
Modifications required in the T-2C to accommodate the new installation are shown on Figure 2. Details of the modifications are discussed in the following sections under four general headings: mechanical system, hydraulic system, electrical system, and instrumentation.

3.2 MECHANICAL SYSTEM

Elements of the mechanical system are listed below and depicted schematically on Figure 3. The installation is shown pictorially on Figure 4; details are given on Figure 5.

<u>Part No.</u>	<u>Description</u>
8691-524001-011	Sector assembly
8691-524001-013	Sector support
8691-524001-021	Bellcrank assembly
8691-524001-041	Rudder actuator mounting bracket
8691-524001-053	Force transducer mounting bracket

The T-2C rudder has a travel of $\pm 25^\circ$. For safety reasons, rudder travel was reduced to $\pm 12^\circ$ in the test installation by limiting actuator stroke. This permits the pilot to land safely with a "hard-over" rudder, opposite engine out, and three knot cross-wind.



NOTES

1. Force on rudder pedal x 2.28
= Load on force transducer
2. 90 lb load on rudder pedal
= Full rudder travel (120°)
3. Metric Conversions:
in. x 2.54 = cm
lb x .454 = kg

FIGURE 3 SCHEMATIC DIAGRAM OF MECHANICAL SYSTEM

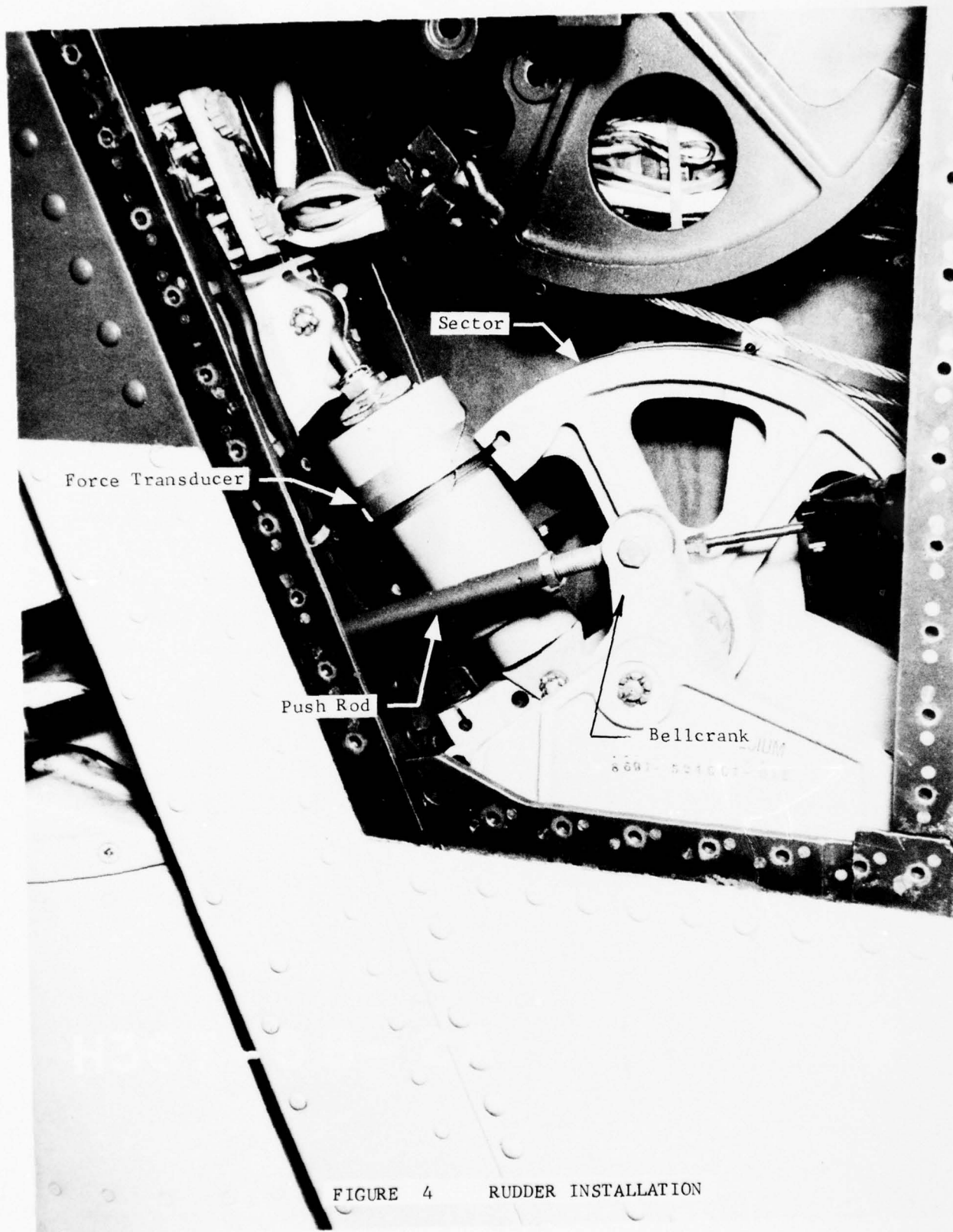


FIGURE 4 RUDDER INSTALLATION

8881-254001

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F
E
D
C
B
A

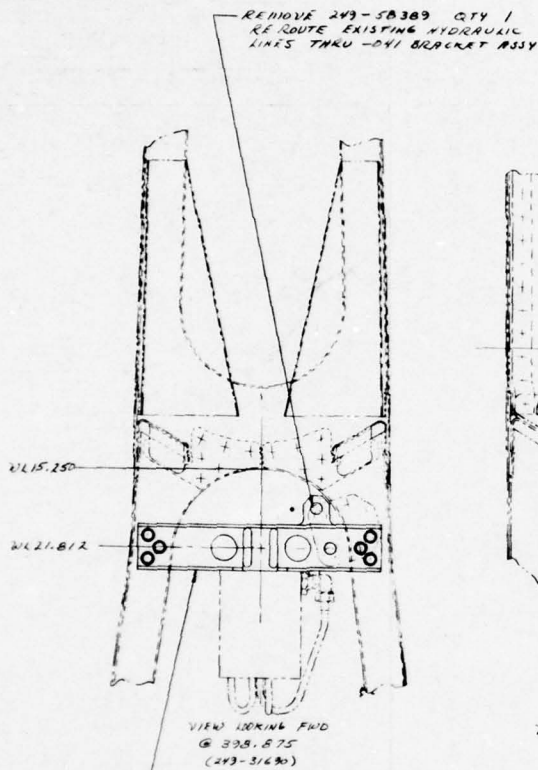
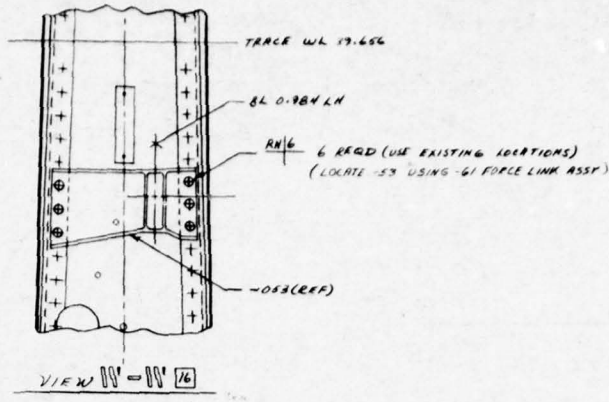
24

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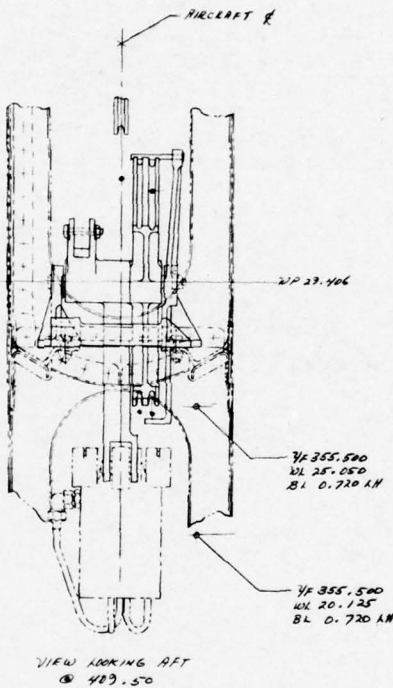
22

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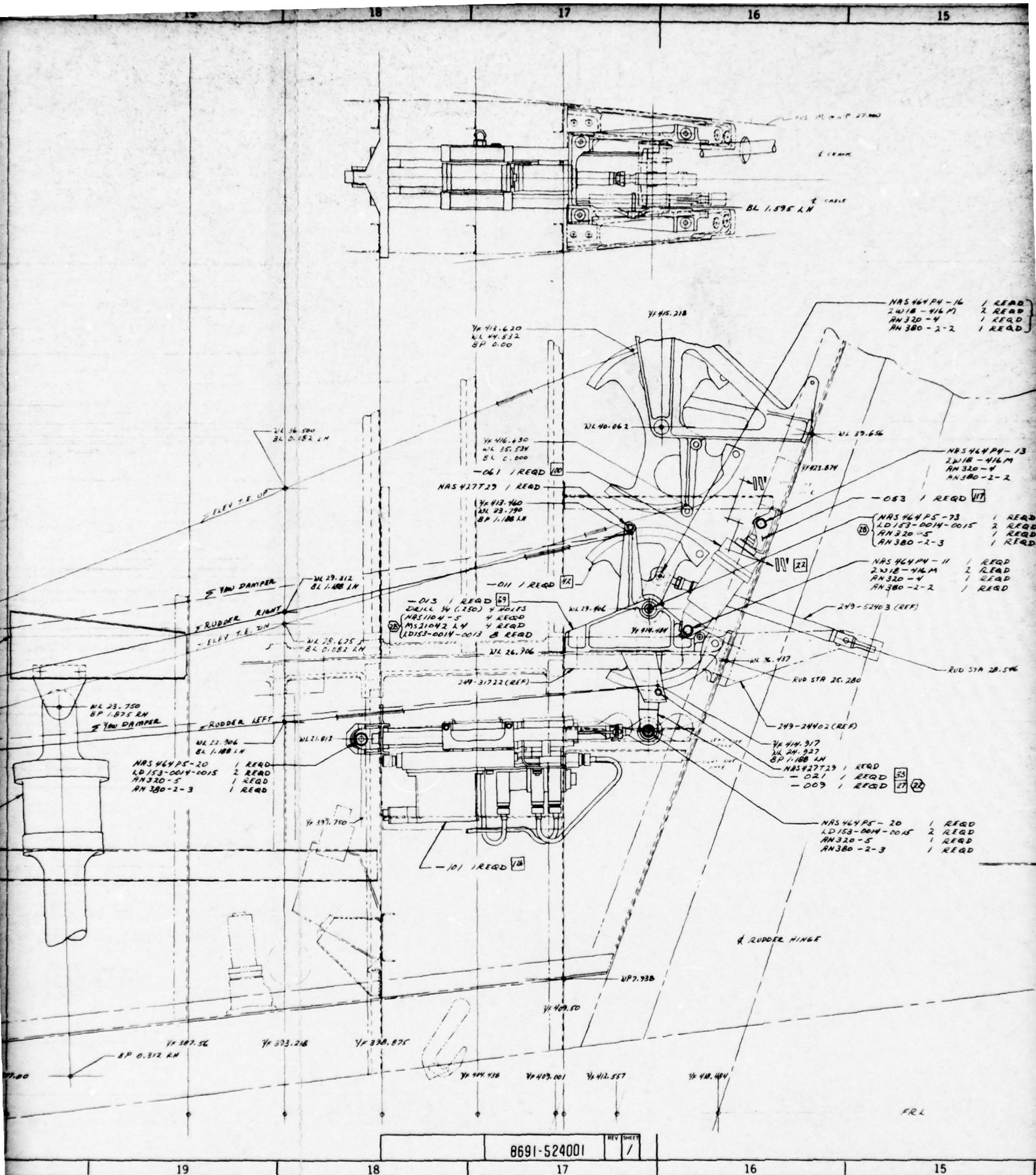
-041 1 REQD 53
DRILL #10 (.1335) 6 HOLES
TO MATCH -041
R43-5 6 REQD
LD 153-0016-1003 6 REQD
LD 153-0016-2003 6 REQD
MS21043-3 6 REQD



249-31731(REF)

249-356 001(REF)

YF 277.00



READ (INSTALL FROM LH SIDE)
READ
READ
READ

- NOTES: UNLESS OTHERWISE NOTED

LAST VIEW USED 11-11
LAST CODE USED (21)

[illegible]

ALTERED ITEM DRAWING

FIGURE 5

REVISIONS									
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635									

The relationship between pedal force and rudder travel is approximately 7.5 lb/deg (33.3 N/deg) of rudder movement or 90 lb (.4 kN) for full travel (12°). Pedal displacement was small, approximately .50 in. (13mm), since the force transducer length changed only 0.025 in. (0.63mm) for full rudder travel (pedal displacement was due primarily to cable stretch). In the original manual control system, pedal displacement was approximately 4 in. (10.2 cm) for full rudder travel (25°).

The maximum hinge moment normally applied to the T-2C rudder is based on pilot strength and is 2200 lb-in (249 N-m). Maximum rudder deflection a pilot can achieve thus depends on air loads present. The AFCAS rudder actuator can develop 13,000 lb-in (1470 N-m). Because of the limited rudder deflection (12° max.) the high moment capability of the rudder actuator required only minor adjustment in the T-2C flight envelope to assure safety.

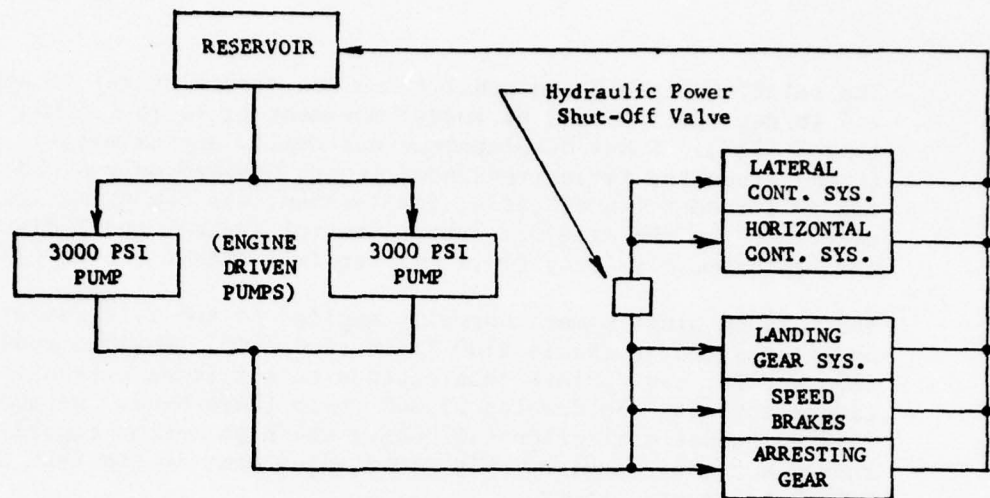
3.3 HYDRAULIC SYSTEM

3.3.1 System Description

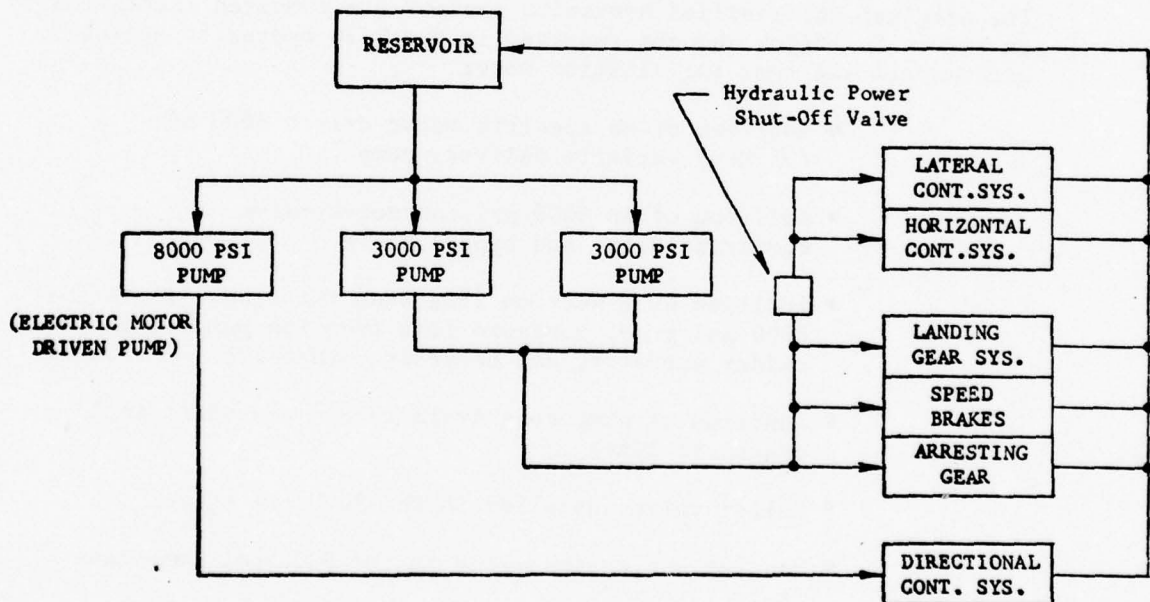
The original and modified hydraulic systems are compared schematically on Figure 6. Major changes required in the T-2C hydraulic system to accommodate the test installation were:

- Addition of an electric motor driven 8000 psi (55 MPa) variable delivery pump
- Addition of an 8000 psi control-by-wire rudder actuator and bypass valve
- Addition of a suction line from the reservoir to the 8000 psi pump, pressure line from the pump to the rudder actuator, and actuator return line
- Addition of pump case drain return and shaft seal overboard lines
- Relief valve installed in the 8000 psi system
- Heat exchanger installed in the 8000 psi pump case drain line

The modified system is shown schematically on Figure 7; 8000 psi components are listed on Table I. The 3000 psi (21 MPa) and 8000 psi (55 MPa) systems shared a common reservoir and common return lines. All major components, except for the rudder actuator, were located in the fuselage compartment above the engines. Plumbing details are given on Figure 8.



ORIGINAL 3000 PSI SYSTEM



MODIFIED HYDRAULIC SYSTEM

FIGURE 6 ORIGINAL AND MODIFIED HYDRAULIC SYSTEMS

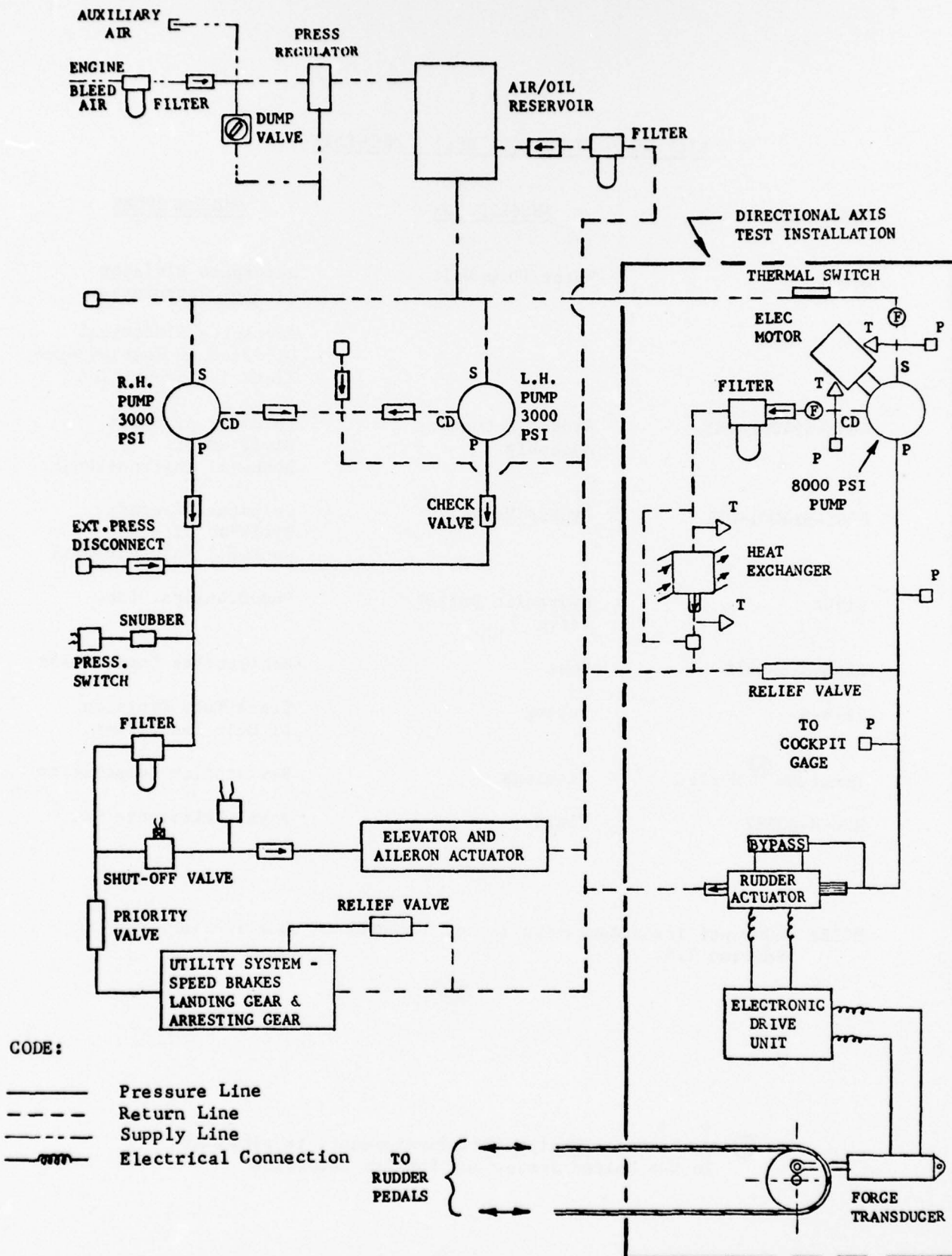


FIGURE 7 SCHEMATIC DIAGRAM OF MODIFIED HYDRAULIC SYSTEM

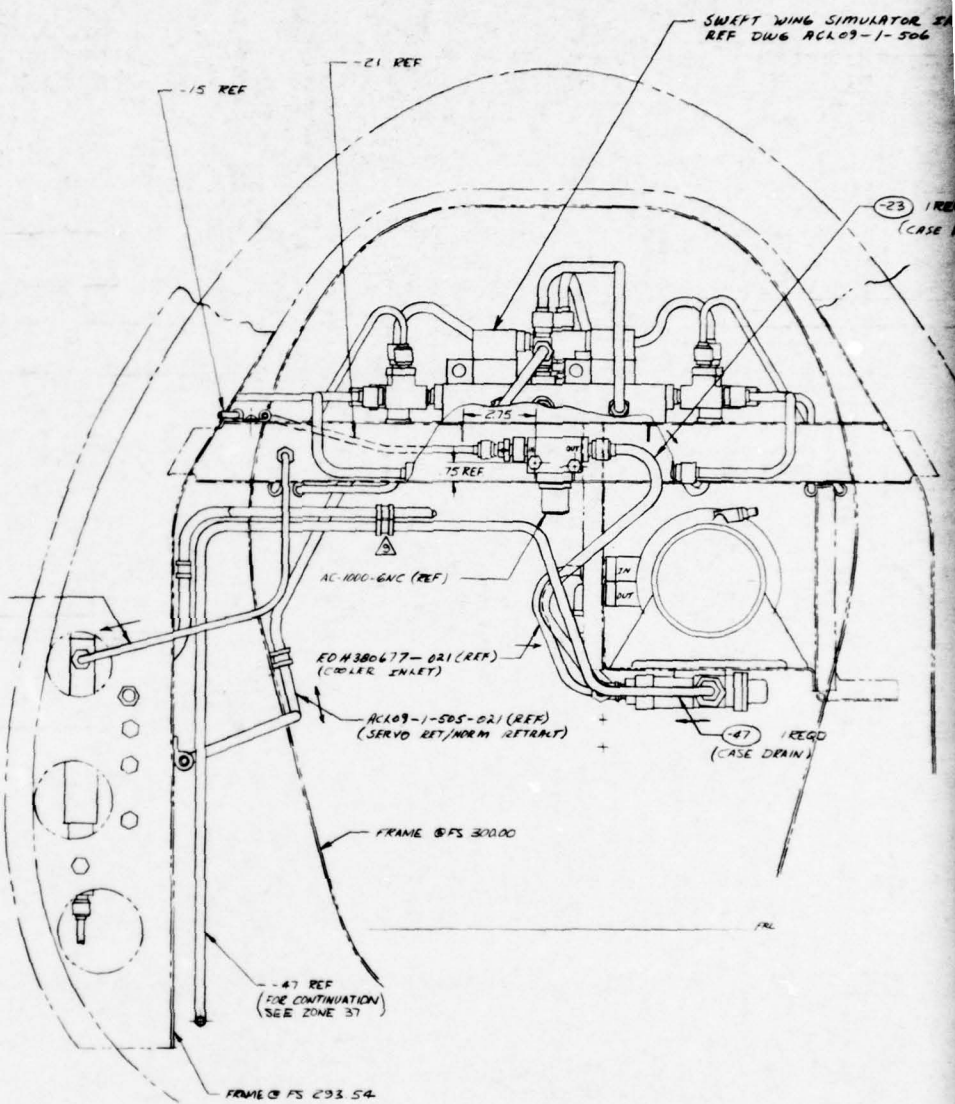
TABLE I
LIST OF 8000 PSI (55 MPa) COMPONENTS

<u>PART NO.</u>	<u>DESCRIPTION</u>	<u>MANUFACTURER</u>
66059	Motor/Pump Unit	Aerospace Division of Abex Corporation Aerospace Electrical Division of Westinghouse Electric Corporation
8691-524001-101	Rudder Actuator Assembly	Columbus Aircraft Division of Rockwell International
8691-524001-051	Bypass Valve	Columbus Aircraft Division of Rockwell International
1180A	Hydraulic Relief Valve	PneuDraulics, Inc.
R44598-6-0310	Hose	Resistoflex Corporation
21-6-9	Tubing	Trent Tube Division of Colt Industries
Dynatube [®] Series	Fittings	Resistoflex Corporation
MIL-H-83282	Fluid	Royal Lubricants Co.

NOTE: 8000 psi instrumentation is not included in this listing, see
Section 3.5.

[®] Dynatube, a Resistoflex development, is patented
in the United States and foreign countries

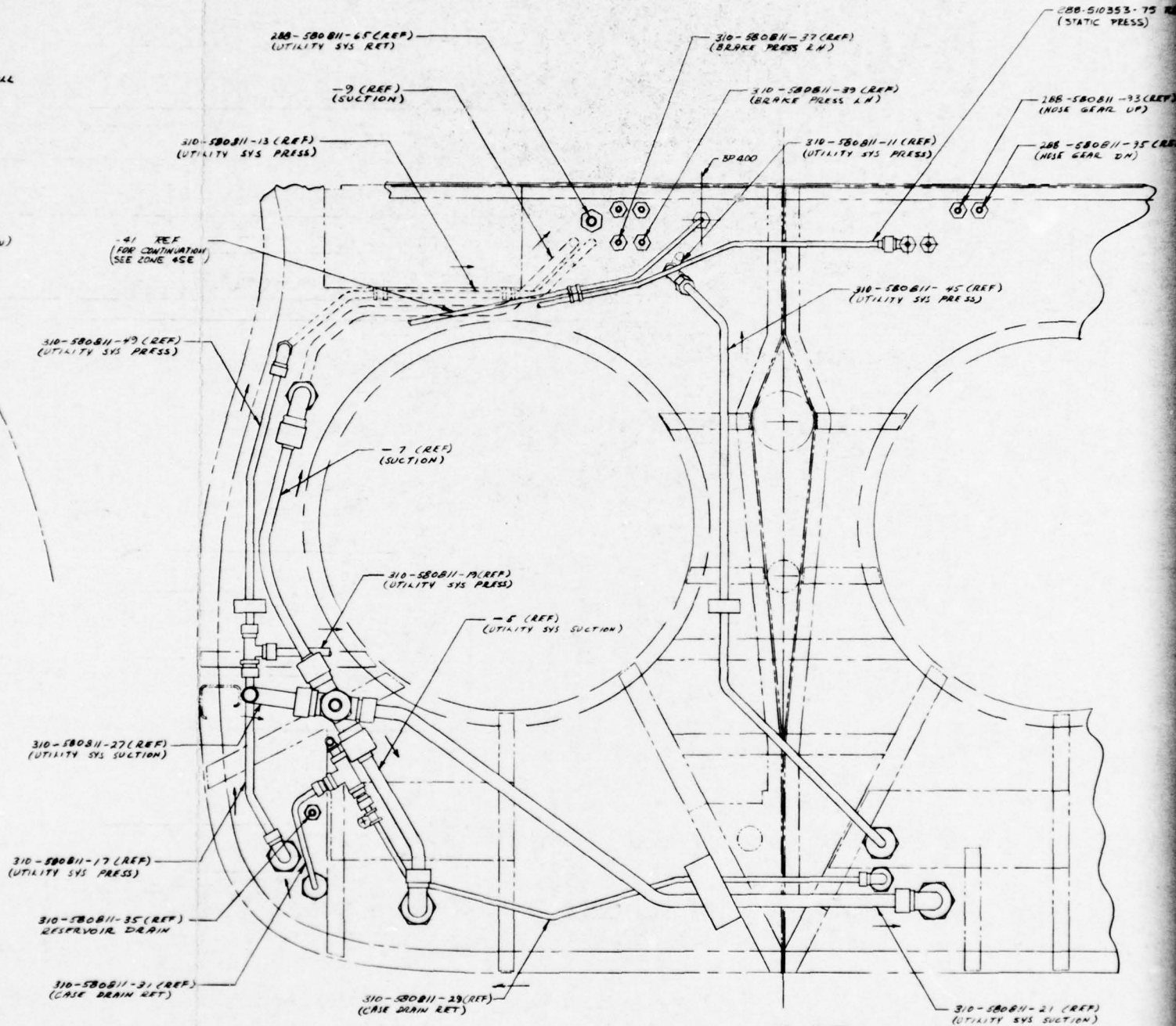
8881-280010



SECTION A-A 30
VIEW LOOKING AFT
FS 293.54

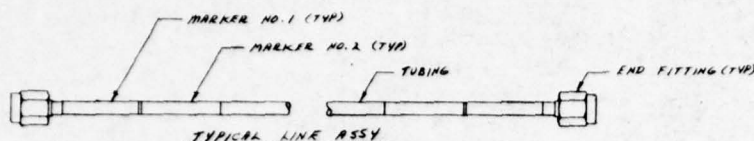
OR INSTALL
TUG

REGD
CASE DRAIN

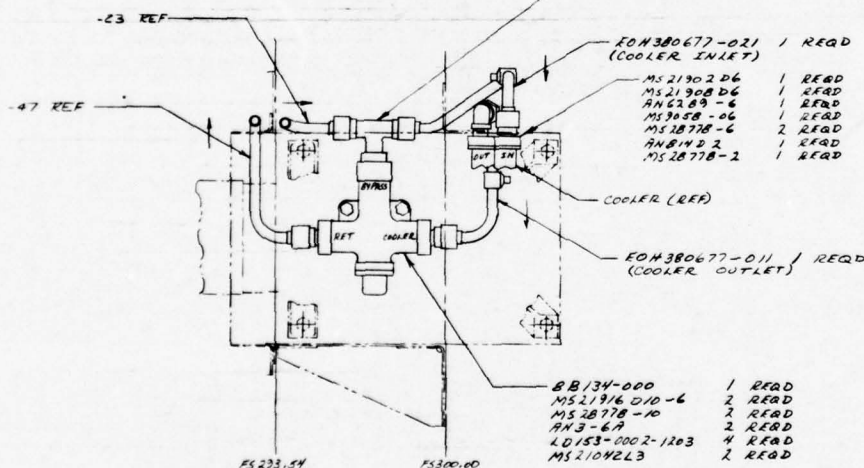


SECTION C-C
VIEW LOOKING AFT
FS 219.56

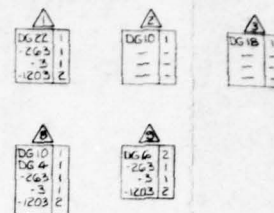
8691-580010		REV	SHEET
		/	/
19	18	17	16
			15



DATA NO.	END	TUBING	MATERIALS	MARKERS
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-3	-103	MS21921-8D	6061-T6 AL ALY	LD172-0006-0001
-5	-105	MS21921-10D	6061-T6 AL ALY	LD172-0006-0004
-7	-107	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-9	-109	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-11	-111	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-13	-113	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-15	-115	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-17	-117	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-19	-119	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-21	-121	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-23	-123	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-25	-125	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-27	-127	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-29	-129	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-31	-131	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-33	-133	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-35	-135	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-37	-137	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-39	-139	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-41	-141	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-43	-143	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-45	-145	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-47	-147	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-49	-149	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-51	-151	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004
-53	-153	MS21921-8D	6061-T6 AL ALY	LD172-0006-0004



SECTION 13-13



SEE SHT 3 FOR

ITEM NO.	QUANTITY	REMARKS	DATE
110	1	MS28778-10	
111	16	-6	
112	6	-4	
113	1	MS28778-2	
114	1	MS28778-2	
115	2	MS28778-06	
116	2	MS28778-04	
117	1	AC-1000-GNC	
118	1	PR76-0486	
119	1	10677 1180A	
120	2	211-35-100	
121	1	RDS F-295	
122	1	B/B134-000	
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290	1	50199 F44352 P.06	
291	1	50199 F44352 P.06	
292	1	50199 F44352 P.06	
293	1	50199 F44352 P.06	
294	1	50199 F44352 P.06	
295	1	50199 F44352 P.06	
296	1	50199 F44352 P.06	
297	1	50199 F44352 P.06	
298	1	50199 F44352 P.06	
299	1	50199 F44352 P.06	
300	1	50199 F44352 P.06	

9. THIS BLOCK DENOTES CALLOUTS FOR CLAMP INSTALL

△ CODE FOR DUPLICATION

CLAMP • DG • MS21912 DG ()
SCREW • MS 35207-263
NUT • MS21042L3
WASHER • LD153-0006-K03
SPACER • NAS43DD-3 ()
NUMBER READ

B. INSTALL FITTINGS AND TUBE ASSYS PER SPEC FRG-20.

7. INSTALL THREADED FASTENERS PER SPEC LAC101-001.

6. INSTALL IDENTIFICATION TAPE PER SPEC

LA0104-005.

5. IDENTIFY PER SPEC LAB104-003 EXCEPT
DO NOT METAL IMPRESSION STAMP

DO NOT WRITE IN THESE SPACES

4. FABRICATE TUBING PER SPEC FAG-153

(3) 3. TO BE DETERMINED AT INSTALLATION.

② 2. TO BE SUPPLIED BY ENGINEERING.

① / ALL PRODUCTION HARDWARE, REMOVED AS A RESULT OF THE INCORPORATION OF THIS TEST INSTALLATION, TO BE IDENTIFIED, PROTECTED AND RETAINED FOR RE-INSTALLATION.

NOTES: UNLESS OTHERWISE NOTED

HT 3 FOR PL CONTINUATION

28778-10	PACKING								
-6									
-4									
28778-2	PACKING								
28773-06	RETAINER								
28773-04	RETAINER								
1000-GNC	FILTER								36
7G-0486	NIPTOK / PUMP								52
10A	RELIEF VALVE								41
-35-000	PRESSURE NO. 0.6								38
5-F-295	HOSE								36
134-000	THERMAL BY PRESS								11
300	COPLER								36
									41
035B.P.06	ELBON								38
181P.0406	CONNECTOR								41
182P.06	CONNECTOR								55
182P.04	CONNECTOR								38
183T-04	TEE								38
181T-06	NUT								41
181T-04	TEE								24
180T-04	CONNECTOR								13
									13
1800P.04	FITTING								13
									13
1813D622	CLAMP								
D618									
D610									
D68									
D66									
1813D64	CLAMP								
1810-6	TEE								
181010-6	REDUCER								
18106-4	REDUCER								36
1814-4D	CAP								41
12-4	TEE								
10916	TEE								
9D4	TEE								
1 OR TYPING SEE	NOMENCLATURE	MATERIAL	THICK MATERIAL	WIDTH MATERIAL	LG SIZE	DRAWING OR SPECIFICATION NUMBER			FOR
	PARTS	LIST							

074			1	MS21908D6	ELBOW					11		
073			2	MS21908-B	FLBOW							
072			1	MS21908D4	ELBOW							
071			1	MS21908-4	ELBOW							
070			1	MS21907D4	ELBOW							
069			1	MS21902-B	UNION							
068			3	MS21902-6	UNION							
067			2	MS21902D6	UNION							
066			1	MS21902D4	UNION							
065			2	MS21042L4	NUT					36		
064			27	MS21042L3	NUT							
063			3	MS2058-06	BACKUP							
062										36		
061			1	AN803-2D	BUSHING					36		
060			5	AN6289-6	NUT							
059			2	AN6289-4	NUT							
058			1	AN937D6	CROSS							
057			2	AN924-8D	NUT							
056			3	AN924-4D	NUT							
055			4	AN901-8A	WASHER							
054			2	AN901-6A	WASHER							
053			5	AN901-4A	WASHER					11		
052			1	AN814-6	PLUG					41		
051			1	AN814DZ	PLUG					11		
050			2	AN4-24A	BOLT					36		
049			2	AN3-6A	BOLT							
048			4	AN3-5A	BOLT					12		
047			4	LD172-0006-0193	MARKER					12		
046			10	-0127						12		
045			2	-0105						12		
044			8	-0001						12		
043			12	-0004						12		
042			6	-0002						41		
041			42	LD172-0006-0001	MARKER					36		
040												
039			4	LD153-0002-1204	WASHER							
038			54	LD153-0002-1203	WASHER							
037												
036												
035												
034												
033												
032												
031												
030												
029												
028												
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007												
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005												
004												
003												
002												
001												
000												
ITEM OR FIND NO.	QUANTITY	REQUIRED	UNITS	CODE IDENT	PART OR IDENTIFYING NUMBER	NOMINATURE	MATERIAL	THICK. INCHES	WIDTH INCHES	LG. INCHES	DRAWING OR SPECIFICATION NUMBER	1 0 1

LAST CODES USED:

FIGURE 8

3			2		1		1/2	
					REVISIONS			
					DESCRIPTION	DATE	APPROVED	
					1. MAY BE REMOVED 2. CANNOT BE REMOVED			
					3. RECORD CHANGE 4. NEW SHOP PRACTICE			
					5. PARTS MADE ON			
037			1	HE23-0018-0003	TEE			
036			1	HE23-0007-0011	TEE			43
035								
034			1	CC12-2	CLIP			34
033			1	EDH380677-021	LINE ASSY.			11
032			1	EDH380677-011				11
031			1	504266-03-012				37
030			1	504266-03-011				40
029								
028								39
027			1	504266-02-010	LINE ASSY			38
026								
025			1	8691-580010-03	LINE ASSY			38
024			1		-47			20
023			1		-45			38
022			1		-43			48
021			1		-41			45
020			1		-39			41
019			1		-37			56
018			1		-35			40
017			1		-33			39
016			1		-31			21
015			1		-29			27
014			1		-27			27
013			1		-25			28
012			1		-23			20
011			1		-21			54
010			1		-19			46
009			1		-17			57
008			1		-15			34
007			1		-13			39
006			1		-11			42
005			1		-9			43
004			1		-7			44
003			1		-5			44
002			1		-3	LINE ASSY		46
001				8691-580010	HYD EQUIP INSTALL			

ITEM OR FIND NO.		QUANTITY REQUIRED	CODE IDENT	PART OR IDENTIFYING NUMBER	NOMENCLATURE	MATERIAL	THICKNESS GAGE	WIDTH GAGE	LENGTH GAGE	DRAWING OR SPECIFICATION NUMBER	7 0 N 1
PARTS LIST											
HEAT TREAT		UNLESS OTHERWISE SPECIFIED		CONTR NO.		Columbus Division North American Rockwell COLUMBUS, OHIO 43021					
FINISH		DIMENSIONS ARE IN INCHES		OWN BY		3-25-77					
		MFG TOLERANCES ON		CHK BY		J. E. G. 1-1-77					
		# 03		APVD		J. E. G. 1-1-77					
		XX (DECIMALS)									
		# 010									
		ANGLES									
		# 30									
		HOLES NOTED									
		DIN 13 THRU 040 + 001 - 001									
		041 THRU 130 + 002 - 002									
		131 THRU 229 + 003 - 003									
		230 THRU 350 + 004 - 004									
		351 THRU 750 + 005 - 005									
		751 THRU 1000 + 006 - 006									
		1001 THRU 2000 + 010 - 001									
		DO NOT SCALE PRINT									
G0 8691		TEST				SIZE CODE IDENT NO.		J 89372		8691-580010	
ST ASST		USED ON		FIND ITEM NO.		THRU		SCALE 1/2		SHEET 1 OF 3	
APPLICATION		EFFECTIVITY									

R441B2 D-06 /REGD
MS 28778-6 /REGD

MS 21902 D4 /REGD
MS 28778-4 /REGD

288-580812-67 REF

R44508-G-0310 /REGD
(FOR CONTINUATION SEE ZONE 39E)

288-580812-21 REF

-21 /REGD
(FOR CONTINUATION)

288-580812-63 REF

15/32 DIA HOLE
MS 21908-4 /REGD
AN 301-4A /REGD
AN 324-4D /REGD

3P 425

-17 /REGD
(SUCTION)

STA
251.780

STA
258.560

STA
263.938

STA
270.000

STA
276.940

STA
284.980

-37 /REGD
(OVERBOARD DRAIN)

VIEW 13-13

HE 273-0018-0003
AN 814-6
AN 6289-6
MS 28773-06
MS 28778-6

8691-580010

REV 3

58

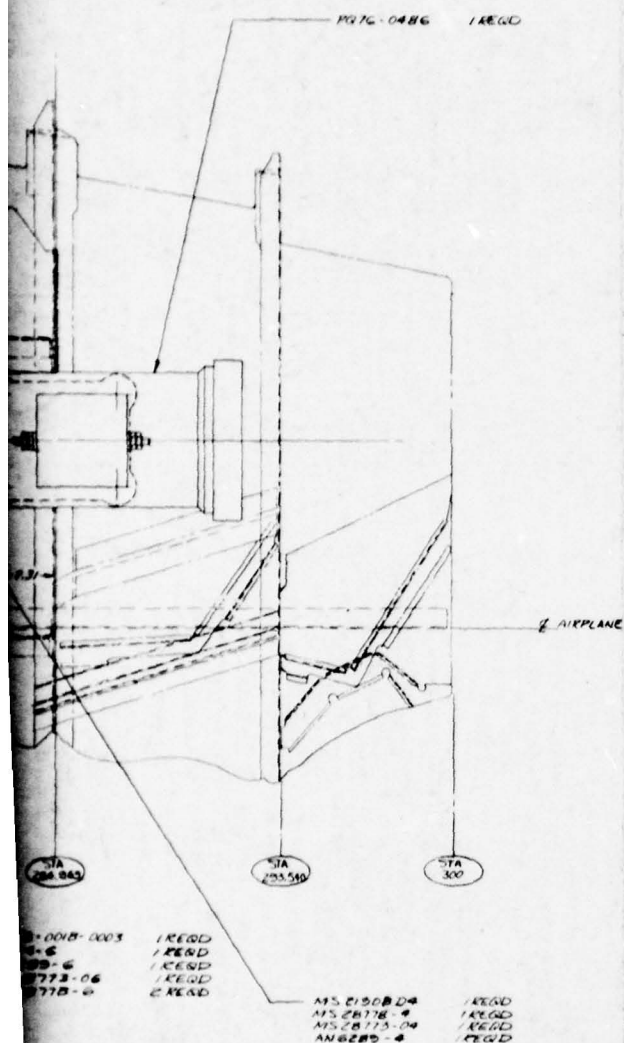
57

56

55

54

1 RECD (CASE DRAW)
CONTINUATION SEE ZONE 304

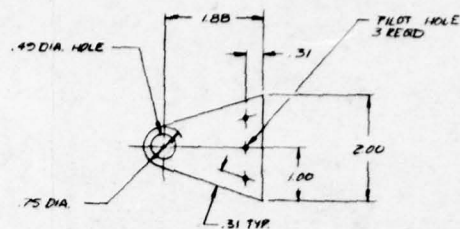


ITEM OR PART NO.	QUANTITY REQUIRED	CODE IDENT	PART OR IDENTIFYING NUMBER	NOMENCLATURE	MATERIAL	THICKNESS (INCHES)	WIDTH (INCHES)	LG (INCHES)	DRAWING OR SPECIFICATION NUMBER	ZONE
②	1	50599	R4459B-G-0310	MOSE						
②	1	50599	R4459C-T-04	TEE						
②	1	50599	R4418T-04	NUT						
	1		-75	BRACKET	CD24 AL CLAD SHEET	163	231	250	GG-A-250/5	
	1		-73	BRACKET	CD24 AL CLAD SHEET	163	210	248	GG-A-250/5	25
	1		-53	LINE ASSY						
	1		-51	LINE ASSY						
	2		NA543D04-40	SPACER						
	2		MS35B42-16	CLAMP						
	15		MS35207-263	SCREW						
	2		MS35207-261	SCREW						
	1		MS24423-6	CHECK VALVE						36
	1		MS24423-4	CHECK VALVE						36
	1		MS21924D6	UNION						
	4		MS21922-10	SLEEVE						
	8		-8							
	8		-6							
	16		MS21922-4	SLEEVE						
	4		MS21921-10D	NUT						
	4		-8D							
	4		-8							
	8		-6D							
	2		-4							
	12		MS21921-4D	NUT						

SCALE 1/2" = 1'-0"
SHEET 3
J 89372 8691-580010

303 DIA HOLE

31 TYP



DETAIL -73

188-580841-13 (REF)
(BOOST SYS PRESS)

-53 (REF)
(BOOST SYS RETURN)

188-580841-27 (REF)
(UTILITY SYS PRESS)

188-580841-37 (REF)
(UTILITY SYS RET)

FS 356.50

FS 367.00

FS 377.00

FS 387.64

53 1 REED
(BOOST SYSTEM RET.)
(REPLACES 288-580841-21)

25 1 REED
(RUDDER ACTR PRESS)

29 1 REED
(RUDDER ACTR RET)

REMARK: MS21924D4 1 REED ①
ADD: MS21912-4 1 REED
AN 901-418 1 REED
AN 924-4D 1 REED

188-580841-19 (REF)
(BOOST SYS RET)

188-580841-20 (REF)
(BOOST SYS RET)

51 1 REED
(BOOST SYS PRESS)
(REPLACES 288-580841-15) ①

188-580841-33 (REF)
(PRESTING HOOK UP)

27 1 REED
(RUDDER ALTR)

31 1 REED
(RUDDER ALTR PRESS)

188-580841-31 (REF)
(UTILITY SYS PRESS)

188-580841-29 (REF)
(UTILITY SYS PRESS)

188-580841-35 (REF)
(UTILITY SYS RET)

VIEW LOOKING OUTBOARD RH SIDE

32

31

30

29

28

27

A vertical diagram showing a sequence of points labeled H, G, F, E, D, C, and B from top to bottom. A horizontal arrow points to point E.

580841-17 (REF)
OF SYS PRESS)

FS 107.50

CTD ES 409,001

— 29 —

27 REF

BYLE 20500

5-19

WP 15.58

- 3/ REF

249-31690-12 RLF

73 PERD 28

-73 (REQD) 28

R44362T-04
R44118T-04
AN301-4A

1 REED
1 REED
2 REED

FOR 4th, C/W, APPLICATION &
REVISIONS, SEE SHEET 1.

SIDE

Rediff International Corporation Columbus Aircraft Division Columbus, Ohio 43216		SIZE J	PSCW NO 89372	8691-580010
DWN BY GRISF/MEIL				
DATE 3-15-77		SCALE 1/1		SHEET 2

FORM 311-G-1-6 88X A-78

A thermal analysis performed in Reference 4 indicated a heat exchanger would be required because of (1) the small surface area and volume of the 8000 psi system, and (2) the oversize 8000 psi pump. The 8000 psi pump had the potential for delivering 18.7 hp (13.9 kW). The output was reduced to 2.6 hp (1.9 kW) to match rudder actuator flow rates and T-2C electrical power generation capabilities.

Hydraulic fluid in the T-2C was changed from MIL-H-5606 to MIL-H-83282 for the LHS flight test program, Reference 13. This fluid was retained for the AFCAS program.

3.3.2 Motor/Pump Unit

Pump - The pump was designed and fabricated by the Aerospace Division of Abex Corporation in Oxnard, California, and was identified as M/N APIV-106, P/N 63077. Design details are given in Reference 12. The pump is a constant pressure, variable displacement, axial piston unit, Figure 9. Rated delivery is 3.2 gpm (12.1 L/m) at 7330 rpm and 7850 psi (54 MPa) with a +180°F (82°C) inlet fluid temperature. Delivery was reduced for the AFCAS program to 0.54 gpm (2.04 L/m) at 8100 rpm and 7850 psi with a +180°F inlet fluid temperature.

Motor - The motor was designed and fabricated by the Aerospace Electrical Division of Westinghouse Electric Corporation, in Lima, Ohio. The unit was originally built to drive a New York Air Brake pump in a commercial airliner--the Lockheed "Jet Star". The motor has a radio noise filter, thermal protector, shock mounts, and is explosion-proof, Figure 9. The unit is rated for 8 hours continuous duty at 35,000 ft. (11 km); motor brushes have a design life of 500 hours. Rated output is 4.75 hp.

The motor, identified as P/N 914F591-2, had the following operating characteristics when coupled to Abex pump M/N APIV-106:

Input voltage:	28 volts DC (nominal)
Running current:	140 amperes (after warm-up)
Starting current:	1000+ amperes for 0.035 sec. (1200 amperes peak)
Speed:	8100 rpm
Pump discharge pressure:	8000 psi (55 MPa)
Pump discharge flow:	250 cc/min

A view of the motor/pump installation is presented as Figure 10.

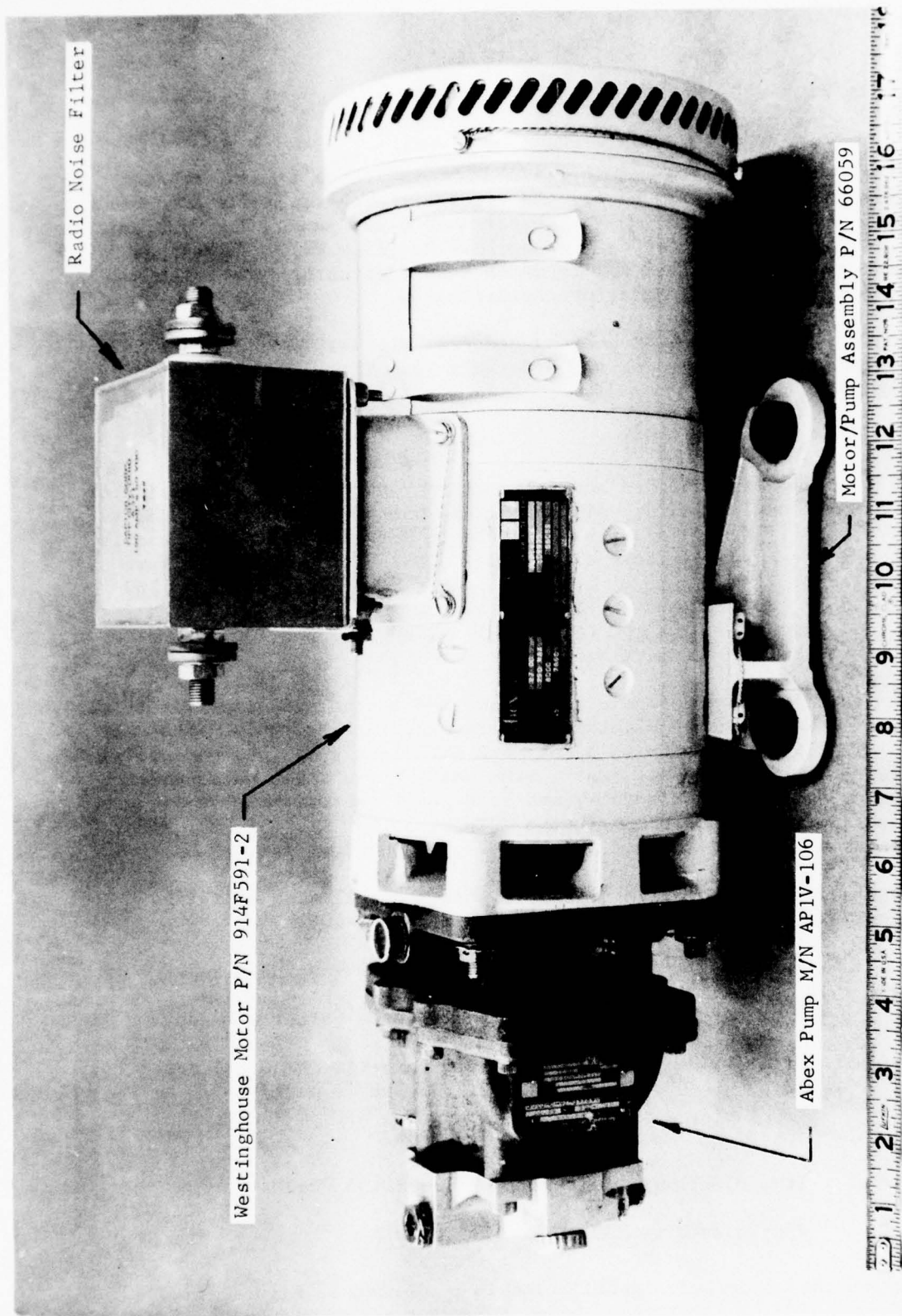


FIGURE 9 MOTOR/PUMP UNIT

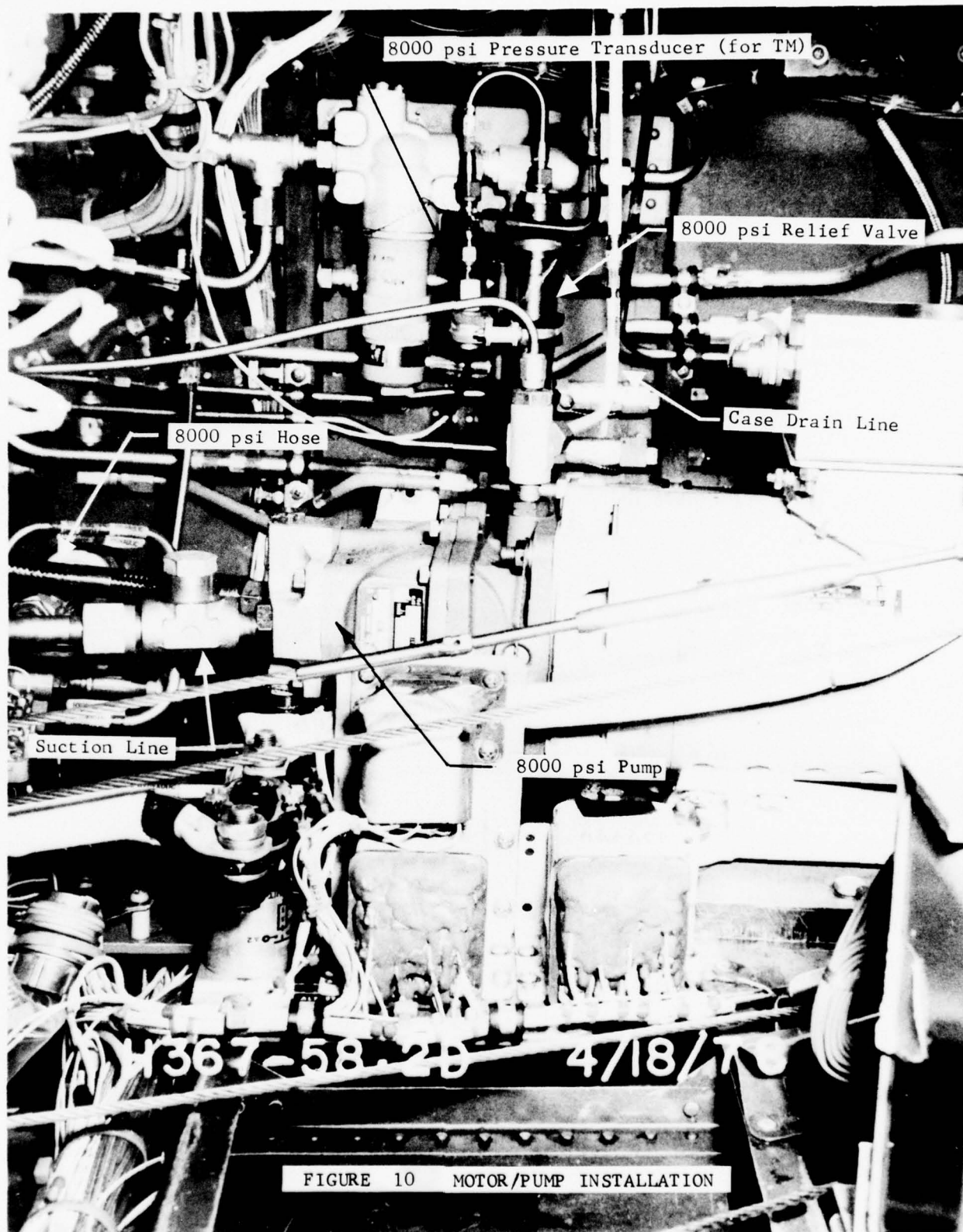


FIGURE 10 MOTOR/PUMP INSTALLATION

3.3.3 Rudder Actuator

The control-by-wire rudder actuator was designed by the Columbus Aircraft Division of Rockwell International under Contract N62269-75-C-0311, Reference 4. The assembly is an engineering model and should not be considered a final design. MIL-C-5503 requirements were followed except as modified for 8000 psi (55 MPa) operating pressure and utilization of some modularization techniques to achieve commonality. The actuator is shown on Figure 11. Design constants are listed below; design details are discussed in Reference 4.

Operating pressure	8000 psi (55 MPa)
Piston stroke (total)	3.5 in. (8.9 cm)
Cylinder bore	0.926 in. (2.3 cm)
Rod diameter	0.748 in. (1.9 cm)
Piston effective area	0.234 in ² (1.5 cm ²)
Force output (max.)	1870 lb (8.3 kN)
Piston velocity (max.)	5.5 in/sec (14 cm/s)
Actuator length (extended)	18.375 in. (46.7 cm)

Actuator piston position and rate are commanded by a spool/sleeve type flow control valve. The valve spool is driven directly by a lever attached to the armature of a permanent magnet force (torque) motor mounted on the valve housing. The motor has four independent coils for redundancy. Actuator piston position feedback is provided by two DC-operated LVDT transducers, one mounted on each side of the unit. A bypass valve was added to automatically interconnect the two cylinder chambers in the event hydraulic power were lost. Manufacturers of major components in the actuator were:

<u>Part No.</u>	<u>Description</u>	<u>Manufacturer</u>
8691-524001-101	Rudder Actuator Assembly	Columbus Aircraft Division of Rockwell International
8691-524001-051	Bypass Valve	Columbus Aircraft Division of Rockwell International
SO 4262-03-21	Control Valve	Ronson Hydraulic Units Corporation Charlotte, North Carolina
99-D0234 (M/N 21-6-200)	Force Motor	Servotronics, Inc. Buffalo, New York
2000 HCD	Position Transducer	Schaevitz Engineering Camden, New Jersey

The rudder actuator installation is shown on Figure 12.

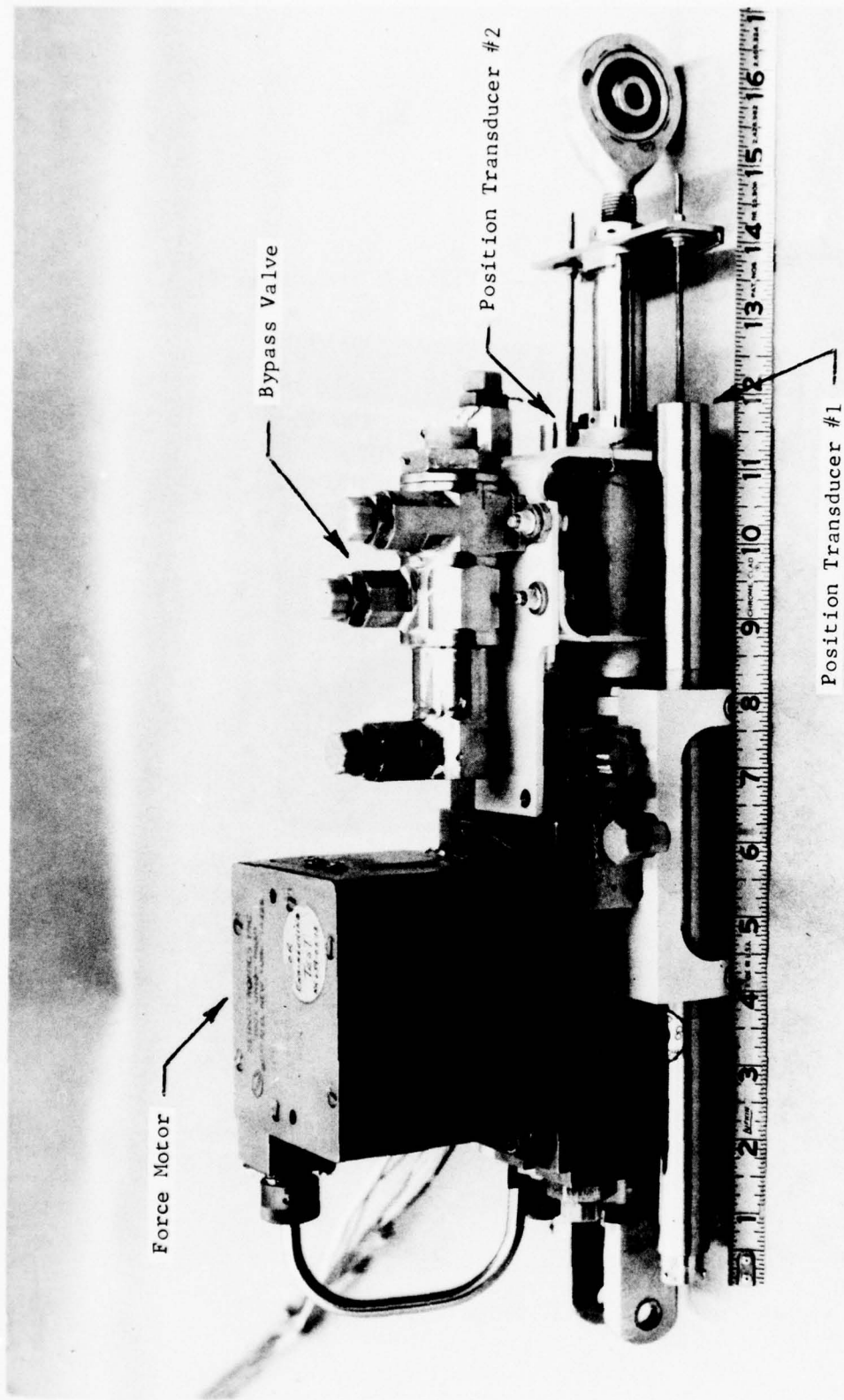


FIGURE 11 RUDDER ACTUATOR ASSEMBLY

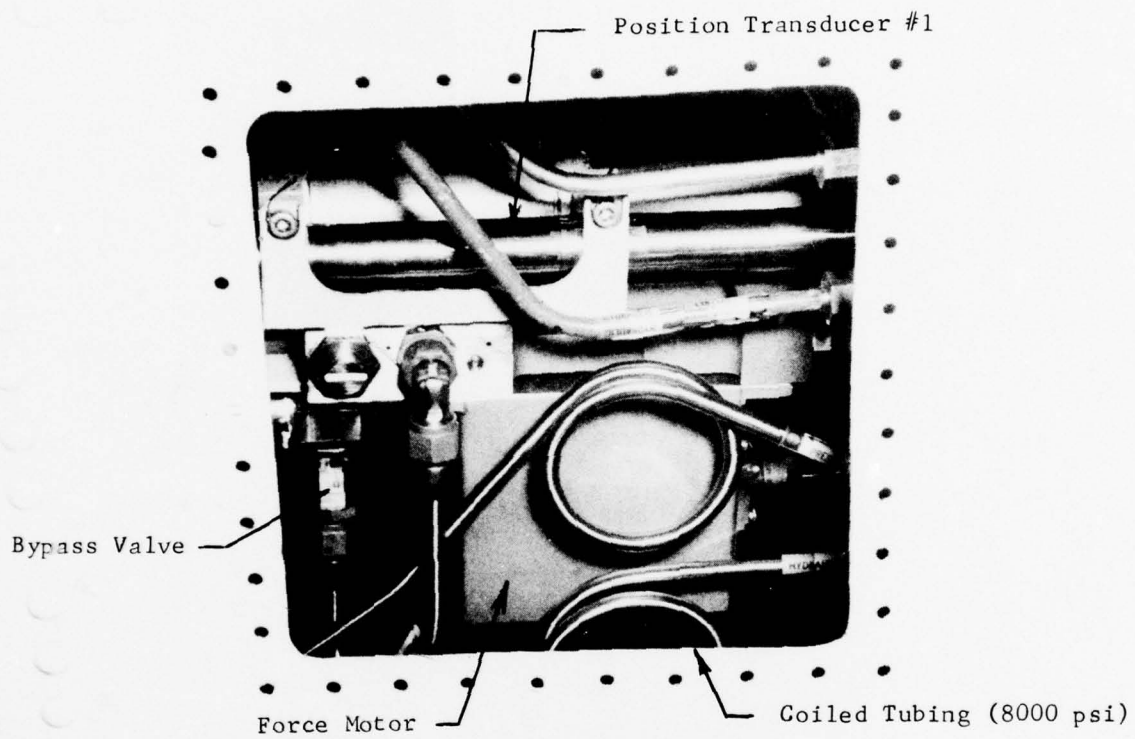


FIGURE 12 RUDDER ACTUATOR INSTALLATION

3.4 ELECTRICAL SYSTEM

3.4.1 Electronic Drive Unit (EDU)

The electronic drive unit, P/N 8691-546604, was designed and fabricated by the Columbus Aircraft Division of Rockwell International Corporation. Circuit concepts employed in the unit were developed under company funded IR&D projects. Innovative application of redundancy and feedback techniques permit EDU operation to be maintained with multiple component failures. Although the assembly was designed and fabricated to be suitable for flight, the EDU was nevertheless an experimental model. The assembly contained discrete components, test points, and external adjustments to facilitate data acquisition. This resulted in a much larger package than would be needed for a production unit. A production design EDU would have approximately 5% of the volume of the AFCAS unit.

The EDU was powered by 115 volt 400 \sim AC and basically had two channels with dual sub-circuits (4 channels total). Bias pots were provided to adjust the input, feedback, and balance of each channel. Two power supplies provided ± 15 VDC for the signal amplifiers and system transducers. All circuitry was contained on two identical printed circuit boards, Figure 15.

3.4.2 Force Transducer

The force transducer housing assembly, P/N 8691-524001-61, was designed and fabricated by CAD, Figure 14. The housing contains two DC-operated LVDT force transducers, P/N FTD-1T-500, built by Schaevitz Engineering in Camden, New Jersey. The units have a maximum capacity of 500 lb (2.2 kN), a spring rate of approximately 8000 lb/in (1.40 MN/m), and an output of 0.01 v/lb (.002 v/N) in tension or compression.

3.4.3 AFCAS Circuitry

A simplified block diagram of all elements in the system is shown on Figure 16. Pilot inputs are transmitted through the rudder pedals via cables, pulleys, and bellcranks to the force transducer located inside the vertical stabilizer. Gearing multiplies pilot effort by 2.28. Transducer output is the command signal (e_i) to the EDU. Amplifiers in the EDU process e_i with a feedback signal (e_{fb}) and power the force motor coils which drive the spool X_i in the control valve. The valve ports 8000 psi hydraulic fluid to the rudder actuator in response to X_i . Actuator piston travel is sensed by position transducers having an output of 5 v/in; this is the feedback signal (e_{fb}). Actuator piston travel (± 1.75 in. max.) is converted through a bellcrank and push rod to angular travel of the rudder ($\pm 12^\circ$ max.).

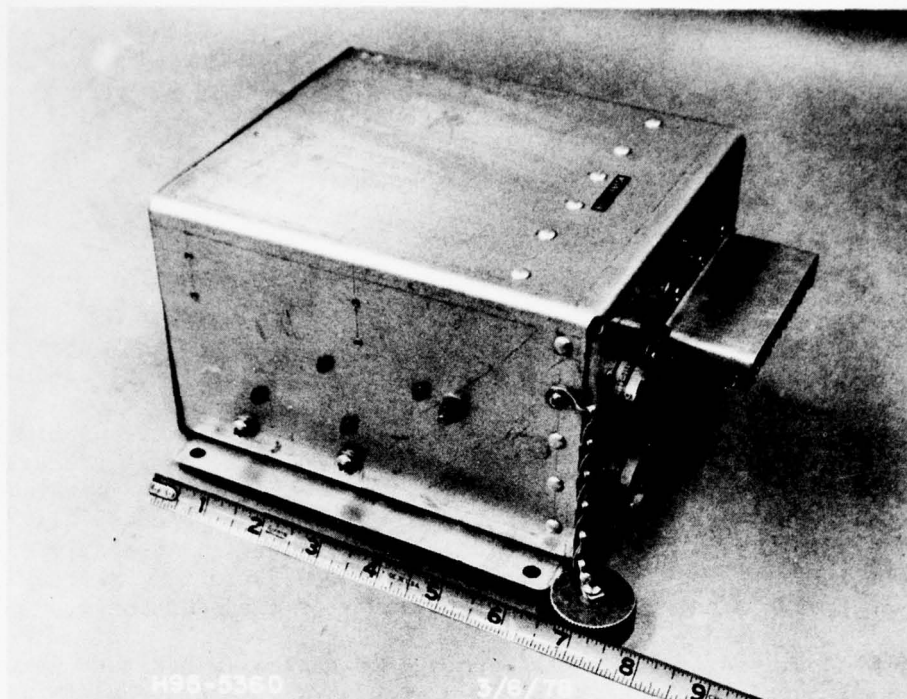


FIGURE 13 ELECTRONIC DRIVE UNIT

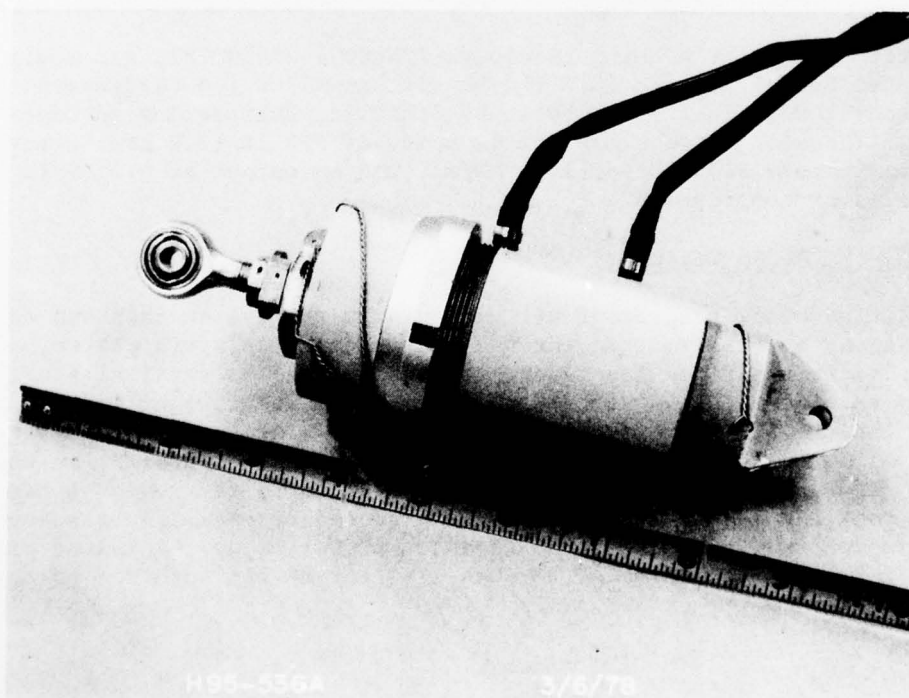


FIGURE 14 FORCE TRANSDUCER

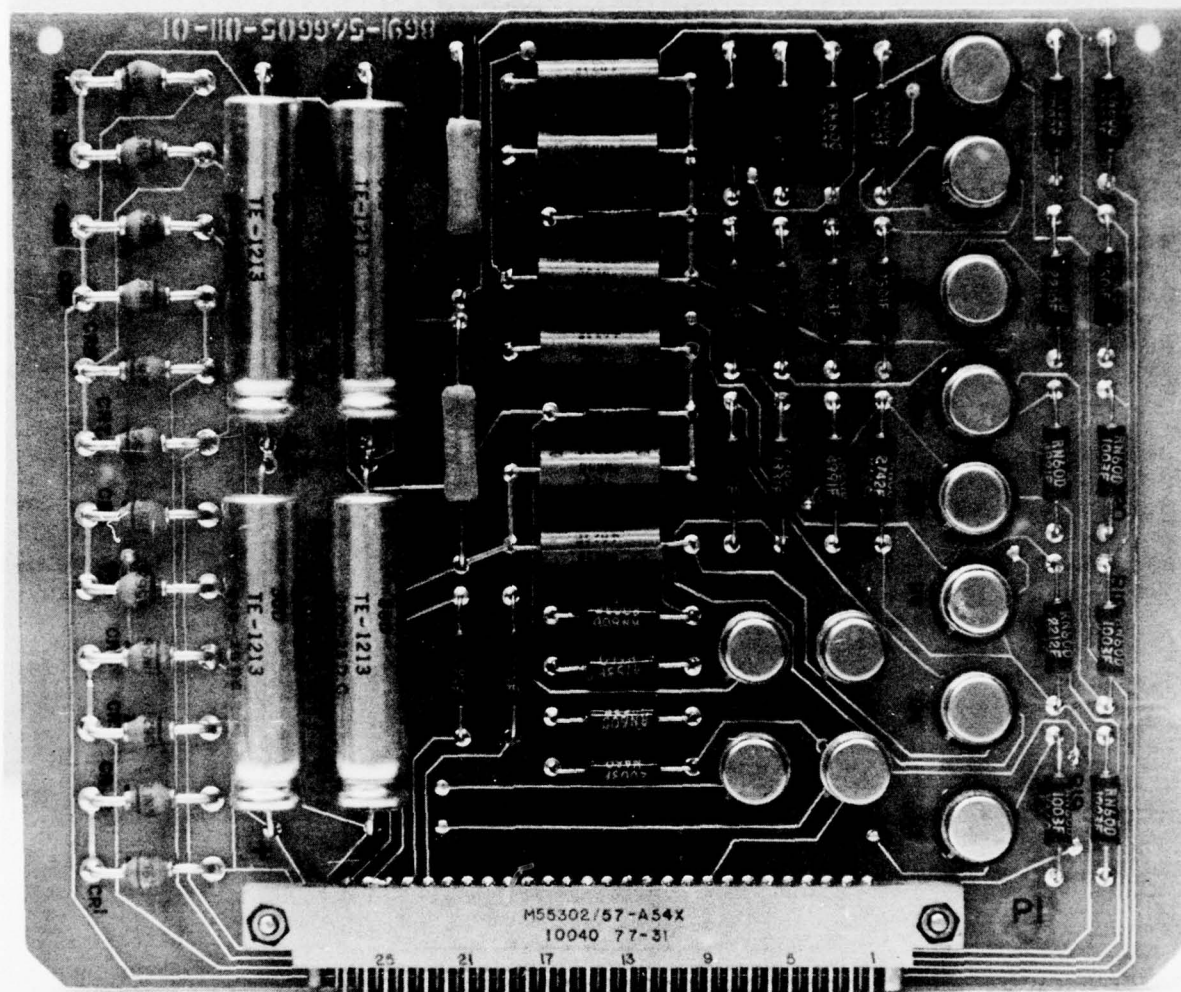


FIGURE 15 EDU PRINTED CIRCUIT BOARD

METRIC CONVERSIONS:

1b x 4.45 = N
 1in. x 2.54 = cm
 psi x 6895 = Pa

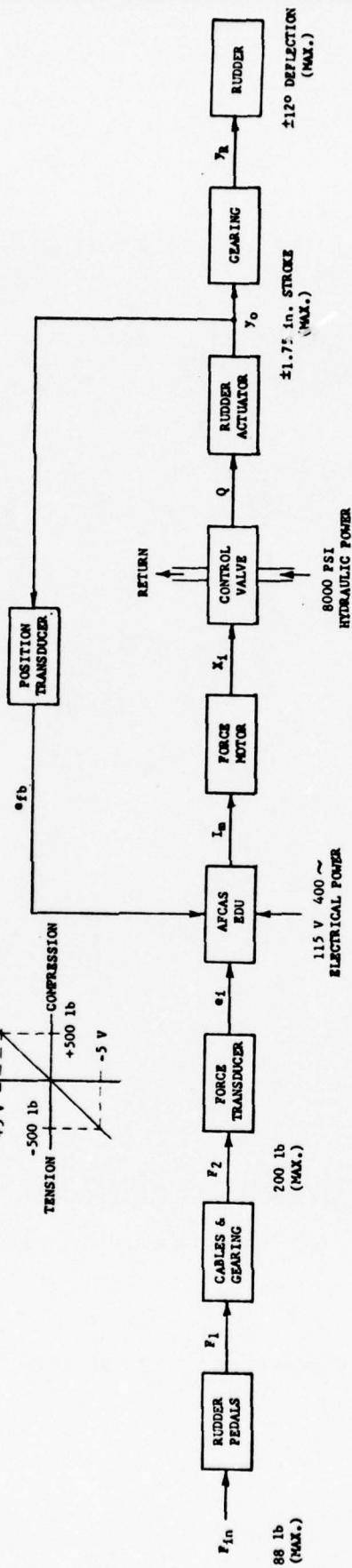
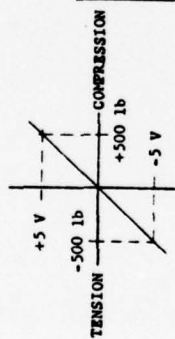
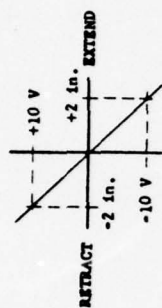


FIGURE 16 BLOCK DIAGRAM OF SYSTEM

A simplified diagram of electrical components in the test installation is presented on Figure 17. System redundancy is illustrated on Figure 18. The system concept developed under CAD IR&D studies is flexible in that various levels of redundancy could be employed (as required) for other applications. AFCAS redundancy features include:

- Dual force (input) transducers
- Dual position (feedback) transducers
- Dual power supplies
- Quad electronics
- Feedback fault correction

A schematic diagram of EDU electronics is presented on Figure 20. The diagram also shows test points and adjustment features added for the AFCAS program. Each of the four power amplifiers employs current feedback with a highly reliable darlington power transistor configuration and independent power supplies. The circuitry is designed so that in the event an output stage fails "hard-over", voltage applied to a motor coil will not exceed its rated value. This limiting feature permits a subunit failure to be compensated or nullified by another subunit. Closed loop tests reported in Reference 3 verified that operation of the redundant subunits provided high immunity to component failures.

A math model of the idealized system is presented on Figure 19. The transfer functions are for "small signal" inputs and do not reflect fluid flow saturation limitations or motor current limitations imposed by coil inductance. System spring-mass effects (actuator loaded) were not included. Optimum loop gain was 90; this provided a theoretical band width of 14.3 Hz and a damping ratio of 0.7.

Performance characteristics of the test installation were higher than could be utilized in the T-2C rudder system. To assure satisfactory operation, AFCAS dynamics were matched with T-2C directional system dynamics. This was accomplished by lowering loop gain to 20 and adding high frequency roll-off filtering to reduce the possibility of system noise.

3.4.4 AFCAS Installation

The wiring diagram detailing AFCAS interconnecting cables and terminals is presented as Figure 21. Standard plugs and shielded wiring were used throughout the system. AC and DC power control relays, circuit breakers, and cockpit wiring are detailed on Figure 22. The electronic drive unit was installed on the forward bulkhead of the T-2C fuselage compartment, Figure 23.

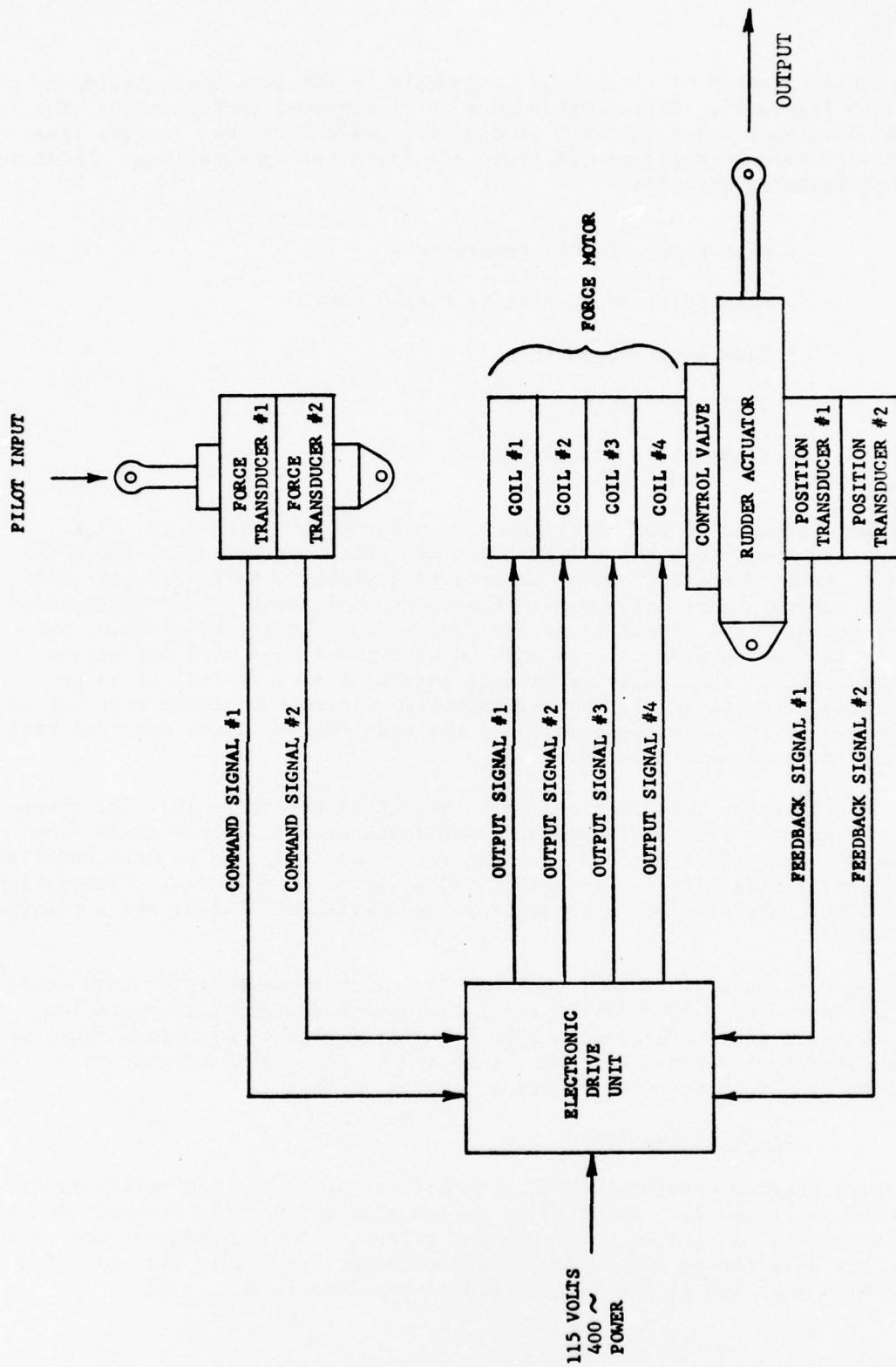


FIGURE 17 SIMPLIFIED DIAGRAM OF ELECTRICAL COMPONENTS

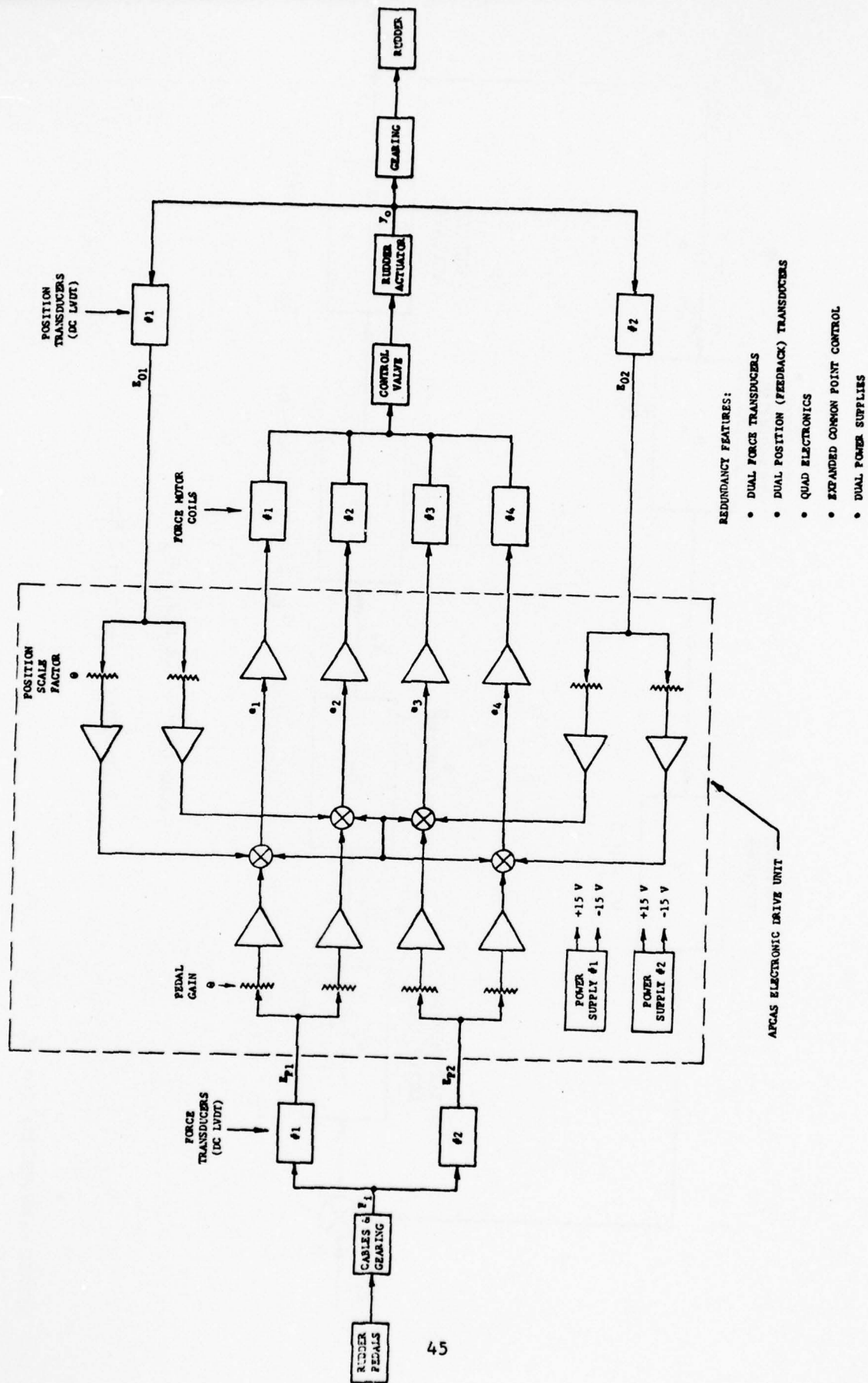
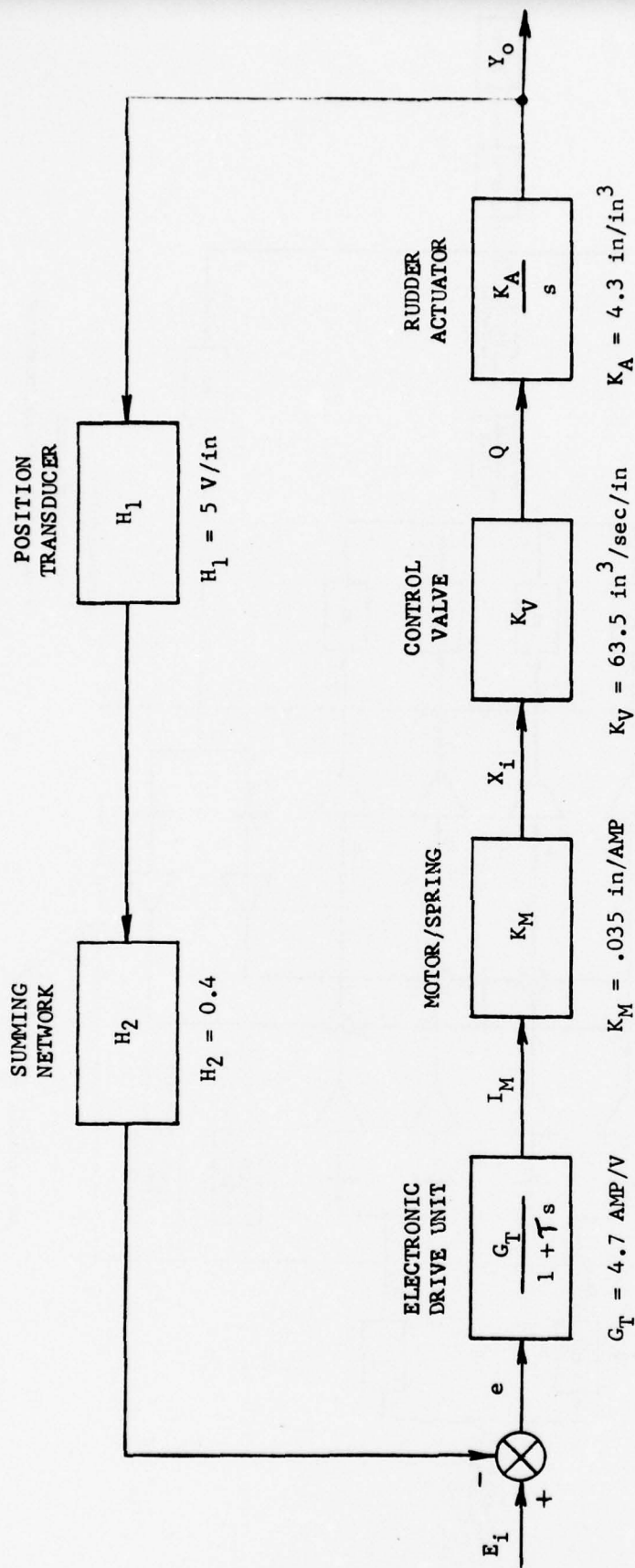
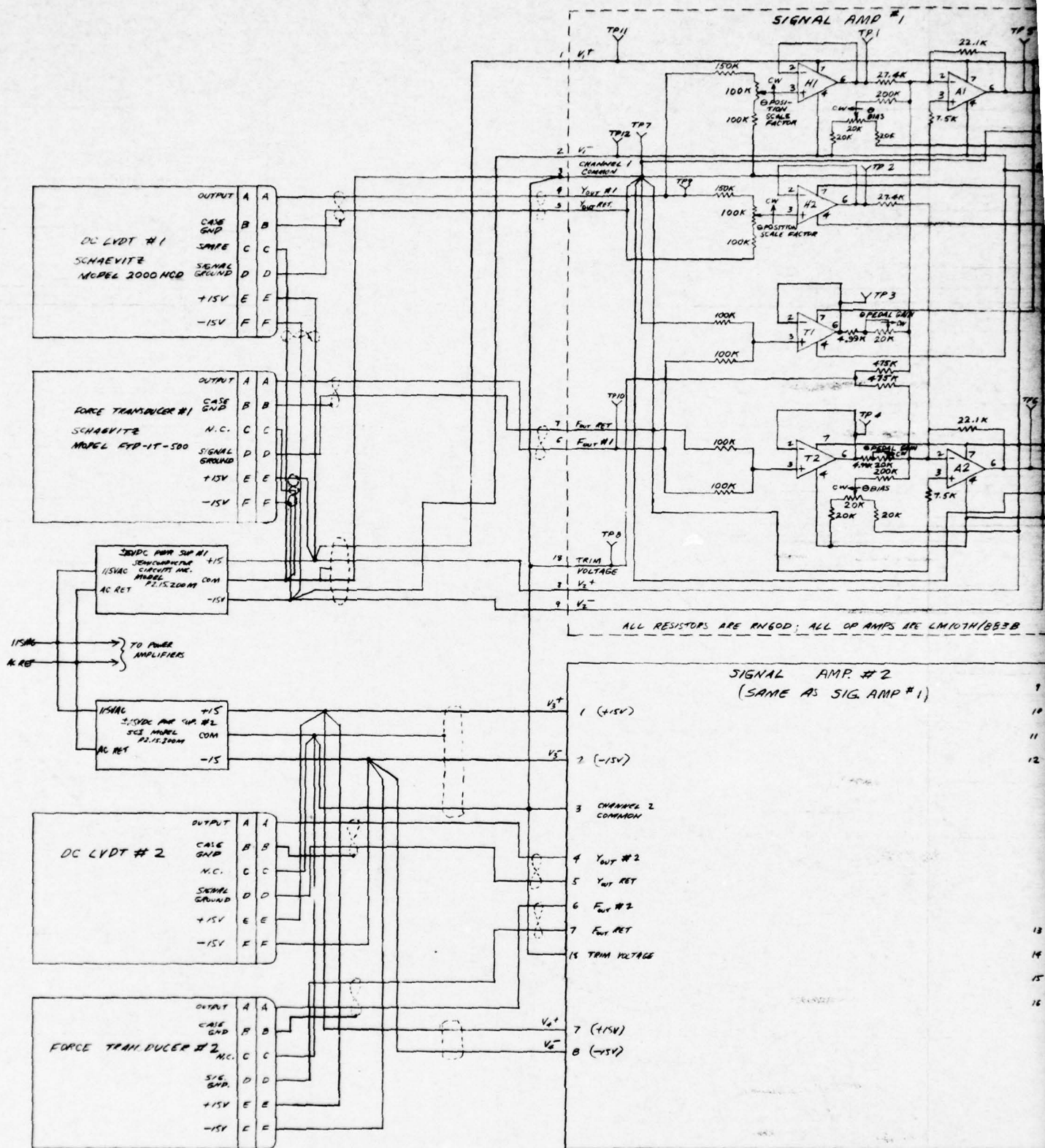


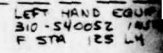
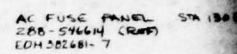
FIGURE 18 SIMPLIFIED DIAGRAM SHOWING SYSTEM REDUNDANCY



METRIC CONVERSION: in X 2.54 = cm

FIGURE 19 MATH MODEL



[illegible]

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12

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C203A22T (WHT)
C204A22T (WHT)
C205A22T (WHT)
C206A22T (WHT)

C207A22T (WHT)
C208A22T (WHT)
C209A22T (WHT)
C210A22T (WHT)

PTO6R-10-65 (SR)

C211A22T (WHT)
C212A22T (WHT)
C213A22T (WHT)
C214A22T (WHT)
C215A22T (WHT)
C216A22T (WHT)

NO. 2 DC LVDT
MODEL 2000HCD
LOC RM SIDE OF R691-524001-01 ACTUATOR
FUS STA 404 CL

SLEEVE 1 REDD
HD 591-0002-0402
"NO. 2 SYS"

PTO6R-10-65 (SR)

C200A22T (WHT)
C201A22T (WHT)
C202A22T (WHT)
C203A22T (WHT)
C204A22T (WHT)
C205A22T (WHT)

NO. 1 DC LVDT
MODEL 2000HCD
LOC LH SIDE OF R691-524001-01 ACTUATOR
FUS STA 404 CL

SLEEVE 1 REDD
HD 591-0002-0402
"NO. 1 SYS"

PTO6R-10-65 (SR)

C215A22T (WHT)
C216A22T (WHT)
C217A22T (WHT)
C218A22T (WHT)
C219A22T (WHT)
C220A22T (WHT)

DISC NO. 2
SECURE WITH WIRE BUNDLE CLAMP
AT APPROX FSTA 410 TO
PROVIDE ACCESS TO CONNECTOR
FROM RUDDER ACTUATOR
ACCESS DOOR

SLEEVE 1 REDD
HD 591-0002-0402
"NO. 2 SYS"

NO. 2 FORCE TRANSDUCER
FTD-1T-500
PART OF R691-524001-01 LINK ASSY
FUS STA 418.48 BY SR LH

PTO6R-10-65 (SR)

C221A22T (WHT)
C222A22T (WHT)
C223A22T (WHT)
C224A22T (WHT)
C225A22T (WHT)
C226A22T (WHT)

DISC NO. 1
SECURE WITH WIRE BUNDLE
CLAMP AT APPROX FSTA 410
TO PROVIDE ACCESS TO CONNECTOR
FROM RUDDER ACTUATOR
ACCESS DOOR

SLEEVE 1 REDD
HD 591-0002-0402
"NO. 1 SYS"

NO. 1 FORCE TRANSDUCER
FTD-1T-500
PART OF R691-524001-01 LINK ASSY
FUS STA 418.48 BY SR LH

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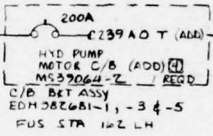
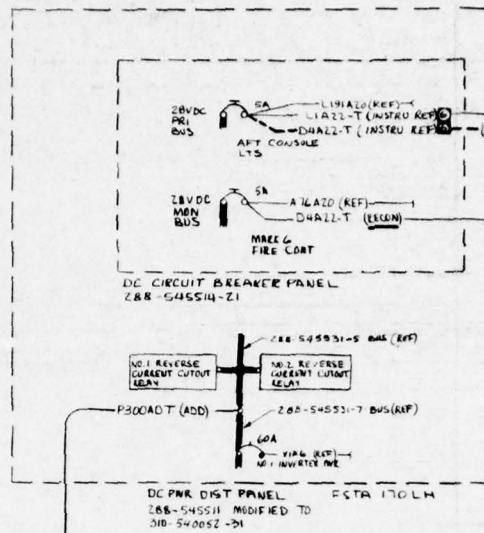
6691-S

FIGURE 21

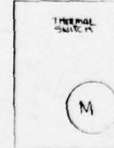
				REVISIONS			
DATE		DESCRIPTION		DATE		APPROVED	
		1. MAY BE REWORKED 2. CANNOT BE REWORKED					
		3. RECORD CHANGE 4. NON-WORK PRACTICE					
		5. PARTS MADE ON					
	4	M7928/1-51	TERMINAL			MIL-T-7928	
	1	M7928/1-24	TERMINAL				
	2	M7928/1-25	TERMINAL				
	5	M7928/1-15	TERMINAL				
045	3	M7928/1-18	TERMINAL			MIL-T-7928	
046	2	LD43-0011-0017	AL WASHER				
047							
048							
049	5	07922 YAN/001/0 STEEL	TERMINAL				
050							
051	8	ST4H0206H0001	SPLICE			TERMINAL USE SOURCE CONT ENG	
052							
053	1	7929 6E74-T	SWITCH			NAS-75655	
054							
055	2	81349 F028-1251-2A	FUSE			MIL-F-1516-00	
056	2	81349 FHN78WB	FUSEHOLDER			MIL-F-19207/17	
057							
058	1	09026 BRZ-615AB-573	RELAY			NAS-75642	
059	3	HE46-0005-0003	TERM PLUGS			RENDER ITEM USE SOURCE CONT ENG	
060	1	HE414-0003-0001	CONN RELAY			RENDER ITEM USE SOURCE CONT ENG	
061							
062	2	HD591-0002-0402	10 SLEEVE - NO 2 SYS	MAKE FROM			
063	2	HD591-0002-0402	10 SLEEVE - NO 1 SYS	MAKE FROM			
064	1	HD591-0002-0702	10 SLEEVE - PDS LEADS	MAKE FROM			
065							
066							
067	4	PT06A10-65(58)	CONN PLUG				
068	1	76706 MS29064-2	CIRCUIT BREAKER - 200A				
069	1	76706 MS3476-18-325	CONN PLUG				
070							
071	AK	76706 M2759/16-18-9	WIRE			MIL-W-22759	
072	3	76706 MS25274-2	TRIP				
073	1	76706 MS24171D1	RELAY				
074	1	76706 MS27473710F359	CONN PLUG				
075	1	76706 MS27473710F359	CONN RECP				
076	AK	76706 M2759/16-20-9	WIRE			MIL-W-22759	
077	AK	76706 M2759/16-22-9	WIRE			MIL-W-22759	
078	AK	76706 M7078/16-22-2	WIRE	2 COND TW		MIL-C-7078	
079	AK	76706 M7078/16-22-3	WIRE	3 COND TW		MIL-C-7078	
080	AK	76706 M7078/26-22-2	WIRE	2 COND TW SHLD		MIL-C-7078	
081	AK	76706 MS90294-01-9	WIRE			MIL-W-22759/4	
082	1	76706 MS35555-46	LOCK WASHER				
083	1	76706 MS21042-26	NUT				
084	1	76706 AN961-6/67	WASHER				
085	1	76706 AN6-13A	BOLT				
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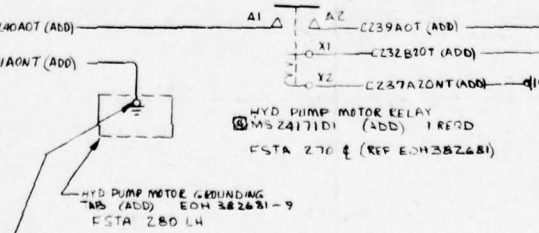
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E
D
C
B
A



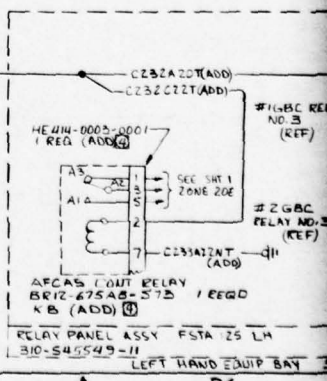
THERMAL SW TERMINAL LOCATED UNDER NEG (-) PUMP MOTOR TERMINAL



TERMINAL DESIGNATORS + & - ARE LOCATED ON END OF TERMINAL STUDS



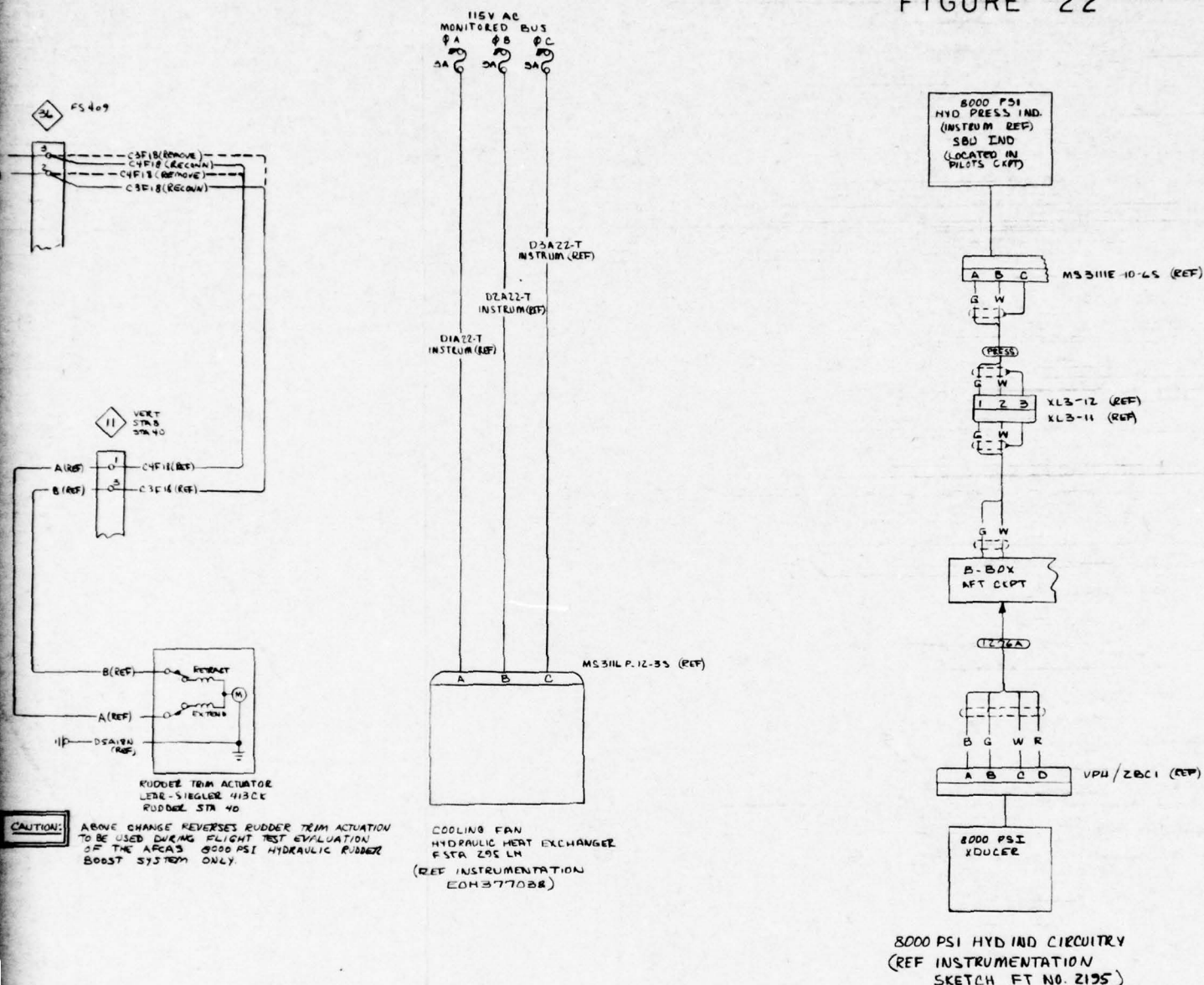
4N6-13A
LOV53-0011-001 2 REQD
MS23064-8 1 REQD
MS23064-9 1 REQD
AN761-616T 1 REQD



40 39 38 37 36

REVISIONS	
NO.	DESCRIPTION
1	MAY BE REMOVED 2 CANNOT BE REMOVED
2	RECORD CHANGE 3 NEW SHOP PRACTICE
3	5 PARTS MADE OK

FIGURE 22



WIRING DIAGRAM
AFCAS POWER CONTROL

Revlon International Corporation Columbus Aircraft Division Columbus, Ohio 43260	DATE 1-1-78	SCALE	SIZE 89372	8691-5
--	-------------	-------	------------	--------

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49/50

3, 100

REVISIONS		DATE	APPROVED
DESCRIPTION			
1. CANNOT BE REMOVED			
2. FROM SHOP PRACTICE			
3. MADE ON			

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G

F

E

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C

8691-546606

GRAM -
R CONTROL

A
SAFE

8691-546606

SHEET 2

25 12

FORM 311 G-21 REV 6-76

3, 8691-546606 SHI 2

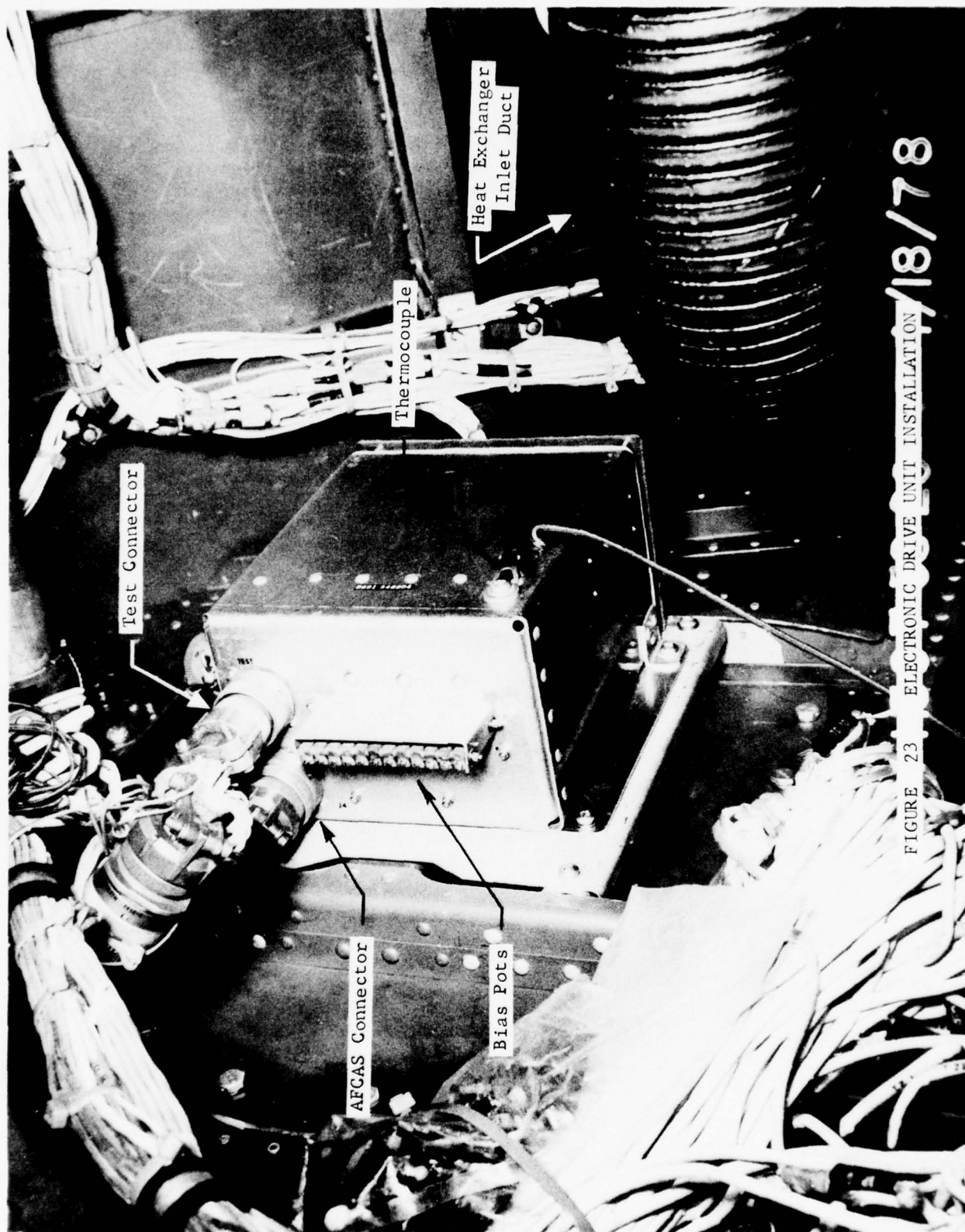


FIGURE 23 ELECTRONIC DRIVE UNIT INSTALLATION 18/78

The T-2C was equipped with several flight data acquisition systems. Two were used in the AFCAS program: (1) an 18 channel telemetry system, and (2) a 21 hole photo recorder system. The telemetry oscillator package was located in the aft cockpit seat area; the photo recorder was installed inside the nose, Figure 24.

Telemetry data were recorded at the CAD Telemetry and Data Processing Center where a UHF receiving/tracking system provided real-time data acquisition and direct read-out on strip charts, Figure 25. Audio communication with the pilot was available for convenience and safety monitoring.

Pilot instrumentation controls were located above the cockpit instrument panel, Figure 26, and on the control stick. Data in the two recording systems were related by means of correlator numbers printed on the photo recorder film, and correlator blips on the TM strip chart. A correlator counter could be read by the pilot for reference purposes.

New equipment installed to permit the pilot to monitor and control the AFCAS system were:

- An indicator was provided for direct readout of the motor/pump discharge pressure
- A switch was provided to turn the motor/pump unit "on" and "off"
- An "oil hot" light was set to illuminate when hydraulic fluid in the motor/pump suction line exceeded approximately +200°F (93°C)

Flight data parameters instrumented in the T-2C for the AFCAS program are listed on Table II. Operating range, accuracies, and response capabilities are also listed.

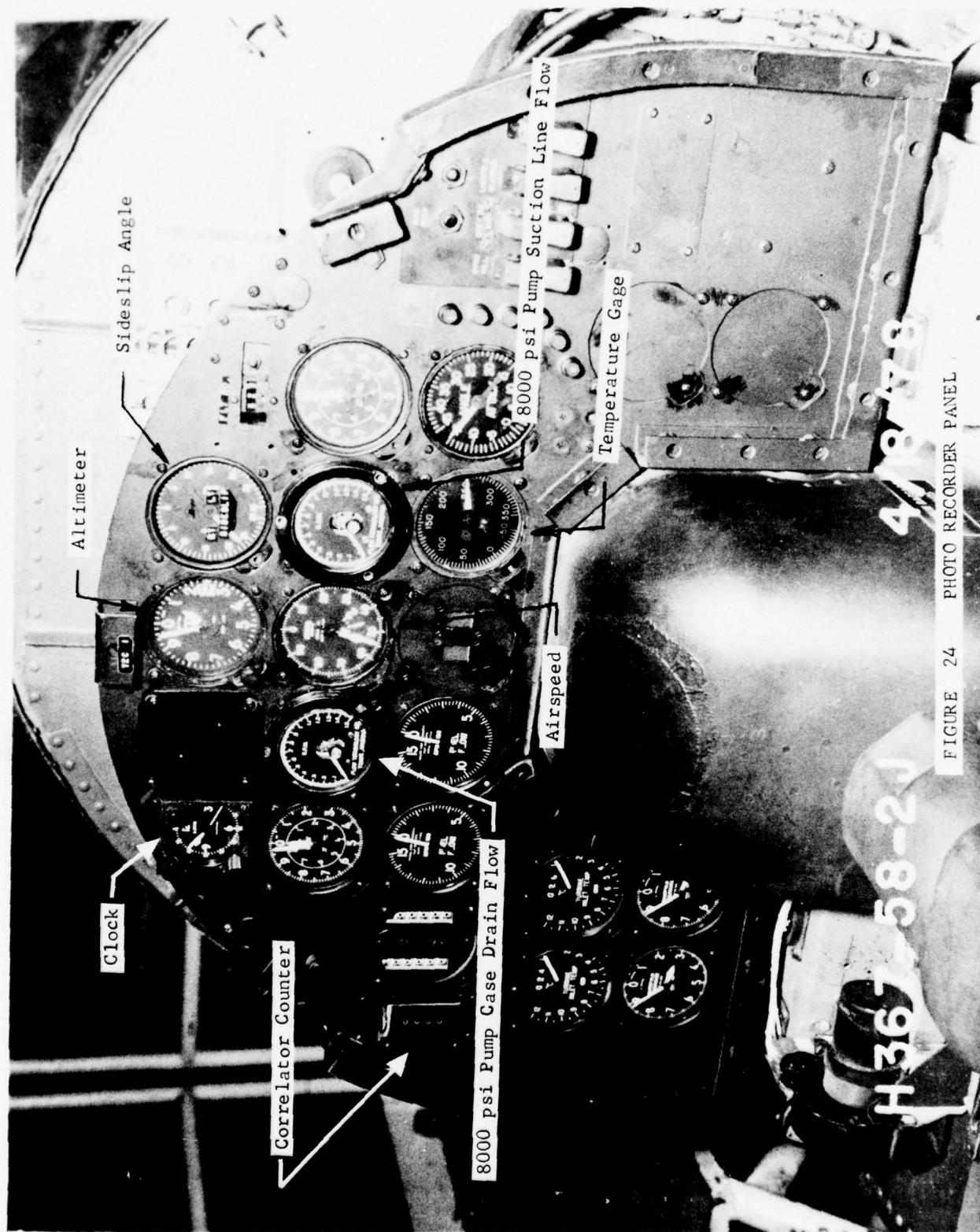


FIGURE 24 PHOTO RECORDER PANEL



FIGURE 25 TELETYPE AND DATA PROCESSING CENTER

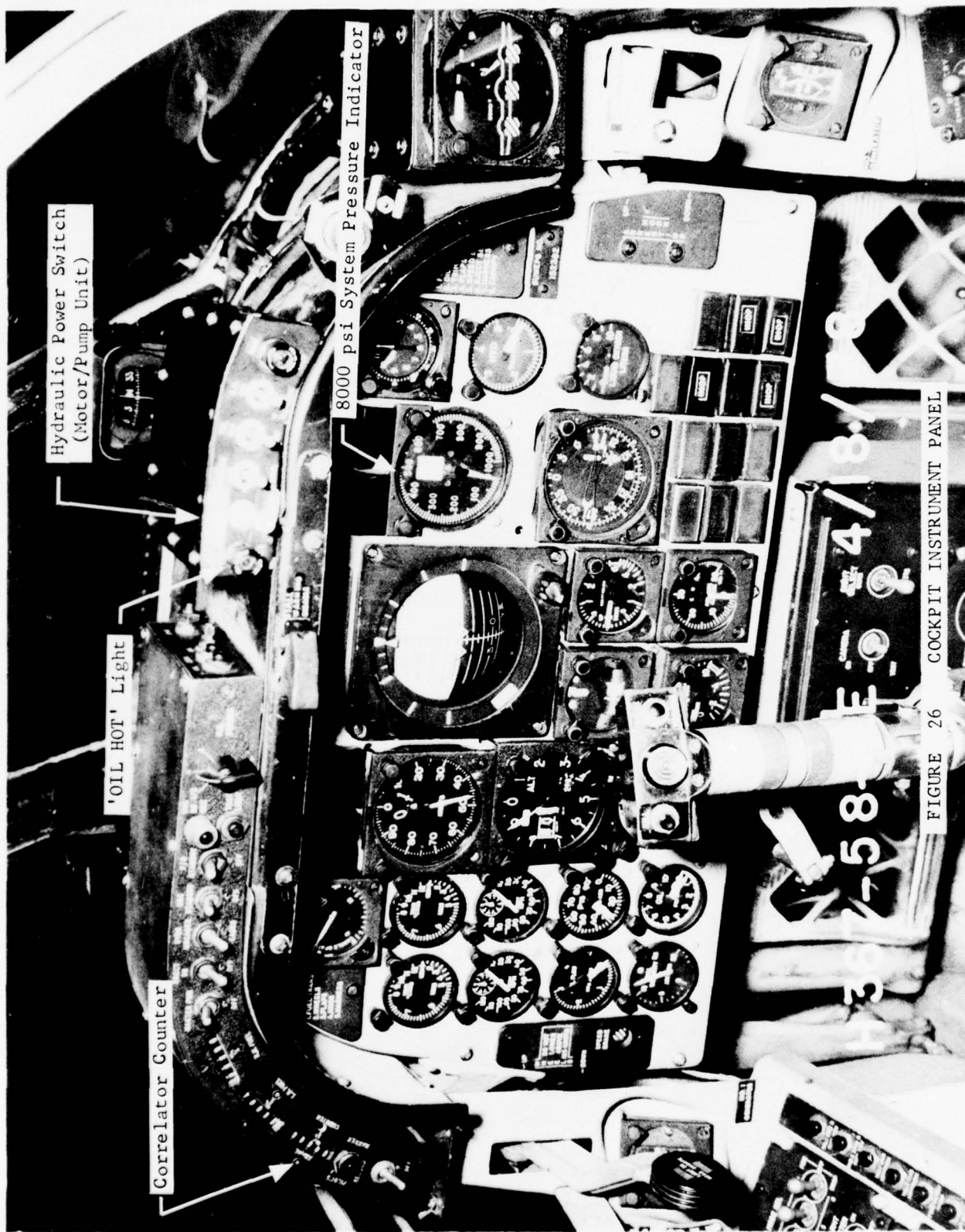


FIGURE 26 COCKPIT INSTRUMENT PANEL

TABLE II LIST OF INSTRUMENTATION

PARAMETER	RANGE	ACCURACY	READOUT RESPONSE
<u>PHOTO RECORDER SYSTEM</u>			
1. Time	N/A		
2. Airspeed	50 to 500 kts (26 to 260 m/s)		
3. Altitude	0 to 50,000 ft. (15.2 km)		
4. RPM, I/R Engines	0 to 8,000 RPM		
5. Fuel Counters, L/R Engines	N/A		
6. Correlation and Pilot Marker	N/A		
<u>AFCAS Parameters</u>			
7. Flow, Pump Case Drain Line	0 to 1.0 GPM (0 to 3.78 L/m)	±2%	2 Hz
8. Flow, Pump Suction Line	0 to 1.0 GPM (0 to 3.78 L/m)	±2%	2 Hz
9. Temp, EDU Housing	-50 to +350°F (-46 to +177°C)	±3%	2 Hz
10. Temp, Fuselage Compartment Air	-50 to +350°F (-46 to +177°C)	±3%	2 Hz
11. Temp, Pump Suction Fluid	-50 to +350°F (-46 to +177°C)	±3%	2 Hz
12. Temp, Pump Case Drain Fluid	-50 to +350°F (-46 to +177°C)	±3%	2 Hz
13. Temp, Heat Exchanger Inlet Fluid	-50 to +350°F (-46 to +177°C)	±3%	2 Hz
14. Temp, Heat Exchanger Outlet Fluid	-50 to +350°F (-46 to +177°C)	±3%	2 Hz
<u>TELEMETRY SYSTEM</u>			
1. Correlation and Pilot Marker	N/A		
2. Temp, Outside Air	-76 to +140°F (+60°C)		
3. Acceleration, Normal (Vertical)	-5 to +10g		
<u>AFCAS Parameters</u>			
4. Press, Pump Suction Line	0 to 50 psia (0 to .3 MPa)	±3%	100 Hz
5. Press, Pump Discharge Line	0 to 10,000 psig (0 to 69 MPa)	±3%	100 Hz
6. Press, Pump Case Drain Line	0 to 100 psia (0 to .6 MPa)	±3%	100 Hz
7. Position, Rudder	±12°	±2%	100 Hz
8. Position, AFCAS Transducer #1	±10 volts DC	±2%	100 Hz
9. Position, AFCAS Transducer #2	±10 volts DC	±2%	100 Hz
10. Force, AFCAS Transducer #1	±2.5 volts DC	±2%	100 Hz
11. Force, AFCAS Transducer #2	±2.5 volts DC	±2%	100 Hz
12. Current, AFCAS Motor Coil #1	±1.0 volts DC	±2%	100 Hz
13. Current, AFCAS Motor Coil #2	±1.0 volts DC	±2%	100 Hz
14. Current, AFCAS Motor Coil #3	±1.0 volts DC	±2%	100 Hz
15. Current, AFCAS Motor Coil #4	±1.0 volts DC	±2%	100 Hz
16. Temp, Oil Hot Light (+200°F)	N/A		

4.0 PREFLIGHT TESTS

4.1 LABORATORY TESTS

4.1.1 Motor/Pump Unit

Pump - The pump was removed from the motor for performance testing, and mounted on a torque meter attached to a varidrive. The test system is described in Reference 12. Data were taken to permit comparisons with rated performance prior to pump de-stroking. Data covering rated performance (from Reference 12) and current performance are shown on Table III. The rework lowered heat rejection approximately 12 percent during zero discharge flow operation (the most prevalent operating mode). Data taken under worst case operating conditions are also given on Table III. Heat rejection increased slightly under these conditions, but this would not significantly affect AFCAS performance.

Motor - The motor was powered with a Hobart 28 volt DC motor-generator rated for 1000 amperes. No. 1 gage stranded copper wire was used between the motor-generator leads and the test motor. Current was measured using a 600 ampere 50 mv shunt, millivolt meter, oscilloscope, and oscillograph with a 240 Hz galvanometer. No-load and load (with pump) operating data were recorded. The results are summarized below:

	<u>No Load</u>	<u>With Pump*</u>
Speed, rpm	8640	8100
Peak starting current, amperes	1200	1200
Starting current >1000 amperes, sec	0.035	0.035
Running current, amperes	20	140
Time to reach full speed, sec	0.4	0.4
Time to stop after shut-down, sec	--	1.1

*Pump powered AFCAS hydraulic system. At start-up, system pressure was zero; after start-up pressure was 8000 psi.

TABLE III

PUMP PERFORMANCE COMPARISONS

PUMP M/N APIV-106, S/N 151353

PUMP SPEED: 7330 RPM

	MAX. DISPL., CIPR	INLET FLUID TEMP., °F/°C	RESERV. PRESS., PSIG	PUMP CASE PRESS., PSIG	DISCH. PRESS., PSIG	DISCH. FLOW, GPM	CASE FLOW, GPM	CASE DR. FLUID TEMP., °F/°C	HEAT REJECT., BTU/MIN
*	.100	110/43	30	50	7500	3.22	.20	184/84	78
**	.016	110/43	30	50	7500	.75	.35	170/77	89
***	.016	110/43	12	80	7500	.68	.29	187/86	100
	.100	180/82	30	50	7500	3.17	.30	239/115	72
	.016	180/82	30	50	7500	.50	.55	230/110	116
	.016	180/82	12	80	7500	.48	.465	246/119	124
	.100	110/43	30	50	8000	0	.57	175/79	142
	.016	110/43	30	50	8100	0	.44	180/82	124
	.016	110/43	12	80	8100	0	.39	202/94	139
	.100	180/82	30	50	8000	0	.75	236/113	179
	.016	180/82	30	50	8100	0	.66	240/116	157
	.016	180/82	12	80	8100	0	.57	260/127	171

* Rated performance. Data taken from Reference 12.

** De-stroked configuration. Same operating conditions as Reference 12 data.

*** 12 psig reservoir pressure and 80 psig pump case pressure are worst case operating conditions.

Metric Conversions:

PSIG X 6895 = Pa

GPM X 3.785 = L/m

BTU/MIN X 17.58 = W

Motor operation was observed to produce significant noise on the supply voltage. Supply voltage ripple was less than ± 0.2 volts DC with the motor off. With the motor running, voltage noise covered a band from approximately 22 to 32 volts DC. The two 24 volt DC batteries in the T-2C were anticipated to be an effective filter which would reduce the noise level in the aircraft installation. To verify this, a 24 volt DC aircraft battery was paralleled into the laboratory system. The supply voltage noise, with the motor running and the battery filter, was reduced to a band from approximately 26 to 29 volts. Two batteries further improved noise filtering, reference Section 4.3.

Motor surface temperature was monitored during simulated flight testing, Section 4.1.4. The temperature stabilized at 60°F (33°C) above ambient after 1.5 hours of driving the AFCAS hydraulic system.

4.1.2 Rudder Actuator

The actuator has three control elements--force motor, flow control valve, and position feedback transducers. Operating characteristics of each of these units were evaluated.

Force Motor - Motor output force and displacement vs. input current were measured using the control valve and housing discussed in Reference 3. (The closed-end design of sleeve P/N SO 4262-03-11 prohibited use of the rudder actuator valve, Reference 4.) Motor output force was measured with a hand-held spring scale applied directly to the spool. Motor output arm displacement was sensed by a dial indicator placed on the opposite end of the spool. The four coils in the motor were connected in series, and an adjustable DC power source was used to apply various amperage levels. Current was measured with a clip-on type DC ammeter. Specific values of current were applied and the force required to position the spool at given displacements from null were measured. The results are presented on Figure 27.

Maximum output force at null was approximately 40 lb (178 N). Saturation began at ± 0.3 amperes and approached 100 percent at ± 0.8 amperes. The restoring force available at ± 0.020 in. (0.76 mm) displacement averaged 62 lb (275 N). (± 0.020 in. was rated spool displacement although displacements up to ± 0.040 in. were possible.) Motor gain at ± 0.020 in. displacement and zero output force averaged 0.033 in/amp (.85 mm/amp) for the coils connected in parallel; the design value was 0.035 in/amp, Figure 19.

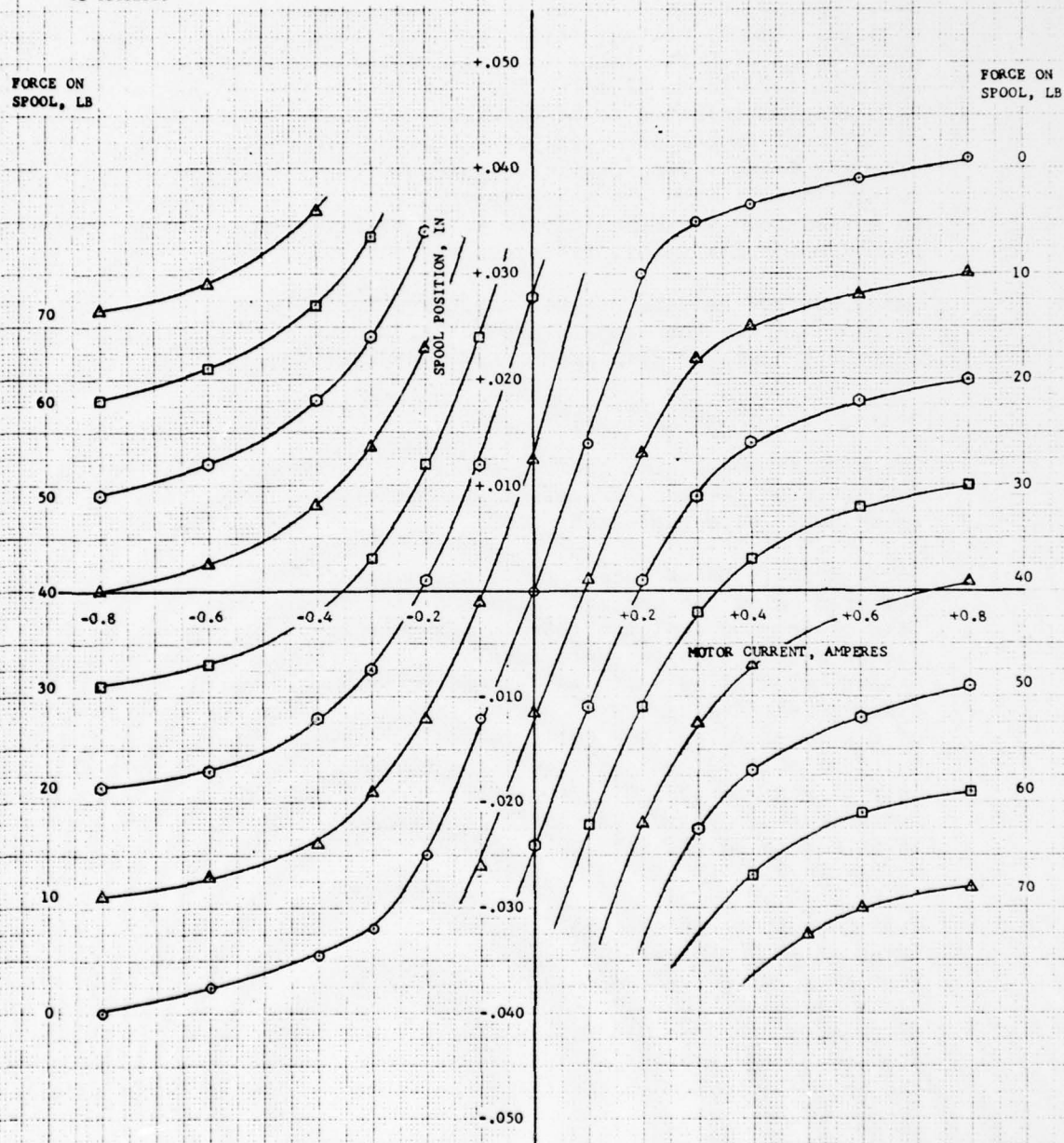
Flow Control Valve - The valve was tested in its housing on the rudder actuator. External ports in the housing were utilized to interconnect the actuator cylinder chambers (C1 and C2). This permitted continuous flow to be controlled through the actuator. A needle valve was put in the line connecting C1 and C2, and gages were installed to measure chamber pressures. Flow was measured with a turbine meter in the actuator return

NOTES

1. Motor output knob inserted into spool hole in spool/sleeve valve. Motor output force measured at spool ends.
2. Motor coils connected in series.
3. + spool position causes actuator to extend; - spool position causes actuator to retract.

METRIC CONVERSIONS

inches X 25.4 = millimeters
pounds X 4.45 = newtons



SERVOTRONICS M/N 21-6-200

FIGURE 27 FORCE MOTOR OPERATING CHARACTERISTICS

line. The valve spool was driven by Servotronics motor M/N 21-6-200 which in turn was driven by an adjustable DC power source. Supply pressure was 8000 psi (55 MPa); inlet fluid (MIL-H-83282) temperature was $+110 \pm 5^\circ\text{F}$ ($43 \pm 3^\circ\text{C}$). Three performance characteristics of the valve/motor assembly were determined: flow gain, pressure gain, and internal leakage.

Flow Gain. The data on Figure 28 show input current vs. output flow characteristics of the motor/valve operating open loop. Flow gain was 4.1 gpm/amp (15.5 L/m per amp) for the extend direction (piston motion), and 2.1 gpm/amp (7.95 L/m per amp) for retraction. The design value was 2.3 gpm/amp (8.70 L/m per amp), Figure 19.

ΔP forces on the spool ends were believed to cause the asymmetrical flow gain. Spool/sleeve type valves usually have exit flow patterns at each end of the spool that are similar, providing approximately equal ΔP forces across the spool for each flow direction. (A hole is sometimes drilled length-wise through the spool to equalize these forces.) The AFCAS valve was designed with one end closed (return flow through the sleeve periphery), one end open (return flow around the large knob on the spool), and no hole through the spool spindle. The closed end of the sleeve housed a position adjustment feature which permitted mechanical alignment of the valve and motor nulls. When return flow was around the spool knob (actuator piston extending), aiding ΔP forces were developed, increasing flow gain, Reference 3. When return flow went through holes in the sleeve periphery, the aiding ΔP forces were not present. Although the motor/valve current/flow gain was asymmetrical, tests on the actuator assembly reported in Section 4.1.3 indicated this discrepancy caused no significant degradation in performance.

Pressure Gain. Pressure gain was determined with ports C1 and C2 blocked. Motor/valve pressure gain was 220,000 psi/amp (1.5 GPa/amp), Figure 29. This value was considered satisfactory. The small null off-set was due to a slight misalignment between the pressure null of the valve and mechanical null of the motor; closed loop operation effectively eliminated this.

Internal Leakage. Internal leakage peaked near null and was 0.058 gpm (220 cc/min). The design value was 0.034 gpm (130 cc/min). The higher than desired leakage was attributed to manufacturing discrepancies. The extra leakage--90 cc/min--was not considered detrimental to system performance.

Feedback Transducers - Position transducer output voltage (efb) vs. core displacement was measured over the range of actuator piston travel (± 1.75 in. or ± 4.44 cm). Data covering piston travel vs. load on the force transducer (F_2) were also taken. The data were linear and were summarized as follows:

NOTES

1. $P_{in} = 8000 \text{ psig (55 MPa)}$
2. $T_{in} = +110 \pm 5^{\circ}\text{F (43} \pm 3^{\circ}\text{C)}$
3. Fluid: MIL-H-83282
4. Motor coils connected in series

FORCE MOTOR ASSY
M/N 21-6-200

SPOOL/SLEEVE ASSY
P/N SO 4262-03-21

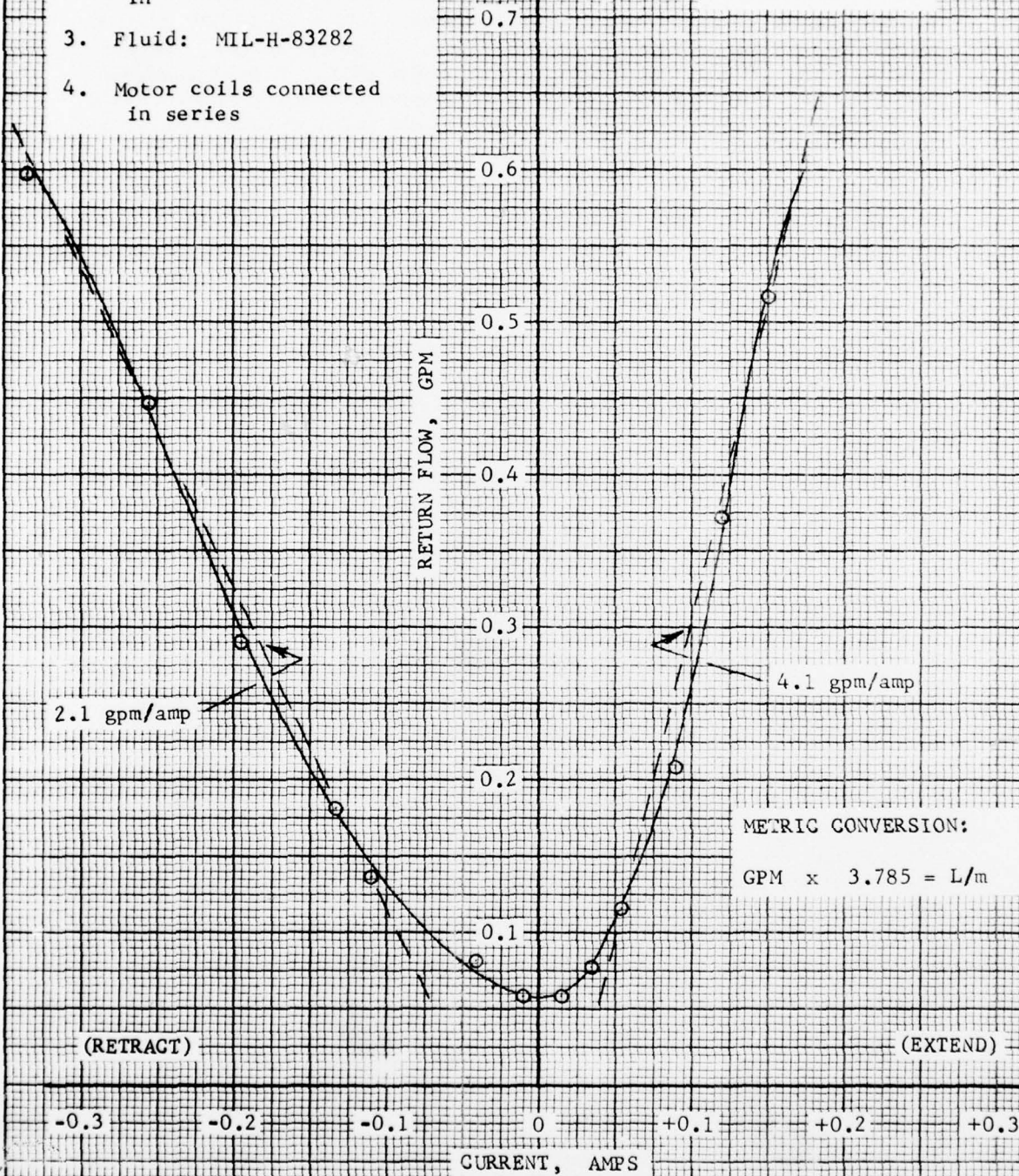


FIGURE 28 FLOW GAIN

FORCE MOTOR ASSY
M/N 21-6-200

SPOOL/SLEEVE ASSY
P/N SO 4262-03-21

NOTES

1. $P_{in} = 8000 \text{ psi (55 MPa)}$
2. $T_{in} = +105^{\circ}\text{F (41}^{\circ}\text{C)}$
3. Ports C1 & C2 blocked
4. Motor coils connected in series

(RETRACT)

(EXTEND)

-0.3

-0.2

-0.1

+0.1

+0.2

+0.3

CURRENT, AMPS

SLOPE = $2.2 \times 10^5 \text{ PSI/AMP}$

LOAD PRESSURE ($P_{C2} - P_{C1}$), PSI

METRIC CONVERSION:

$\text{PSI} \times 6894.8 = \text{Pa}$

FIGURE 29 PRESSURE GAIN

FORCE MOTOR ASSY
M/N 21-6-200

SPOOL/SLEEVE ASSY
P/N SO 4262-03-21

NOTES

1. $P_{in} = 8000 \text{ psi (55 MPa)}$
2. $T_{in} = +108^{\circ}\text{F (42}^{\circ}\text{C)}$
3. Fluid: MIL-H-83282
4. Ports C1 & C2 blocked
5. Motor coils connected in series

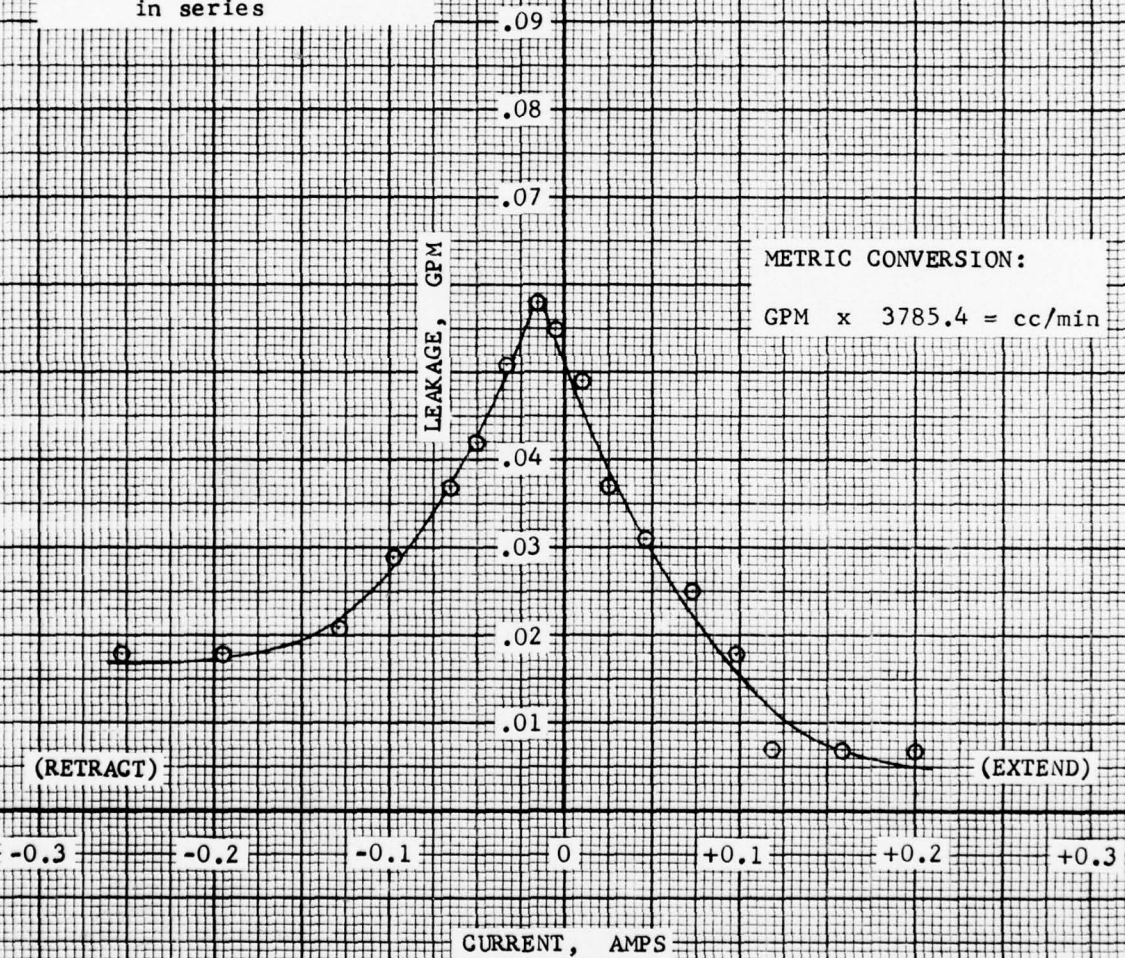


FIGURE 30 INTERNAL LEAKAGE

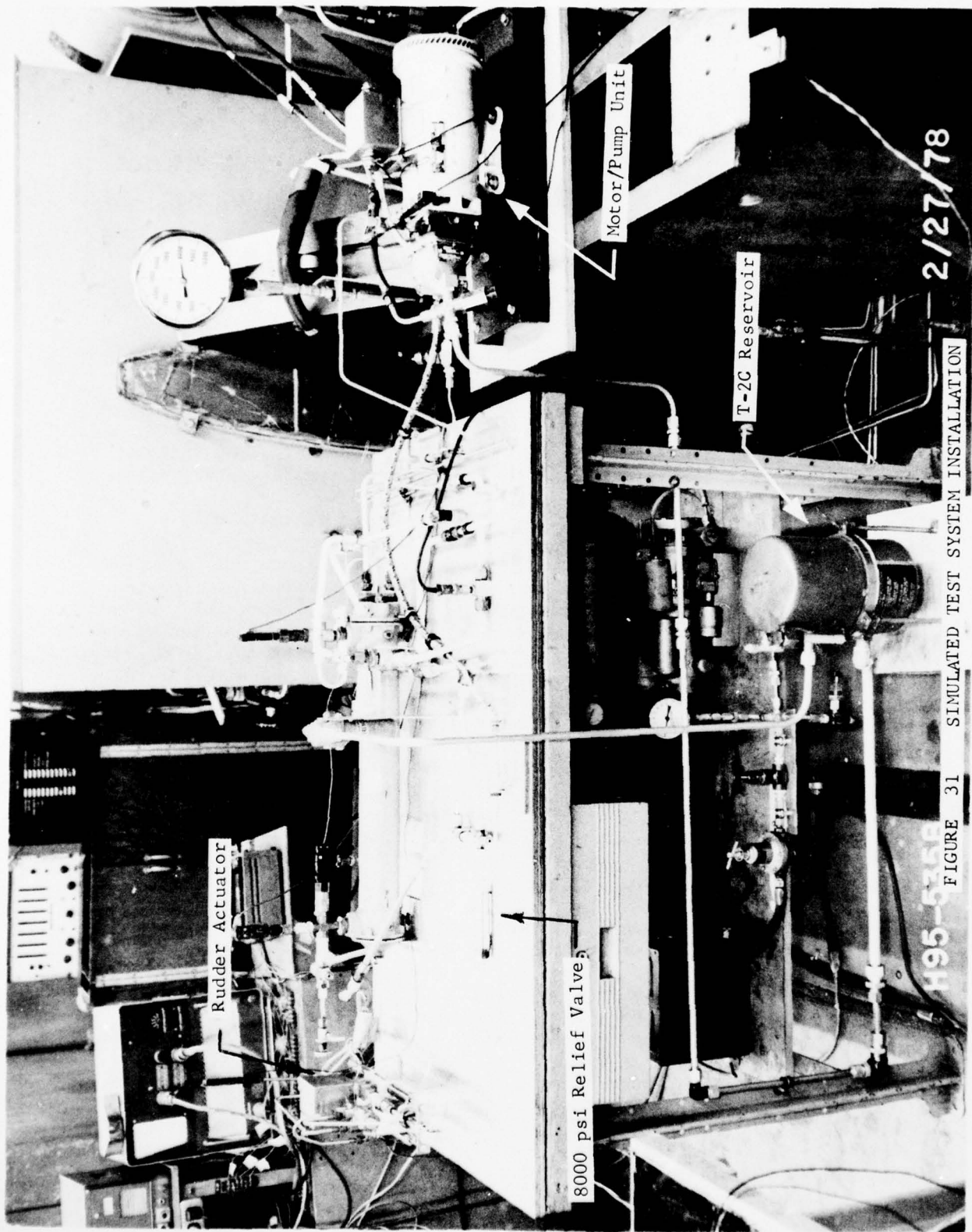
<u>Position Transducer</u>	<u>efb vs. Displacement</u>	<u>F2 vs. Displacement</u>
#1 (S/N 373)	4.7 V/in (1.8 V/cm)	116 lb/in (20.3 kN/m)
#2 (S/N 374)	4.9 V/in (1.9 V/cm)	112 lb/in (19.6 kN/m)

4.1.3 System Integration

The laboratory setup integrating all major components to be installed on the T-2C is shown on Figures 31 and 32. The system contained all the equipment listed on Table I plus the electronic drive unit, force transducer, T-2C hydraulic system reservoir, oil-to-water heat exchanger, and two 10 micron (nominal) filters. An oil-to-air heat exchanger used in the aircraft test installation was not employed in the laboratory setup.

The 8000 psi (55 MPa) laboratory system is depicted schematically on Figure 33. The T-2C 3000 psi (21 MPa) hydraulic system (pump, aileron actuator, elevator actuator, utility functions, etc.) were not included in the laboratory setup because potential benefits derived from testing a complete system did not warrant the added expense. Temperature, pressure, and fluid flow instrumentation were installed at several locations in the setup; this equipment did not simulate flight test instrumentation, Section 3.5. Hydraulic line lengths and sizes used in the aircraft test installation were duplicated (as nearly as practical) in the laboratory setup. Tubing bends were not simulated, however, all elbow type fittings employed in the aircraft were put in the laboratory system. High pressure tubing was 1/4 x .025 in. 21-6-9 CRES. All 8000 psi tubing connectors were standard MS flareless fittings. (8000 psi fittings used in the aircraft were "Dynatube" series.) Static seals were MS 28778 O-rings. The system contained approximately 2.0 gal (7.6 L) of MIL-H-83282 fluid. The reservoir was placed 28 in. (71 cm) below the pump suction port to simulate aircraft orientation.

Fluid flow in the pump case drain and actuator return lines was measured by turbine meters with readout on frequency counters. Static pressures were monitored with bourdon tube dial gages teed into the pump suction, discharge, and case drain lines. Dynamic pressures were recorded on a high response oscillograph using strain gage type transducers plumbed in the pump discharge and actuator return lines. Fluid temperatures were sensed by thermocouples at seven locations: pump suction line, case drain line, actuator return line, heat exchanger inlet and outlet ports, pump motor surface, and ambient air. Temperature readout was on a multi-channel, selector-button type indicator. Fluid temperature stabilization was achieved by means of a duration-adjusting type controller and the oil-to-water heat exchanger.



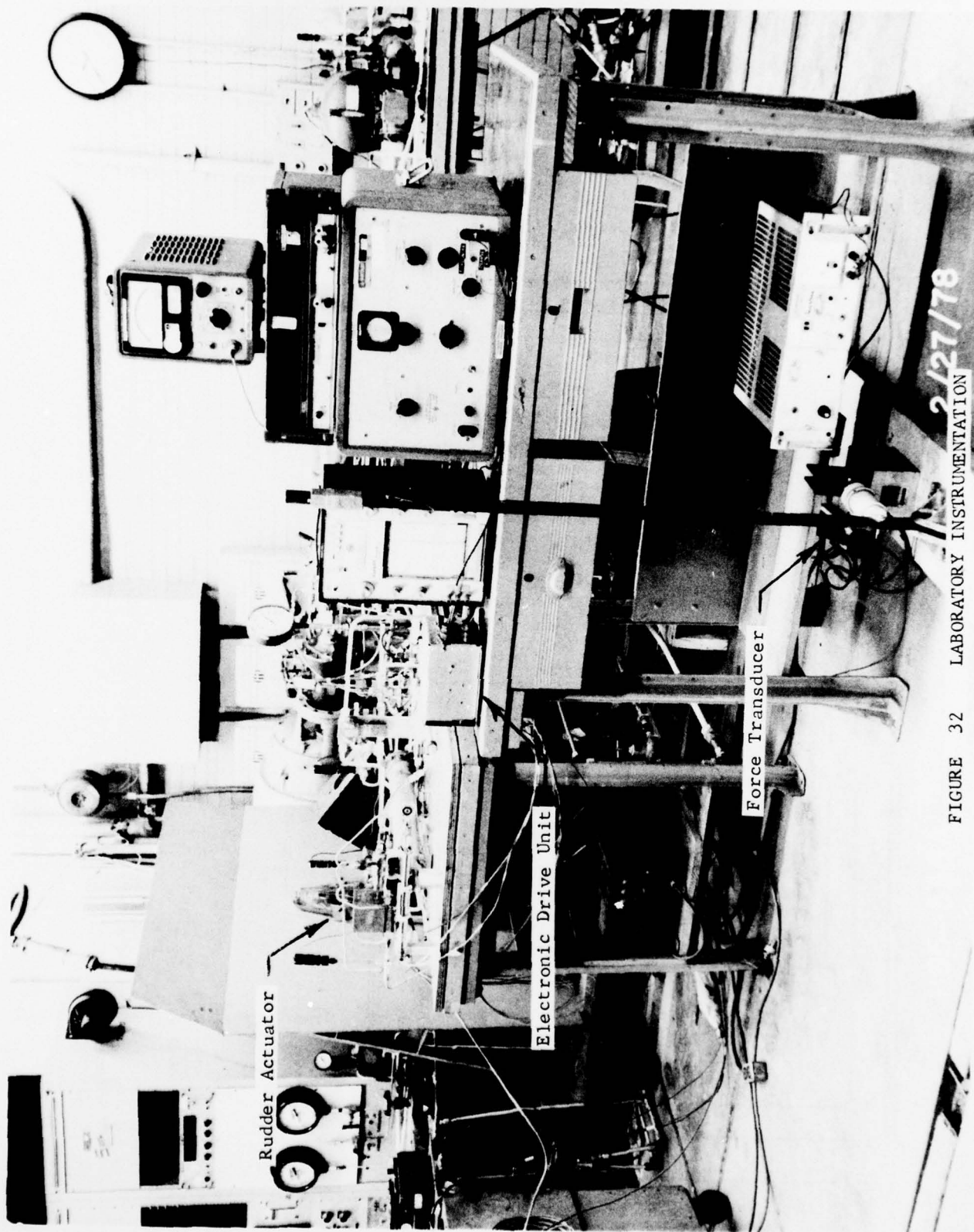


FIGURE 32 LABORATORY INSTRUMENTATION



FIGURE 33 SCHEMATIC DIAGRAM OF SIMULATED TEST SYSTEM HYDRAULIC INSTALLATION

Electrical components in the system are shown on Figure 32. The force transducer was installed in a fixture with a lever for applying simulated rudder pedal loads. Laboratory harness was made for interconnecting the EDU with the rudder actuator and force transducer. A terminal board was built which permitted voltage checks at 24 test points within the EDU. A function generator was used to apply sinusoidal and step commands to the EDU. Input and output (command signal and actuator piston position) were recorded simultaneously on a dual channel strip chart for response evaluations. A 400 cycle 110 volt AC power supply, digital voltmeter, and clip-on DC ammeter completed the instrumentation.

Initial Checkout Tests - The first tests were continuity and circuit checks on the EDU. Several minor wiring errors and omissions were found and corrected. Power was applied and voltage checks were made at 24 test points, Figure 20. After all circuitry was confirmed, and EDU operation was determined to be satisfactory, the transducer nulls were adjusted. The position transducer cores were set so that when the actuator piston was mid-stroke, transducer output was less than ± 0.100 volts DC. The force transducers were removed from the housing and their cores were adjusted so that, with no load applied, transducer output was less than ± 0.125 volts DC. With all the transducers at null, the current in each of the motor coils was then adjusted to zero by external bias pots on the EDU.

System Integration Tests

Hydraulic System Stability - Persistent pressure oscillations were observed in the 8000 psi (55 MPa) circuit during initial start up of the system. The oscillations were 800 psi (5.5 MPa) peak-to-peak at 25 Hz and considered excessive. Attempts were made to attenuate the oscillations by adding various amounts of fluid volume at different locations in the system. Eight configurations were tried with no success. After consultation with the pump manufacturer (Abex), it was decided to turn the pump compensator spring end-for-end and determine if variations in the squareness of the ground ends were affecting stability. This was done and the stability problem was corrected; discharge pressure was a steady 8000 psi (55 MPa) (no oscillations) and pump ripple was less than ± 100 psi (.7 MPa).

Pressure Surges - Magnitudes of peak pressures measured during system start-up and actuator hard-over commands are summarized below. The surges were well below the 120 percent (9600 psi) maximum allowable, Reference 8.

<u>Location</u>	<u>Start-Up</u>	<u>Hard-Over Commands</u>
Pressure Line (60 in. from pump)	8400 psi (58 MPa)	8350 psi (57.6 MPa)
Return Line (at rudder actuator)	12 psig (83 kPa)	50 psi (345 kPa)

System Temperatures - Based on data reported in Reference 13, fluid temperature in the pump suction line could be expected to range from +130 to +140°F (54 to 60°C). Results of the laboratory test with fluid temperature stabilized at +132°F (57°C) in the suction line are summarized below. The temperatures, flows, and heat removed were all considered satisfactory.

<u>Pump Suction</u>	<u>Pump Case Dr.</u>	<u>Actuator Return</u>	<u>Ambient Air</u>	<u>Pump Case Dr.</u>	<u>Actuator Return</u>	<u>Heat Removed By Heat Exchanger</u>
+132°F	+214°F	+151°F	+85°F	0.56 gpm	0.07 gpm	157 BTU/min
+57°C	+101°C	+66°C	+29°C	2110 cc/min	275 cc/min	2.7 kW

System Operation - Manual operation of the force transducer, reference Figure 32, provided a subjective indication of overall system performance. Actuator piston motion was controlled positively, with little overshoot and little apparent hysteresis. Sensitivity was excellent. (Less sensitivity could be obtained, if desired, by simple adjustments in the EDU.) Minimum time for full stroke (24° rudder deflection) was approximately 0.5 sec. Pump speed was noted to sag slightly during actuator stroking (pump loaded) and return to full speed when the pump was unloaded. No external leakage was observed except for normal wetting of the actuator piston rod.

The general consensus of several engineers and a pilot who operated the force transducer lever was the laboratory system functioned very well.

Response Characteristics - Overall system response was determined from square wave, saw tooth, and sinusoidal inputs applied to the EDU by a function generator while the rudder actuator was pressurized at 8000 psi. (Force transducer inputs were not used.) The input signal and output piston motion were recorded simultaneously.

Response to 0.5 Hz square wave and saw tooth inputs are shown on Figure 34. Slight rounding at the saw tooth peaks was due to control valve dead band; this was not significant considering the amplitude and frequency of the data. Overshoot peaks on the square wave were near optimum.

Frequency response was determined for three amplitudes at 0.5 Hz: ± 0.050 , ± 0.100 , and ± 0.200 in. (± 1.27 , ± 2.54 , and ± 5.08 mm). The 3 dB bandwidth for ± 0.050 , ± 0.100 , and ± 0.200 in. amplitudes were 9, 13, and 11 Hz, respectively, Figure 35.

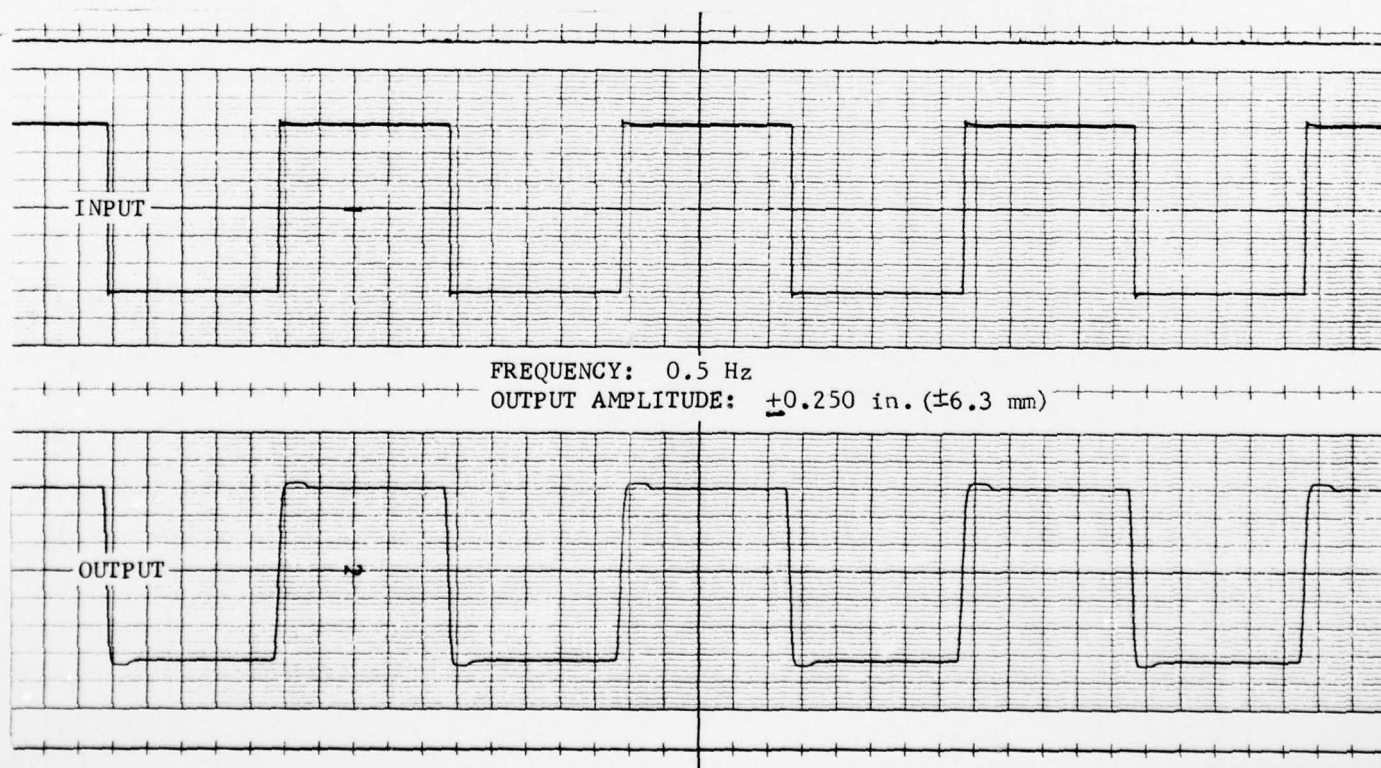
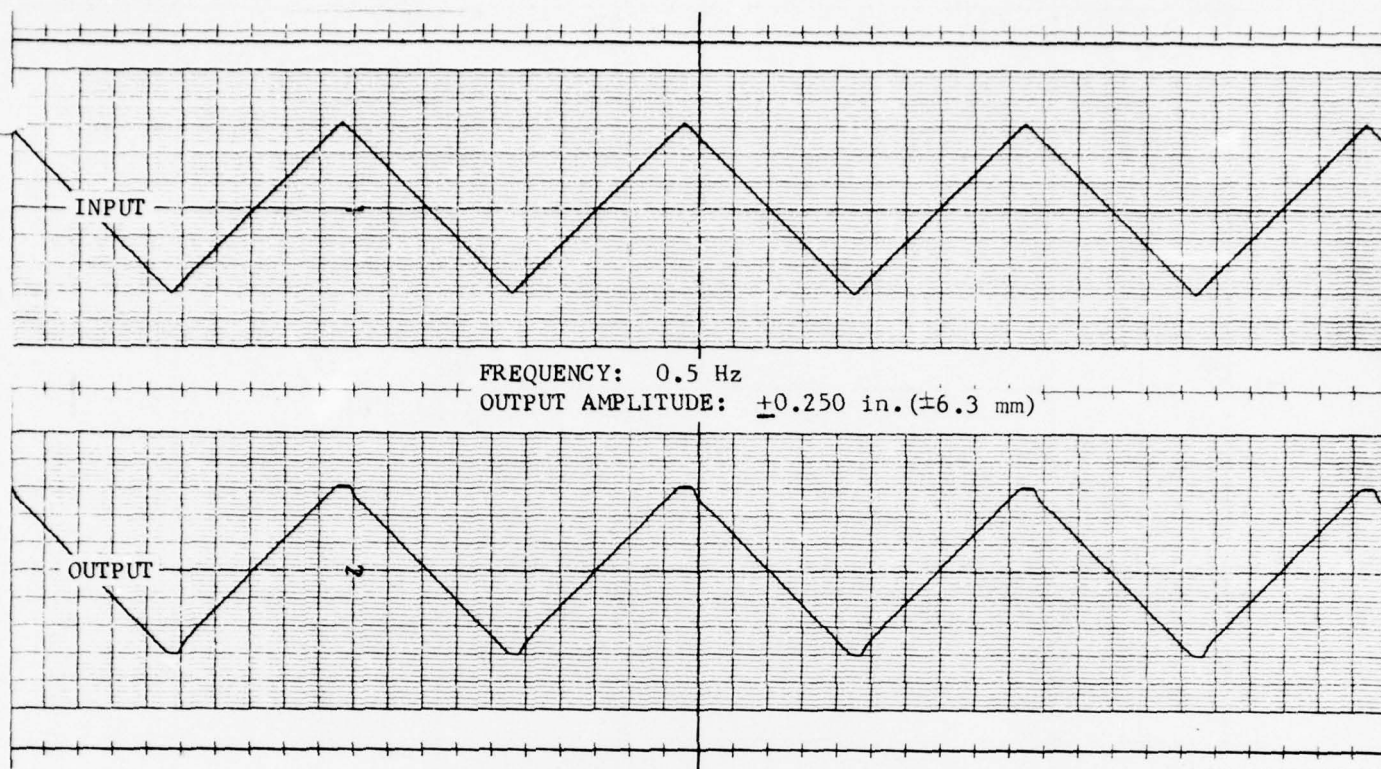


FIGURE 34 SAW TOOTH AND SQUARE WAVE RESPONSE

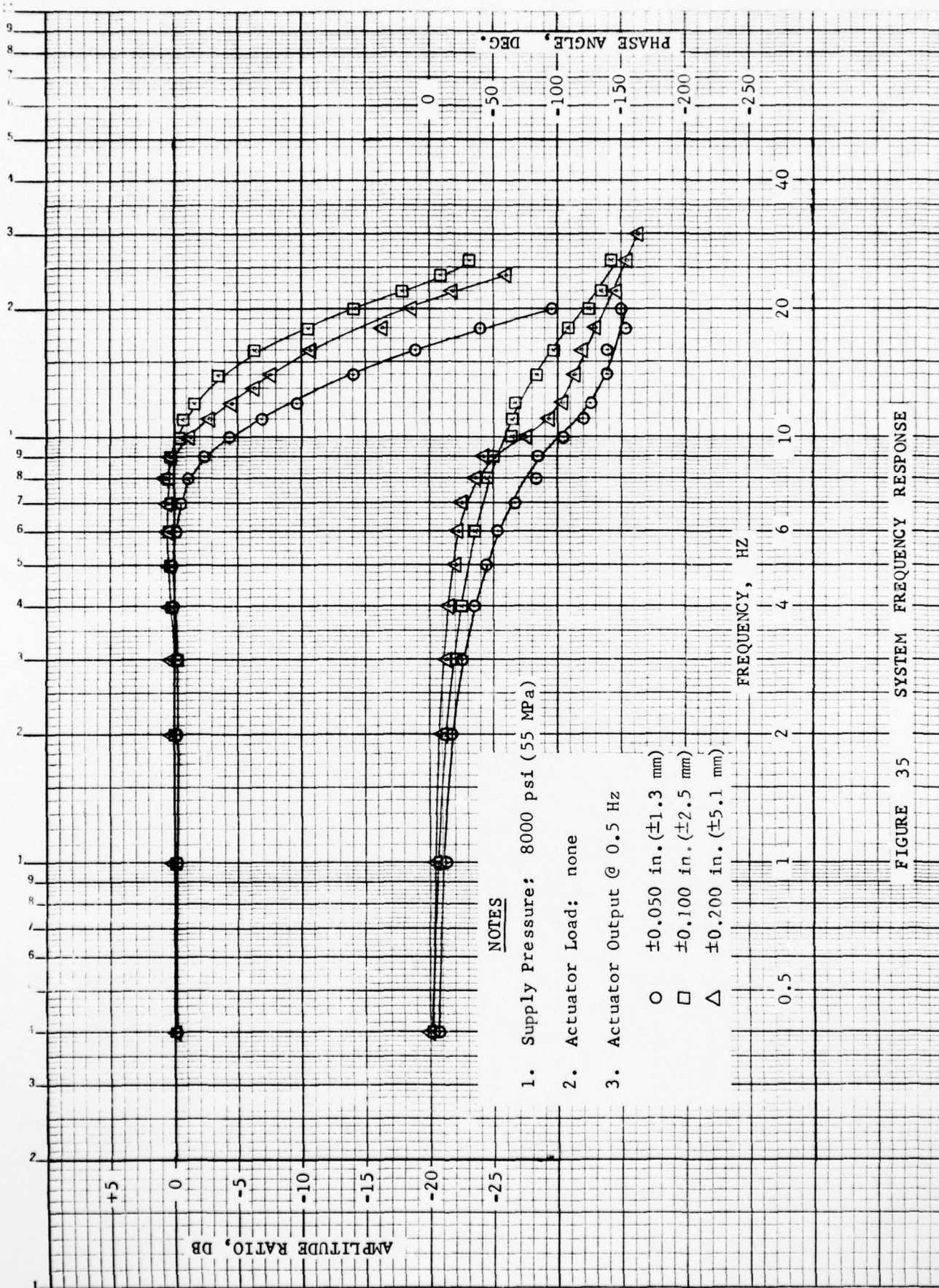


FIGURE 35 SYSTEM FREQUENCY RESPONSE

Higher response could be achieved by adjusting loop gain. Because of T-2C directional system dynamics the AFCAS response utilized was approximately 3 Hz. This bandwidth proved entirely satisfactory for the test installation, Section 5.0.

4.1.4 Simulated Flight Testing

A total of approximately 10 flight hours were to be accumulated on the AFCAS test installation, Section 5.0. The laboratory simulation consisted of six 1-1/2 hour "flights". The following schedule profiles a typical flight:

<u>Flight Phase</u>	<u>Duration, Min.</u>	<u>Rudder Operation</u>
Ground checkout and taxi out	15	Periodic
Take-off	7.5	Periodic
Cruise	45	Periodic
Full-power flight	7.5	Periodic
Landing and taxi in	15	Periodic
<hr/>		
Total	90	

All tests were conducted with minimum reservoir pressurization (12 psig/ 83 kPa) and maximum case drain pressure (80 psig/551 kPa). Pump suction line fluid temperature was maintained in the range of +130 to +140°F (+54 to 60°C). The following parameters were tracked during the test:

- (1) Pump case drain flow
- (2) Actuator null flow
- (3) External leakage (pump and actuator)
- (4) Pump motor current
- (5) Actuator motor null current
- (6) Pump case drain filter debris (patch test)

Performance data taken before and after six 1-1/2 hour simulated flights are summarized on Table IV.

TABLE IV

SIMULATED FLIGHT PERFORMANCE SUMMARY

	<u>BEFORE TEST</u>	<u>AFTER SIX "FLIGHTS"</u>
Pump Case Drain Flow (at +212°F/100°C)	0.53 gpm (2000 cc/min)	0.56 gpm (2120 cc/min)
Actuator Null Flow (at +160°F/71°C)	0.07 gpm (270 cc/min)	0.07 gpm (275 cc/min)
Pump External Leakage	None	None
Actuator External Leakage	None	Trace (at rod seal)
Pump Motor Current (after warm-up)	156 amperes	144 amperes
Actuator Motor Null Current		
Coil #1	0	+0.02 ampere
Coil #2	0	+0.03 ampere
Coil #3	0	-0.01 ampere
Coil #4	0	-0.03 ampere
Pump Case Drain Filter Debris (from bowl and element)	Clean	<ol style="list-style-type: none"> 1. Normal quantity of metallic wear particles. 2. Large number of very small black particles.

Pump performance was satisfactory throughout the test. System pressure was a steady 8000 psi (55 MPa) with nominal pressure fluctuations during rudder actuator operation. The pump had no external leakage or leakage at the shaft seal. A normal quantity of visible metallic wear particles collected in the case drain filter. Total operating time on pump M/N APIV-106 prior to AFCAS flight testing is summarized below:

	<u>Hours</u>
LHS testing reported in Reference 12	52
LHS testing reported in Reference 13	16
Miscellaneous testing	2
Abex tests (rework to 0.016 CIPR)	5
AFCAS component tests (reported herein)	14
AFCAS simulated flight tests	<u>9</u>
Total	98

Actuator performance was satisfactory throughout the laboratory test. No external leakage occurred except for normal wetting of the piston rod.

Motor running current decreased somewhat during the course of testing (from 156 to 144 amperes). The 1-1/2 hours of continuous operation during a "flight" did not produce overheating; motor surface temperature never exceeded +141°F (61°C).

Operation of the force transducer and EDU was satisfactory. Only slight drifting was observed in the EDU as evidenced by the motor null currents. Response characteristics of the system were unchanged by the test.

A large number of black particles less than 1 micron in size (0.00004 in.) were flushed from the filter element when a patch test was made of debris in the pump case drain filter. This type of particle was observed in a prior LHS test, Reference 11. The source and composition of the particles were not determined; Reference 14 may provide some insight into this. Mobil Oil Corporation has recently developed fluid analysis techniques which disclosed that nitrogen-fixing and thermal cracking cause oil degradation in 3000 psi systems; this may also be occurring at 8000 psi. Effects of the black particles on the operation of 8000 psi systems were considered minimal in view of the excellent performance characteristics observed thus far in both the LHS and AFCAS programs. An experiment involving the use of an inert gas (argon) for reservoir pressurization is currently in progress in an LHS test. This should minimize oxidative-type thermal/chemical reactions and provide an indication whether these reactions (if present) are producing the black particles.

4.2 HANGAR TESTS

4.2.1 Electrical Checks

Procedure details for checking the electrical system are given in Appendix A. A summary of steps taken to assure proper operation of the test system in the T-2C is presented in the following paragraphs.

Continuity checks were made on all AFCAS electrical harness newly installed in the aircraft. Chassis ground checks were conducted to verify that specified pins in the EDU A/C disconnect were grounded. 400 Hz 115 volt power was applied to the harness and measurements were taken to determine that voltage appeared on specified pins in the EDU disconnect. Power was applied to the EDU only. (AFCAS electrical harness was not connected to the force transducers or force motor.) The transducer supply voltage (+15 VDC) was verified on each transducer disconnect. All the foregoing preliminary checks were completed satisfactorily.

The AFCAS harness was connected to the transducers and force motor. With power applied to the EDU, the rudder actuator piston was moved (by manually deflecting the rudder) so that position transducer output was within ± 0.100 volts. The bellcrank-to-surface push rod was then adjusted to obtain $0 \pm 1/4^\circ$ rudder position. Transducer and valve driver output voltages were recorded. Sufficient force was then applied (alternately) to the rudder pedals to produce between 1 and 2 volts DC on the force transducer outputs. The corresponding LED illumination on the AFCAS test box was observed (indicating four hard-over signals). No problems were encountered with any of the foregoing steps.

4.2.2 Hydraulic Checks

Procedure details are given in Appendix A. The first task involved filling and bleeding the 8000 psi (55 MPa) system. The rudder actuator pressure and return lines were temporarily connected together and temporary plumbing was used to join the pump suction line with the discharge and case drain lines (bypassing the 8000 psi pump). A bleed valve was installed in the return line in the RH speed brake well. A ground cart was connected to the aircraft and the system was filled with MIL-H-83282 fluid. With 25 psig (.2 MPa) applied to the T-2C reservoir, air was bled from a port on the heat exchanger and from the bleed valve in the return line.

A leak check was made on the 8000 psi portion of the system using a 0.2 gpm (.76 L/m), 10,000 psi (69 MPa) power supply containing an adjustable relief valve. Pressures up to 8,000 psi were applied; no external leakage occurred. Pressure was increased sufficiently to operate the test system relief valve (9000 psi/62 MPa). No leakage or malfunctions were observed.

The T-2C 3000 psi system was then pressurized with a service ground cart and the various subsystems were operated. No problems were encountered.

4.2.3 System Checkout

System checkout procedure is detailed in Appendix A. Twenty-five psig was applied to the T-2C reservoir and a 300 ampere, 28 volt DC rectifier-type power supply was connected to the aircraft. With electrical power on the aircraft, the motor/pump unit was energized. The cockpit gage was observed to read 8000 psig (55 MPa). Operation of the 8000 psi hydraulic system was satisfactory; no malfunctions or leaks occurred.

The rudder pedals were operated. Rudder control was smooth and positive. A small amount of hysteresis was noted due to normal friction in the cables, pulleys, and bellcranks used in the T-2C directional system. Rudder operation details were determined and are summarized below. Pedal force vs. rudder deflection data are presented on Figure 36.

<u>Description</u>	<u>Data</u>
Maximum rudder deflection	11.8° Right 11.8° Left
Pedal force required for 11.8° rudder	93 lb (414 N)
Pedal deflection at 11.8° rudder	0.5 in. (1.3 cm)
Dead band at 0° rudder with cable/pulley friction (no pedal corrections)	1°
with cable/pulley friction minimized (pedals alternately tapped lightly)	1/4°
Moment required to cause rudder to trail with no pressure on rudder actuator, breakout	L→T 325 lb-in (37 N-m) R→T 455 lb-in (51 N-m)

All of the above data were satisfactory.

Temperature of the fluid in the pump suction line versus time was checked with the heat exchanger blower running. The temperature stabilized at +179°F (82°C) after 25 minutes of operating at 8000 psi. In a second test conducted with the heat exchanger blower off, the suction line fluid temperature reached +200°F (93°C) in 24 minutes; stabilization was not achieved.

A pressure transducer was installed (temporarily) in the 8000 psi system pressure line to measure surges and oscillations. Readout was on an oscillograph using a galvanometer with 600 Hz response capability. The maximum

METRIC CONVERSION:

1b X 4.45 = newtons

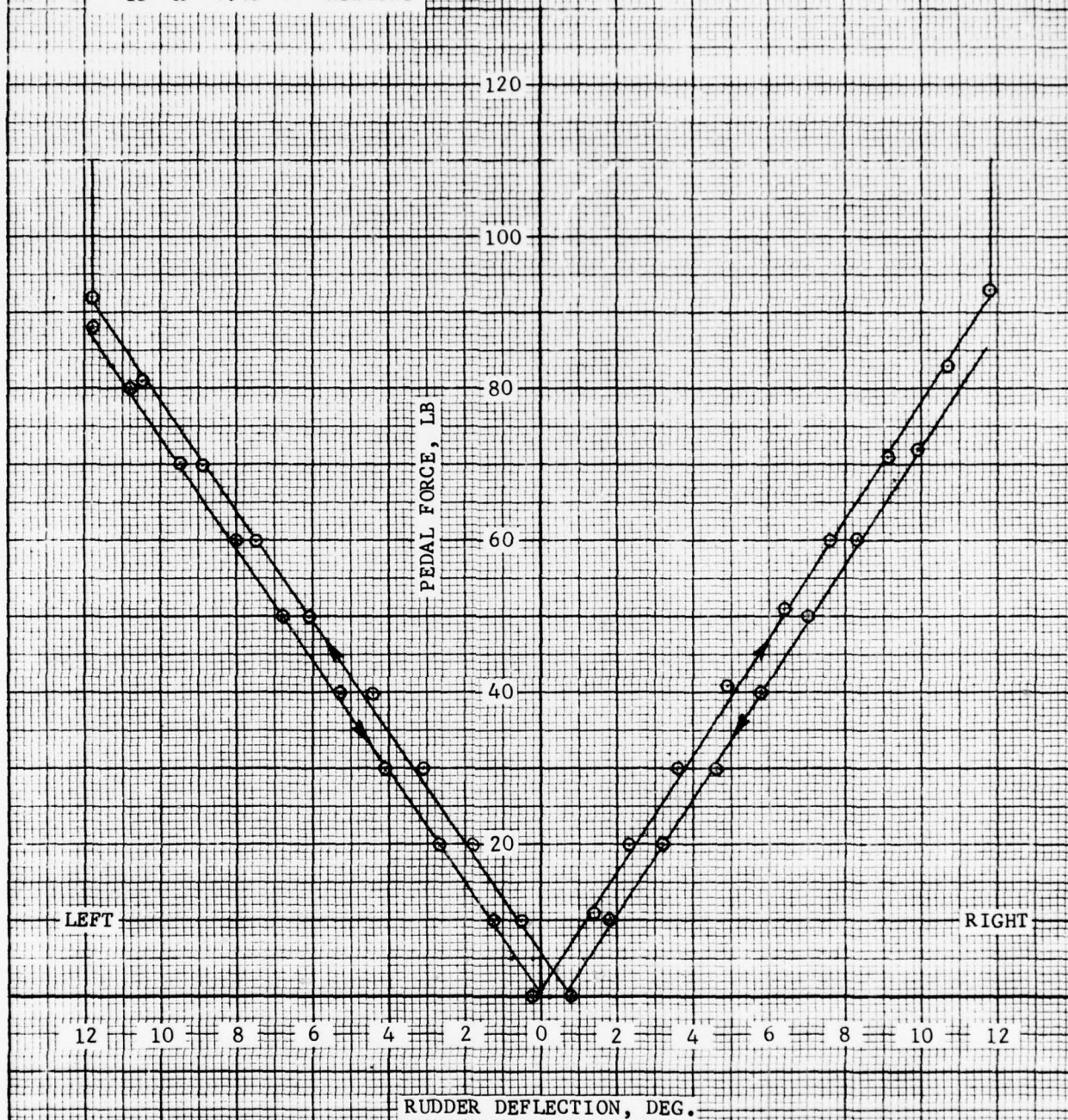


FIGURE 36 PEDAL FORCE VS. RUDDER DEFLECTION

pressure overshoot observed with hard-over inputs to the rudder actuator was 200 psi (1.4 MPa). Pump ripple was less than 100 psi (0.7 MPa) peak-to-peak. No persistent pressure oscillations were present. The 8000 psi system was exceptionally "quiet" with respect to pressure fluctuations.

A visual inspection was made of hydraulic lines in the 8000 psi system to determine if any destructive vibrations were present when the system was operating. No excessive vibratory conditions were observed.

A fluid sample was taken at the conclusion of hangar testing to determine system contamination. Particulate size and quantity were measured with a HIAC automatic particle counter. Contamination was well below NAVAIR 01-1A-17 Class 5 allowables, and was equivalent to a Class 3, Page 95.

4.3 GROUND DEMONSTRATION TESTS

This test was conducted to simulate a one hour flight from takeoff to landing, and provide a means to final check hydraulic system operation and instrumentation. Engine speed, stick, and pedal inputs were varied to simulate actual flight. Both engines and the motor/pump unit were run continuously during the test. A portable potentiometer was used to monitor fluid temperature in the suction line of the 8000 psi pump. A summary of the "flight" is given below; details are given in Appendix A.

<u>Description</u>	<u>Engine Speed, % MRT</u>	<u>Duration, Min.</u>
System checkout & Taxi Out	48%	10
Takeoff	100%	4
Cruise & Maneuver	80 to 100%	36
Landing & Taxi-In	48%	10

The engine access, hydraulic reservoir, and fuselage compartment doors were open and the vertical stabilizer side panel was removed. Inspections for hydraulic leaks and line vibrations were made during the simulated flight. No leaks or excessive line vibrations were observed. A check was made to determine the magnitude of the ripple imposed on the supply voltage by operation of the motor/pump unit. An oscilloscope was used to measure the noise at the 28 volt DC bus. With both engines operating at idle, the induced ripple was approximately 1 volt peak-to-peak. This was considered acceptable for the AFCAS program.

The simulated flight was stopped prematurely when aircraft fuel was erroneously depleted after 43 minutes of satisfactory testing. Completion of the remaining 17 minutes of run time was not considered necessary since the most severe parts of the test (90% and 100% engine speeds) were finished, reference Appendix A.

METRIC CONVERSIONS

in.	X	2.540	=	cm
ft	X	.3048	=	m
lb	X	4.448	=	N
psi	X	6895	=	Pa
K	X	.5144	=	m/sec

OIL HOT LIGHT

GROUND DEMONSTRATION TEST

Engine RPM: 100%

Maneuver: Hard over pedal inputs

Light Flickering

Light Off

CORRELATION MARKS

10 Sec

1150

Chart Time Synch

1200

Elapsed Time = 38 Min

12° RIGHT

RUDDER POSITION

12° LEFT

2 in. (RETRACT)

ACTUATOR POSITION TRANSDUCER #1

2 in. (EXTEND)

2 in. (RETRACT)

ACTUATOR POSITION TRANSDUCER #2

2 in. (EXTEND)

500 lb (COMPR.)

FORCE TRANSDUCER #1

500 lb (TENSION)

500 lb (COMPR.)

FORCE TRANSDUCER #2

500 lb (TENSION)

+2 AMPERES (EXTEND)

VALVE MOTOR COIL #1

-2 AMPERES (RETRACT)

ACCUCHART

Gould Inc. Instrument Systems Division

+2 AMPERES (EXTEND)

VALVE MOTOR COIL #2

GROUND DEMONSTRATION TEST

Engine RPM: 100%

Maneuver: Hard over pedal inputs

-2 AMPERES (RETRACT)

+2 AMPERES (EXTEND)

VALVE MOTOR COIL #3

Chart Time Synch

-2 AMPERES (RETRACT)

+2 AMPERES (EXTEND)

VALVE MOTOR COIL #4

-2 AMPERES (RETRACT)

+10 G

NORMAL (VERTICAL) G'S

1 g

50 psia

PUMP SUCTION PRESSURE

34 psia (avg)

14,100 psig

PUMP DISCHARGE PRESSURE

7890 psig

100 psia

PUMP CASE PRESSURE

62 psia (avg)

+60°C

OUTSIDE AIR TEMPERATURE

-60°C

+14°C/+58°F

Outside air temperature was 58°F/14°C; fuselage bay air increased from 70°F/21°C to 94°F/34°C during the run. Hydraulic fluid temperatures are summarized below. Flow in the 8000 psi pump case drain averaged 0.85 gpm (3.2 L/m). TM data are presented on pages 80 and 81.

<u>Elapsed Time, Min</u>	<u>Engine Speed</u>	<u>Pump Suction</u>	<u>Temperature, °F/°C</u>		<u>Heat Exch. Outlet</u>
			<u>Pump Case Dr.</u>	<u>Heat Exch. Inlet</u>	
1	48%	95/35	165/74	160/71	139/59
9	48%	120/49	195/91	190/88	166/74
14	100%	143/62	213/101	209/98	181/83
25	90%	168/76	227/108	224/107	191/88
28	100%	167/75	233/112	230/110	194/90
34	90%	170/77	233/112	230/110	194/90
37	100%	173/78	234/112	230/110	194/90
40	90%	174/79	235/113	232/111	194/90

Total operating time on the aircraft test installation prior to flight testing was:

	<u>Hours</u>
Hangar Tests	4.0
Ground Demonstration Tests	0.7
	<hr/>
Total	4.7

5.0 FLIGHT TESTS

5.1 FLIGHT PLAN

The primary objective was to verify the feasibility of the Advanced Flight Control Actuation System (AFCAS) concept by flight testing a control-by-wire, direct-drive actuation system powered by a localized 8000 psi (55 MPa) motor/pump unit. Demonstration of flying qualities was not part of the program, however pilot comments were encouraged. Ten flight hours were expected to be sufficient to evaluate AFCAS performance, confirm prior analyses and laboratory testing, and provide a measure of confidence in system reliability.

The flight plan was designed to determine directional control characteristics at several altitudes up to 30,000 ft. (9.1 km) and various speeds up to 340 knots (174 m/s). The first two flights were dedicated to confirming satisfactory operation. Subsequent flights were scheduled to evaluate system performance and reliability while accumulating 10 flight hours. Flight plan details are given in Appendix A.

5.2 RESULTS

The AFCAS flights are summarized on Table V. Two pilots participated in the program and prepared reports detailing each test flight. Additional comments were made during flight de-briefings. Both pilots stated that performance of the AFCAS test installation was completely satisfactory. Comments made by the pilots concerning their flights were:

- The AFCAS installation worked exactly as designed
- No malfunctions occurred
- System pressure was steady
- Hydraulic fluid temperatures were normal
- Directional control response was judged to be superior to the production T-2C
- Pilot adaptation to "force control" of the rudder was quickly and easily acquired. Reaction of the aircraft provided the clues to close the loop.
- The force system had an advantage during take-offs and landings in high cross-winds. The fixed pedals provide full rudder and allow much easier braking (in combination) without severe leg and foot extension that is required for conventional deflection controls.

TABLE V

AFCAS OPERATING TIME

<u>AFCAS FLIGHT</u>	<u>DESCRIPTION</u>	<u>PILOT</u>	<u>TIME, HRS</u>		<u>REMARKS</u>
			<u>GROUND</u>	<u>FLIGHT</u>	
	Ground Run	--	.7	--	Simulated Flight
1	T-2C Flt #623	Wenzell	.2	1.3	AFCAS Performance Checks
2	T-2C Flt #624	Wenzell	.1	1.2	AFCAS Performance Checks
3	T-2C Flt #625	Wenzell	.1	1.3	System Endurance Evaluation
4	T-2C Flt #626	Wenzell	.1	1.3	System Endurance Evaluation
5	T-2C Flt #627	Wenzell	.1	1.5	System Endurance Evaluation
6	T-2C Flt #628	Cockburn	.1	1.9	System Endurance Evaluation
7	T-2C Flt #629	Wenzell	<u>.1</u>	<u>1.7</u>	System Endurance Evaluation
	TOTAL		1.5	10.2	

TM data covering various tests and maneuvers are presented on pages 86 through 93. Rudder kicks, shown on page 86, demonstrated that damping was dead beat, and rudder re-centering was rapid and accurate. The ability of air loading on the rudder to drive the actuator to the null position with AFCAS "off" is shown on page 88. Decreasing air loads and actuator rod seal friction prevented the rudder from reaching 0°; the 3° (avg) achieved was considered satisfactory. Sideslip and landing data are given on pages 90 and 92.

Photo recorder data taken during Flight #4 is presented on page 94.. Fluid temperatures were similar for all flights. Pump inlet fluid temperature ranged from +125 to 135°F (52 to 57°C); case drain temperatures were +190 to +200°F (88 to 93°C). Fluid temperature decrease through the heat exchanger averaged 21°F/12°C. A one hour cold soak at 25,000 to 30,000 feet (7.6 to 9.1 km) during Flight #5 produced a compartment air temperature of +13°F/-11°C minimum; outside air temperature averaged -5°F/-21°C. The cold soak did not affect AFCAS operating characteristics. All temperatures were considered to be nominal during the seven AFCAS flights.

Case drain flow from the 8000 psi pump ranged from .80 to .85 gpm (3.0 to 3.2 L/m) throughout all tests (hangar, ground demonstration, and flight). Flow measured during the "simulated flights" in the laboratory averaged 0.56 gpm (2.1 L/m). Schedule limitations prevented resolution of the difference. Flow instrumentation used in the laboratory was a Fisher and Porter turbine meter with readout on an electronic counter. Instrumentation used on the aircraft was a Flow Technology paddle-wheel meter with readout on a Flow Technology indicator. Since pump case drain flow was unchanged by ground and flight testing, the disagreement was due to an instrumentation malfunction or a calibration discrepancy. The laboratory flow data were considered to be correct.

System fluid contamination was measured periodically during the flight program, and after 10.2 flight hours was equivalent to a NAVAIR Class 3, page 95. This low level of contamination was evidence of minimal pump wear and excellent fluid lubricity.

No external leakage occurred in the 8000 psi system based on the fact that the reservoir fluid level remained constant throughout flight testing (the level was noted before and after each flight).

ACCUCHART

Scale: 1 in. = 10 Sec

METRIC CONVERSIONS

in.	X	2.540	=	cm
ft	X	.3048	=	m
lb	X	4.448	=	N
psi	X	6895	=	Pa
K	X	.5144	=	in/sec

'OIL HOT' LIGHT

AFCAS FLIGHT #3

Altitude: 18,670 ft (H_p)

Air Speed: 289 Knots

Maneuvers: Rudder kicks
Check re-centering
and damping

Light 'Off'

CORRELATION MARKS

Chart Time Synch

470

520

12° RIGHT

RUDDER POSITION

12° LEFT

2 in. (RETRACT)

ACTUATOR POSITION TRANSDUCER #1

2 in. (EXTEND)

2 in. (RETRACT)

ACTUATOR POSITION TRANSDUCER #2

2 in. (EXTEND)

500 lb (COMPR.)

FORCE TRANSDUCER #1

500 lb (TENSION)

500 lb (COMPR.)

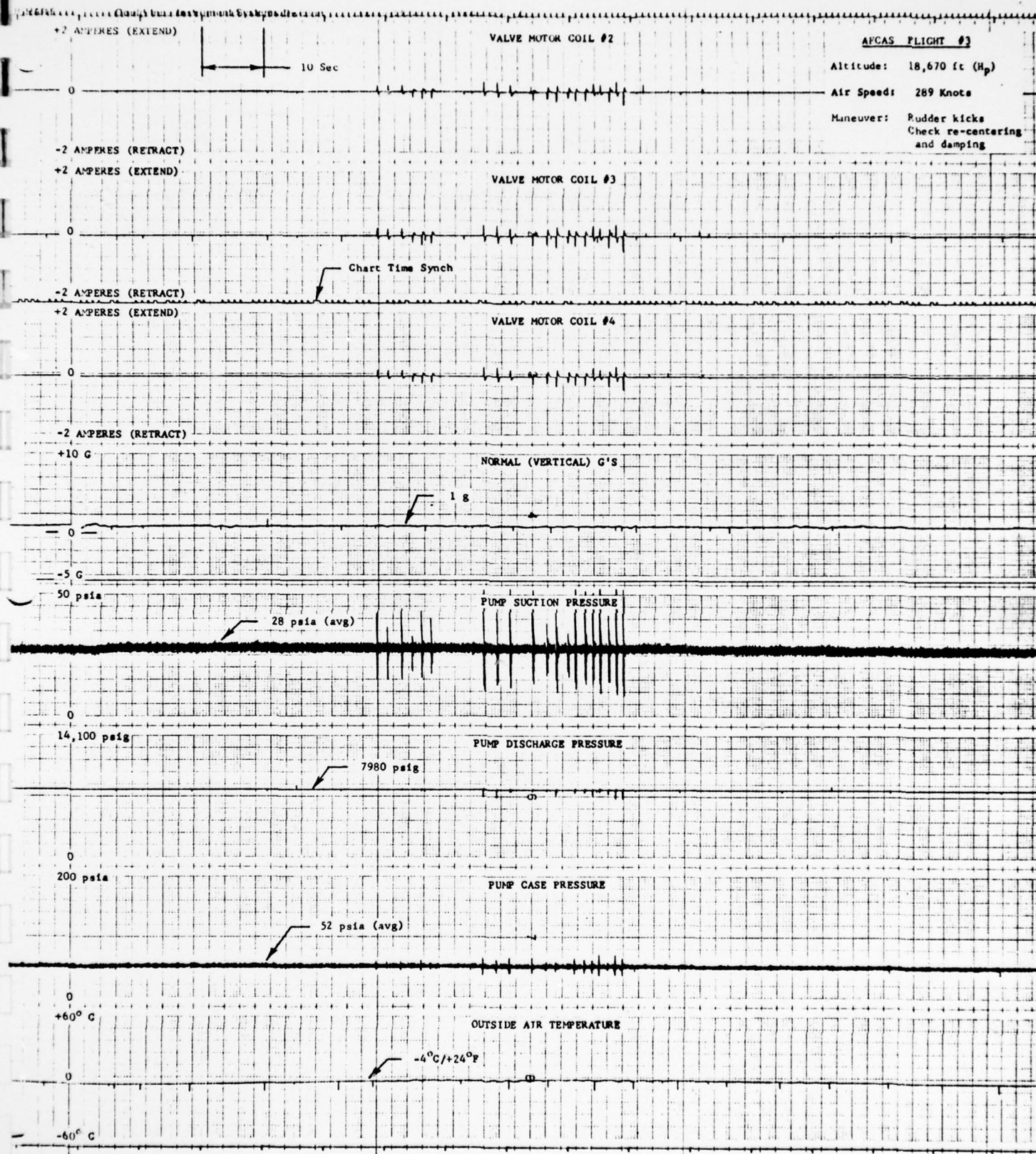
FORCE TRANSDUCER #2

500 lb (TENSION)

+2 AMPERES (EXTEND)

VALVE MOTOR COIL #1

-2 AMPERES (RETRACT)



METRIC CONVERSIONS

in.	X	2.540	=	cm
ft	X	.3048	=	m
lb	X	4.448	=	N
psi	X	6895	=	Pa
K	X	.5144	=	m/sec

'OIL HOT' LIGHT

AFCAS FLIGHT #4

Altitude: 20,210 ft (H_p)

Air Speed: 253 Knots

Maneuver: Hard over rudder

Turn AFCAS 'off'

Check rudder re-centering

Light 'Off'

CORRELATION MARKS

Chart Time Synch

200

300

12° RIGHT

RUDDER POSITION

AFCAS 'off'

AFCAS 'off'

2° Right

4° Left

12° LEFT

2 in. (RETRACT)

ACTUATOR POSITION TRANSDUCER #1

2 in. (EXTEND)

2 in. (RETRACT)

ACTUATOR POSITION TRANSDUCER #2

TM 'noise'

2 in. (EXTEND)

500 lb (COMPR.)

FORCE TRANSDUCER #1

500 lb (TENSION)

500 lb (COMPR.)

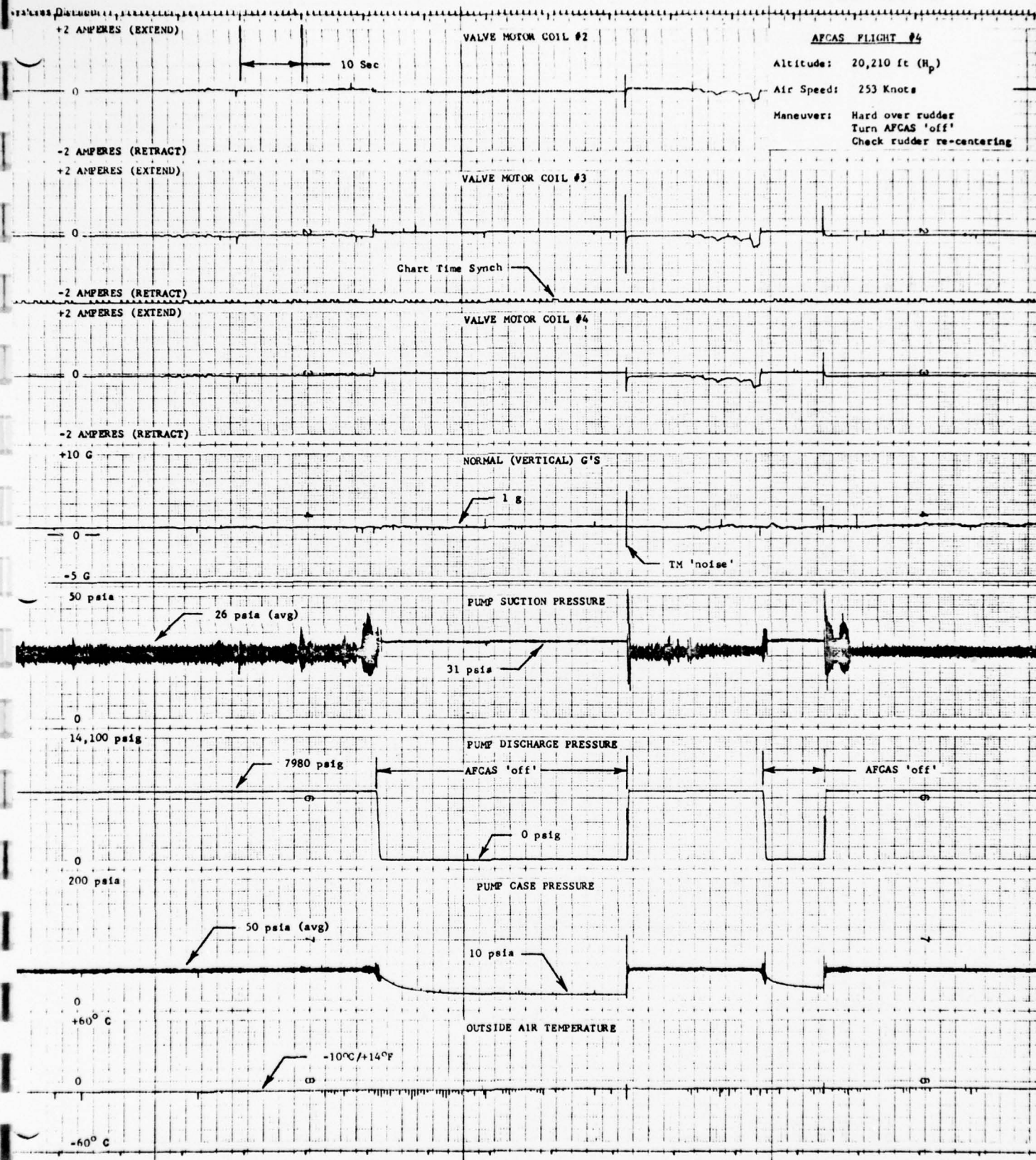
FORCE TRANSDUCER #2

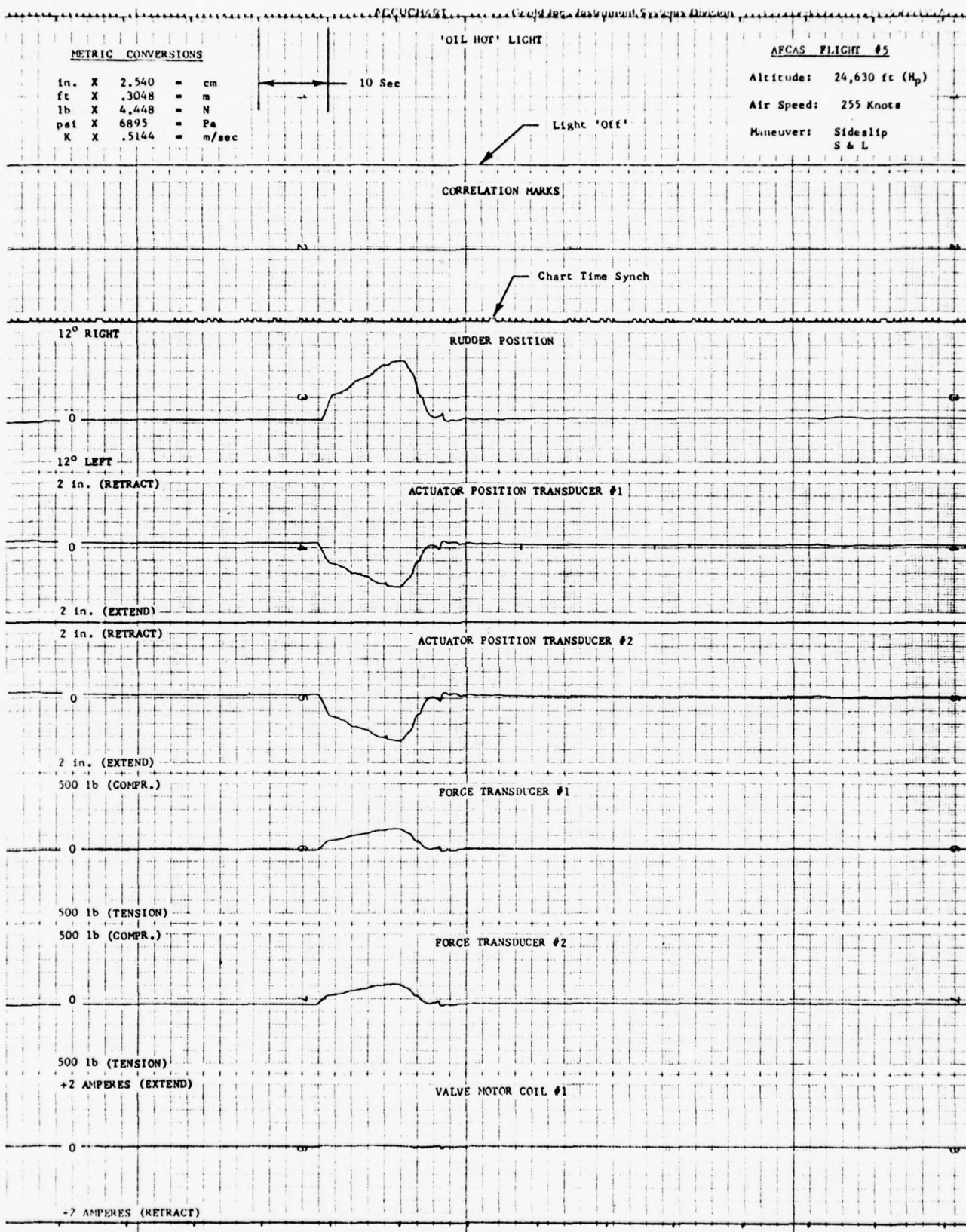
500 lb (TENSION)

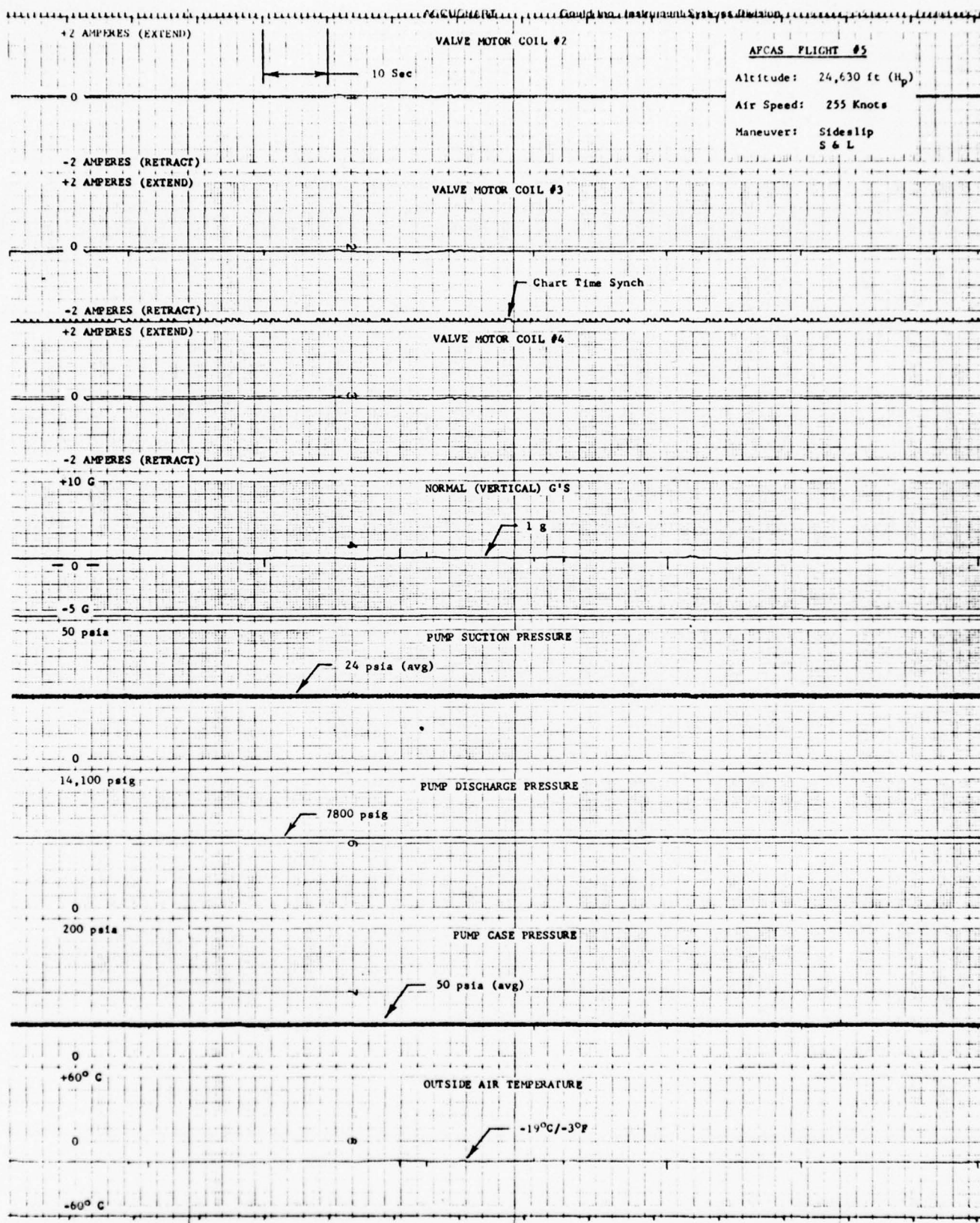
+2 AMPERES (EXTEND)

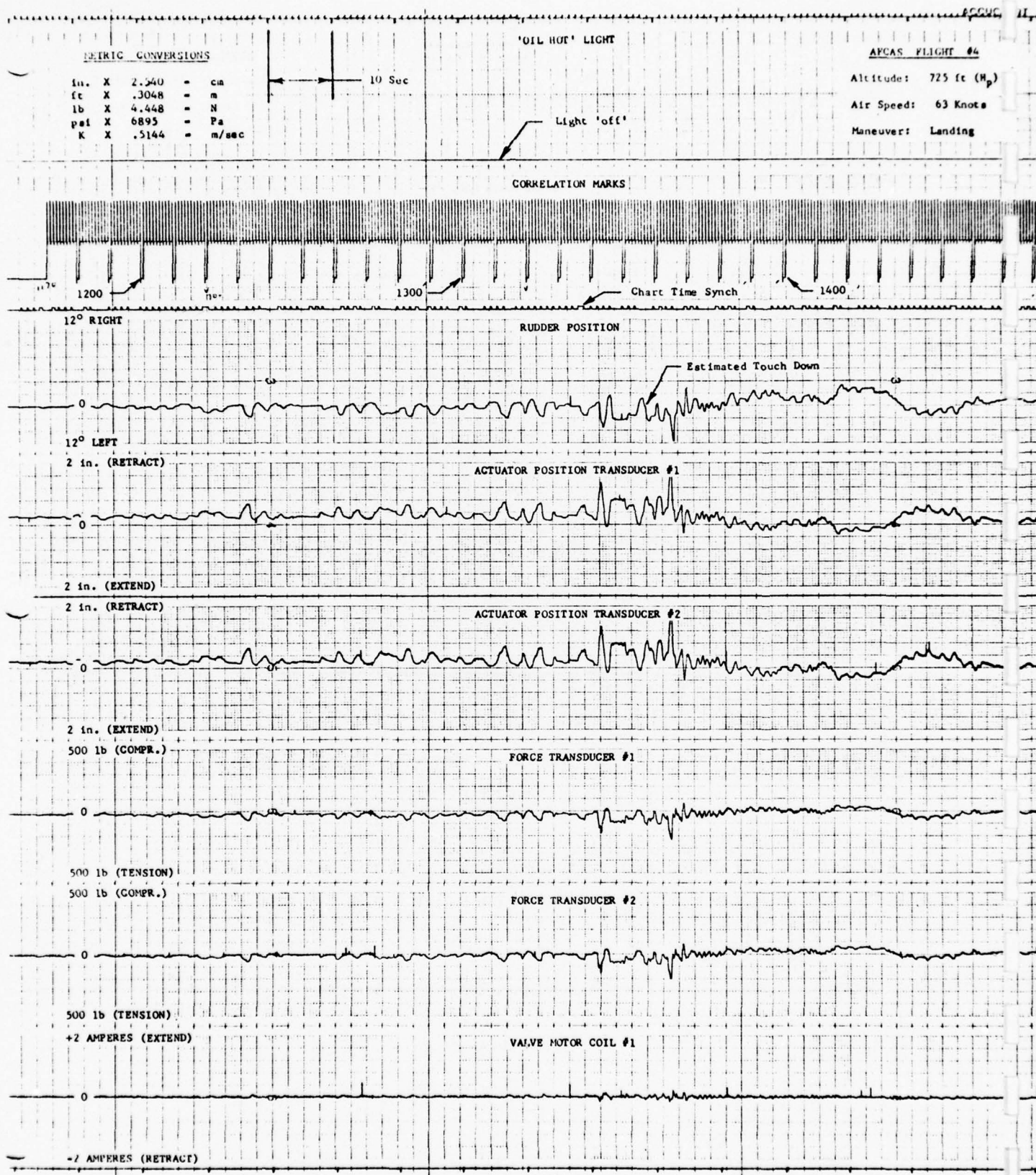
VALVE MOTOR COIL #1

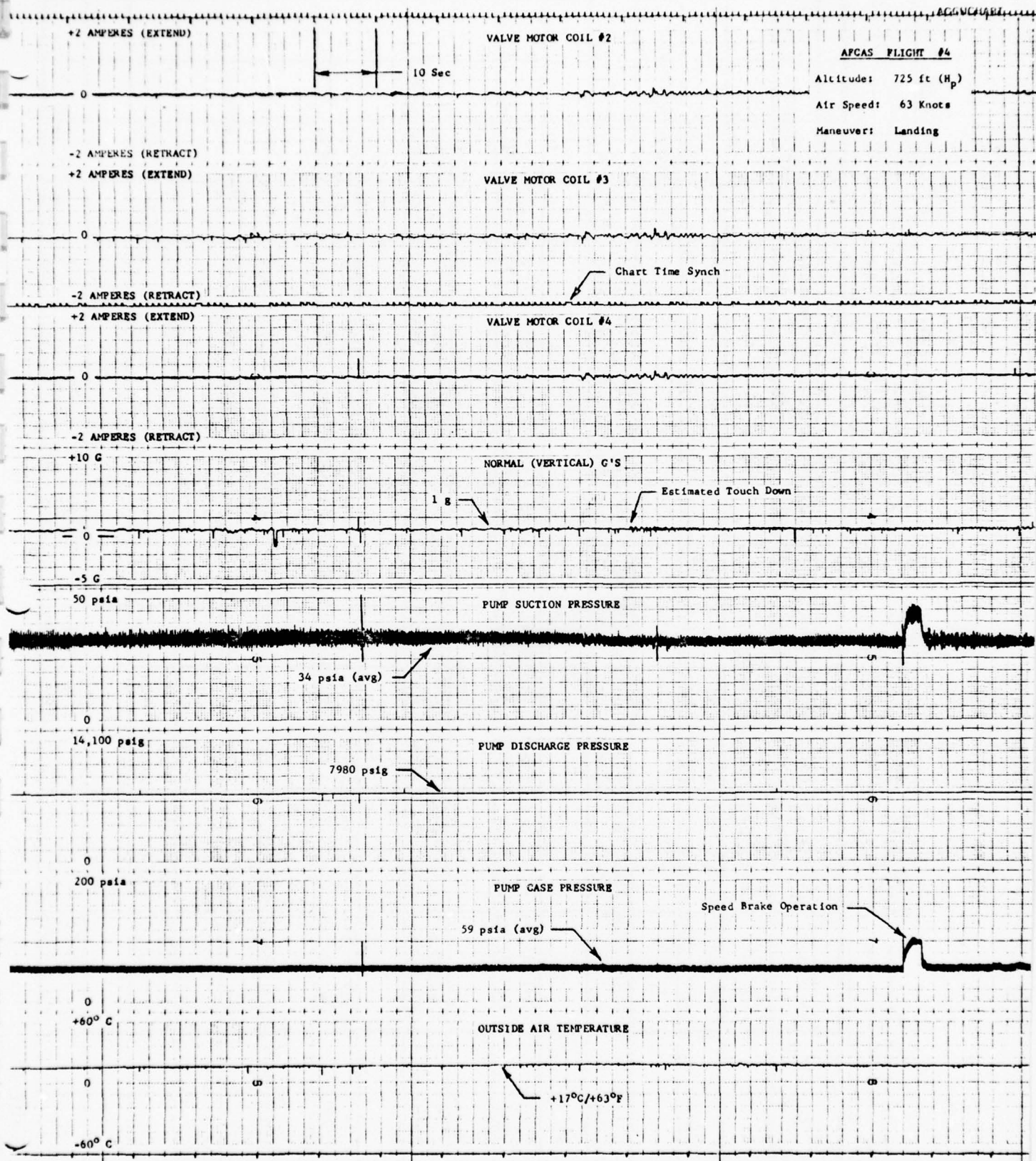
-2 AMPERES (RETRACT)











A/C PILOT WENZELL 7-20 W 1 626 4-28-78 OSC. NO. 1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58 59 60 61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77 78 79 80 81 82 83 84 85 86 87 88 89 90 91 92 93 94 95 96 97 98 99 100

RUN	CORR	TIME	CLOCK	ALT.	A/S	SIDE SLIP	ENG.	PUMP C.D.	PUMP GPM	PUMP GPM	COMP. AIR	PUMP SUCTION	PUMP C.D.	BDU	H. EXCH. INLET	MANEUVER
			2:13	FT	KNOTS	DEG.	RPM	GPM	GPM	GPM	0F	0F	0F	0F	0F	RAW DATA
	169		2:22	20,500	247	7L	6840	85	98	98	+42	131	145	56	140	YAW
			9	20,570	245		6868	84	98		+44	+136	1200	+58	+196	REDUCED DATA
			2:23	20,800	260	6R	6930	92	118	118	43	125	187	53	180	R/L SIDE SLIPS
	253		10	20,870	256		6908	91	117		+44	+131	+193	+55	+186	
			2:42	21,700	272	1L	6820	81	119	119	30	125	192	40	185	"S" TURNS
	429		29	21,780	268		6798	81	118		+30	+131	+198	+41	+191	
			2:50	19,800	274	65L	6860	85	118	118	32	125	192	40	186	360° ROLLS
	507		37	19,870	270		6838	84	117		+32	+131	+198	+41	+192	
			2:51	19,300	275	1R	6870	0	0	0	33	122	184	40	176	AF CAS OFF
	573		38	19,370	271		6848	0	0	0	+33	+128	+190	+41	+182	
			3:15	20,400	226	0	5500	85	112	112	33	126	192	36	187	TRIM ROLLS
	702		62	20,470	225		5495	84	111		+33	+132	+198	+37	+193	
			3:20	10,500	240	0	4850	79	111	111	39	124	190	40	184	IDLE DESCENT
	949		67	10,500	239		4845	79	109		+40	+130	+196	+41	+190	
			3:29	1250	136	0	6540	82	117	117	68	120	181	56	175	LANDING
	1249		76	1280	129		6570	82	116		+70	+126	+187	+58	+181	

DATA SHEET 384 N-1(11-73)

QUALITY ASSURANCE LABORATORIES



TEST REPORT

Data Rec'd in Lab

4-17-78

Articles

Hydraulic Fluid

Source

1 T2C # 152382

Specification

Phil-H. 83282

Test Required

CONTAMINATION

Quantity

See, as noted

Submitted By:

Note

R. Hanning

Dest.

071

Ext. 2847

[illegible]

Disposition and, or Comments:

Signed

Harry J. Bayne

Approved

Date _____

5-3-78

Distribution:

W. Bethel DO90-B7

W. Doan D054-133

U.F. Santry D071-543-736

6.0 DISCUSSION

A direct-drive control-by-wire muscle actuator, powered by a localized 8000 psi hydraulic system, was used to control the flight of a T-2C. Successful operation of the test installation represented a significant milestone in the development of advanced flight controls. No problems whatsoever were encountered; the system functioned exceptionally well and pilot response was favorable. The test results confirmed analyses and laboratory investigations reported in References 1 through 4. The ease with which flight testing was accomplished verified that AFCAS type systems can be designed, fabricated, and maintained without special techniques or state-of-the-art advances.

The AFCAS concept is intended for application to automatic, computer operated flight control systems. The current AFCAS flights did not demonstrate the full performance capabilities of the test hardware since the T-2C did not have computer operated controls. Company funded investigations at the Columbus Aircraft Division have verified the feasibility of controlling AFCAS actuators directly by a digital computer. The following section is a brief discussion of this work.

DIGITAL CONTROL OF AFCAS

A laboratory setup was assembled which interfaced AFCAS direct-drive actuation circuits with a PDP-11 programmable digital processor. The processor was equipped with an I/O (input/output) module which converted digital data into analog commands, and which could also be used as a signal buffer for the transmission of digital commands directly to the input stages of the actuator control circuits.

The surface actuation control loop could be closed external to the computer or internal at the computer, Figure 37. With the loop closed externally, computer output corresponded to a surface position command. With the loop closed internally, computer output represented the surface position error signal. The error signal could be a dc analog voltage or, by changing the software, a pulse modulated signal representative of other waveforms.

The PDP-11 processor was used for closed loop control of the direct-drive dual tandem actuator built early in the AFCAS program, Reference 2. The processor was programmed to generate the error signal in four formats: (1) dc analog, (2) pulse-width modulation, (3) bang-bang, and (4) time dwell modulation. Frequency response was determined for all four modes, and was approximately equivalent to that obtained with all-analog control, i.e., without the computer in the loop.

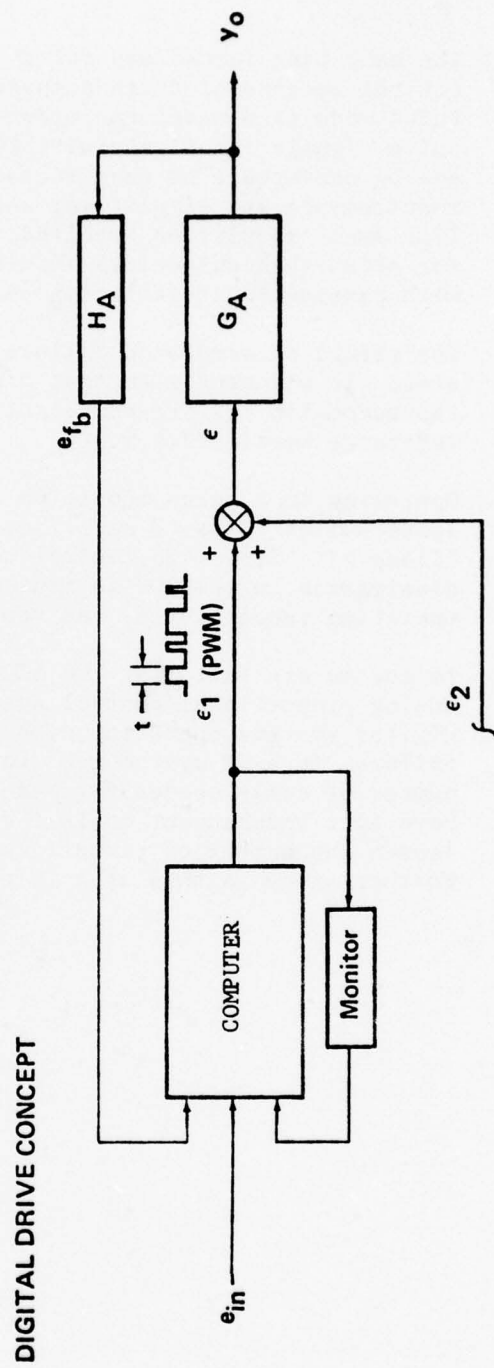
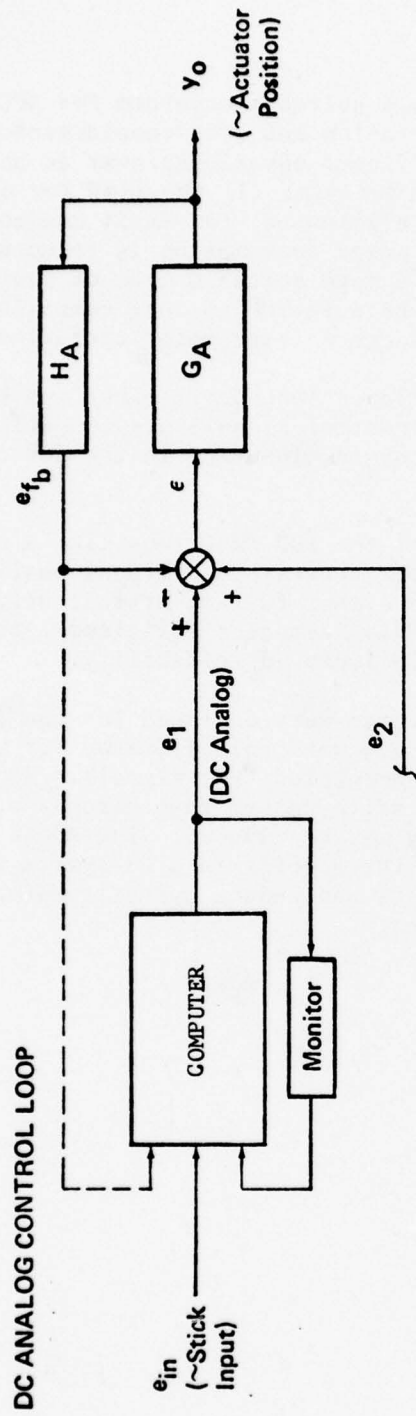


FIGURE 37 COMPUTER CONTROL CONFIGURATIONS

The bang-bang format was ruled out as a suitable waveform for AFCAS control because of rough actuator operation and wear considerations. Pulse modulated waveforms offer significant advantages over dc analog drive signals for fly-by-wire systems because: (1) the need for digital-to-analog converters at each surface is eliminated; (2) fault monitoring requirements are simplified; and (3) power consumption is reduced. Time dwell modulation appeared to be a more suitable mode of control for AFCAS than pulsewidth modulation because TDM is more compatible with passive fault-isolation and produces quieter motor operation.

The effect of simulated failures on closed loop performance was evaluated. It was confirmed that digital control signals are compatible with the automatic failure-compensation features inherent in the EDU concept, reference Section 3.4.3.

Operating in a pulse modulated system, the EDU functions like a high speed switch (Class D amplifier) rather than a proportional amplifier (Class A). Since the control unit is either full on or full off, power dissipation in the EDU is minimal. This increases efficiency, reduces operating temperatures, and results in improved reliability.

Torque motors built for the AFCAS program were designed for use in analog proportional control systems, and were not optimized for use in digital systems operating with pulse modulated (PM) signals. Since failures in a PM system tend to be passive rather than hard-over, the number of coils needed for redundancy may be reduced. The AFCAS motors have four independent coils. Use of three coils in a PM system would lessen the number of circuit components and reduce system complexity. Further study in this area is required.

7.0 RECOMMENDATIONS

Flight verification of the AFCAS concept was completed in Phase V using a system with a direct-drive actuator, localized hydraulic power supply, electronic drive unit, and force transducer. The test installation was an analog control-by-wire system; the AFCAS concept is intended for application to digital control-by-wire systems. The Columbus Aircraft Division has confirmed by laboratory testing that AFCAS components are compatible with digital control-by-wire components. Therefore, it is recommended that the AFCAS test system currently installed on the T-2C be modified by the addition of a micro-processor and that additional flight testing be conducted. This will provide the Navy with an economical approach to demonstrate, in flight, advantages of the direct-drive features of AFCAS with computer control. A further benefit is the availability of a flight test vehicle for future digital control-by-wire developmental effort.

The following tasks are recommended as logical next steps in the AFCAS development cycle:

- | | |
|----------|---|
| PHASE VI | <u>FLIGHT DEMONSTRATION OF DIGITAL CONTROL OF AFCAS
IN THE T-2C AIRPLANE</u> |
| Task 1 | Procure an off-the-shelf digital micro-processor |
| Task 2 | Program the micro-processor |
| Task 3 | Design the interface electronics and aircraft modifications required to accommodate the additional components |
| Task 4 | Install components and make wiring changes per drawings completed in Task 3 |
| Task 5 | Conduct flight testing to demonstrate the computer/actuator interface method |

A second recommendation is concerned with the direct-drive control module (force motor and spool/sleeve valve). The motors and valves procured for prior AFCAS projects were designed for concept verification only, and were not intended to be production configurations. LHS and AFCAS technology have progressed sufficiently that effort can now be directed toward optimized designs which can be integrated into future production applications. The following tasks are therefore recommended: (1) establish force motor design criteria required to achieve a reduced envelope and the reliability necessary for digital control systems; (2) develop classes of control modules to cover the span of actuator classes established in Reference 1; (3) prepare procurement specifications and solicit potential suppliers for the control modules.

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REFERENCE NO.

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- 2 NR73H-107, Control-by-Wire Actuator Model Development for AFCAS, Rockwell International Corporation, Columbus Aircraft Division, Contract N62269-73-C-0405, January 1974, Unclassified. AD 772 588
- 3 NR75H-1, Control-by-Wire Modular Actuator Tests (AFCAS), Rockwell International Corporation, Columbus Aircraft Division, Contract N62269-73-C-0405, January 1975, Unclassified. AD A-006 371
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- 5 D. Deamer, S. Brigham, Theoretical Study of Very High Pressure Fluid Power Systems, NA66H-822, North American Aviation, Inc., Columbus Division, Contract N0w 65-0567-d, 15 October 1966, Unclassified. AD 803 870
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- 13 J.N. Demarchi and R.K. Haning, Flight Test of an 8000 psi Lightweight Hydraulic System, NR77H-21, Columbus Aircraft Division, Rockwell International Corporation, Contract N62269-76-C-0254, April 1977, Unclassified. AD-A039 717/4GA
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LIST OF ABBREVIATIONS

AC	alternating current
AFCAS	Advanced Flight Control Actuation System
A/S	air speed
alt.	altitude
amp	ampere
approx.	approximately
avg	average
BTU/min	British Thermal Units per minute
°C	degrees Celsius
CAD	Columbus Aircraft Division
cc/min	cubic centimeters per minute
CIPR	cubic inches per revolution
c	centi (10^{-2})
cm ³	cubic centimeters
CORR	correlator
CRES	corrosion resistant
dB	decibel
DC	direct current
deg	degree
EDU	electronic drive unit
°F	degrees Fahrenheit
FRP	flight reference plane
F.S.	fuselage station

ft	feet
ft/sec	feet per second
G	giga (10^9)
gpm	gallons per minute
hp	horsepower
H _p	pressure altitude (29.91 in. Hg = Sea Level)
Hr	hour
Hz	Hertz (cycles per second)
I.D.	inside diameter
in.	inch
in ²	square inches
I/O	input/output
J	joule (metric unit of work)
k	kilo (10^3)
K	knots
kg	kilogram
km	kilometer
KOAS	Knots Observed Airspeed (uncorrected)
kW	kilowatt
L	liter
LED	light emitting diode
L/m	liters per minute
LH, L/H	left hand
LHS	Lightweight Hydraulic System
L/R	left and right

LVDT	linear variable differential transformer
m	meter, also milli (10^{-3}), also minute
M	mega (10^6)
max.	maximum
mm	millimeter
M/N	model number
min	minute (time)
MMH/FH	maintenance man-hours per flight hour
MPa	megapascals
MRT	military rated thrust
MN	mach number, also meganewtons
m/s	meters per second
N	newton (metric unit of force)
NADC	Naval Air Development Center
No.	number
OAT	outside air temperature
O.D.	outside diameter
P-P	peak-to-peak
ΔP	differential pressure
Pa	pascal (metric unit of pressure)
PLF	power for level flight
psi	pounds per square inch
psia	pounds per square inch absolute pressure
psig	pounds per square inch gauge pressure
PM	pulse modulation

P/N	part number
PWM	pulse width modulation
RH, R/H	right hand
rpm	revolutions per minute
s	second (time), also LaPlace transform operator
sec	second (time)
S&L	straight and level
TDM	time dwell modulation
TM	telemetry
T/O	take-off
T	time constant
V	volt
V_h	maximum velocity for level flight
W	watt

SUMMARY OF METRIC CONVERSIONS

Area	in ²	x	6.452	=	cm ²
	ft ²	x	.0929	=	m ²
Fluid Flow	gal/min	x	3785	=	cc/min
	gal/min	x	3.785	=	L/min
	in ³ /sec	x	16.39	=	cc/sec
Force	lb	x	4.448	=	N
Heat Flow	BTU/min	x	17.58	=	W
Length	in	x	2.540	=	cm
	ft	x	.3048	=	m
Mass	lb	x	.4536	=	kg
Power	hp	x	.7457	=	kW
	ft-lb/sec	x	1.356	=	W (= J/sec)
Pressure, Stress	psi	x	6895	=	Pa (= N/m ²)
	psi	x	.06895	=	bar
Spring Rate	lb/in	x	175.1	=	N/m
Torque	lb-in	x	.1130	=	N-m
	lb-ft	x	1.356	=	N-m
Velocity, Speed	in/sec	x	2.540	=	cm/sec
	ft/sec	x	.3048	=	m/sec
	knots	x	.5144	=	m/sec
Volume	in ³	x	16.39	=	cm ³ (= cc)
	gal	x	3.785	=	L
	L	x	1000	=	cm ³
	m ³	x	1000	=	L
Work, heat	ft-lb	x	1.356	=	J
	BTU	x	1.055	=	kJ

APPENDIX A

TEST PROCEDURES

This section details procedures used for ground checking and flight testing the AFCAS directional control installation on the T-2 aircraft.

<u>SECTION</u>		<u>PAGE</u>
A.1	SYSTEMS CHECKOUT TESTS	108
A.2	GROUND DEMONSTRATION TESTS	115
A.3	PILOT INFORMATION	117
A.4	FLIGHT TESTS	120

A.1 SYSTEMS CHECKOUT TESTS

1.0 FILL AND BLEED 8000 PSI SYSTEM

A 3000 psi ground cart containing MIL-H-83282 fluid is required.

- Temporarily connect rudder actuator pressure and return lines together. Adaptor fitting provided by Department 071.
- Install bleeder valve furnished by Department 071 in system return line in RH speed brake well. Plug open line.
- Connect plumbing supplied by Department 071 to pump suction line bleed port, pump pressure hose, and case drain line.
- Leave pump case drain port open. Cap pump pressure port.
- Attach ground cart fill line to aircraft. Fill at approximately 1.0 gpm and 85 psi (max.)
- Bleed air from heat exchanger bleed port located at the upper left aft corner of heat exchanger.
- Bleed air at valve in speed brake well.
- Fill reservoir to full mark.
- Dump reservoir pressure.
- Remove temporary plumbing in fuselage compartment and re-install A/C lines. Take care to avoid losing fluid and introducing air.

2.0 LEAK CHECK 8000 PSI SYSTEM

- Connect portable hydraulic power supply (furnished by Department 071) to 8000 psi pressure hose at the motor pump unit. Remove bleed valve and connect power supply return line to the system return line in the RH speed brake well. Lines and fittings furnished by Department 071.
- Apply 1000 psi pressure. Check for leaks. Increase pressure to 4000 psi, then to 8000 psi. Look for leaks.
- Apply pressure sufficient to crack relief valve (8600 to 9000 psi). Do not relieve for more than 15 seconds to avoid overfilling A/C reservoir. Look for leaks.
- Remove portable 8000 psi power supply, re-install bleeder valve in RH speed brake well, and reconnect 8000 psi pump hose and case drain line.

3.0 BLEED 3000 PSI SYSTEMS AND OPERATIONAL CHECK

- Connect ground cart pressure pressure and suction lines to aircraft. Make sure pressure line is full of fluid before making connection.
- Connect elevator actuator pressure and return hoses together.
- Apply 100 psi to pressure port. Bleed air at bleeder valve.
- Dump pressure. Reconnect elevator actuator hoses. Remove bleeder valve and reconnect return line in RH speed brake well.
- Apply 3000 psi pressure. Look for leaks.
- Cycle, in order, the following:

	<u>Complete Cycles</u>
1. Speed brakes	10
2. Ailerons	10
3. Arresting hook	5
4. Elevator	10

- Assure proper operation of above subsystems.

4.0 ELECTRICAL WIRING VERIFICATION

- Continuity check all wiring per AFCAS drawing No. 8691-546606.
- Fit check and verify all mating connectors used on rudder actuator, position LVDT's, force transducers, and electronic drive unit (EDU).
- With the EDU, position LVDT's, force transducers, and actuator disconnected, verify that the following aircraft harness pins, and no others, have continuity to aircraft chassis ground:

<u>EDU A/C Disconnect</u>	<u>Pin No.</u>
J4	D
J4	L
J4	P

- Disconnect power plug from heat exchanger blower. Disconnect wire No. C240AOT from terminal A1 of the A4D pump motor relay KA. Protect relay terminals to prevent shorting when electrical power is applied to aircraft busses.

- Connect 28 VDC external ground power cart to aircraft. DO NOT TURN ON.
- Remove relays K108 and K109, #1 and #2 generator bus control relays #3, from relay sockets. Add jumper wire between pins 3 and 5 of both relay sockets, XK 108 and XK 109.
- With the EDU, position LVDT's, force transducers, and actuator disconnected, apply 28 VDC power to aircraft and turn #1 inverter on. Turn hydraulic power switch on.
- Verify there is 115 VAC from J4 on the EDU A/C disconnect, pins N and M, to aircraft ground. Verify that no voltage appears on the remaining pins of this connector and on the LVDT and force transducer A/C disconnects.
- Verify operation of hydraulic pump motor relay KA by testing for 28 VDC present at terminal A1. Turn off #1 inverter and 28 VDC external power.

5.0 SYSTEMS CHECKOUT

5.1 Electrical System

- Connect J4 on the EDU to the A/C harness. Disconnect position LVDT's, force transducers, and actuator motor plugs.
- Turn on 28 VDC ground power supply and #1 inverter.
- Turn rudder hydraulic power switch to "on". Motor/pump unit should not run, but power will be applied to EDU.
- Verify the following voltages:

<u>Disconnect</u>	<u>Pins</u>		<u>Required Voltage (Tolerance: +0.2 VDC)</u>
	<u>High</u>	<u>Low</u>	
Position LVDT #1	E	D	+15 VDC
Position LVDT #1	F	D	-15 VDC
Position LVDT #2	E	D	+15 VDC
Position LVDT #2	F	D	-15 VDC
Force Transducer #1	E	D	+15 VDC
Force Transducer #1	F	D	-15 VDC
Force Transducer #2	E	D	+15 VDC
Force Transducer #2	F	D	-15 VDC

- Turn rudder hydraulic power switch "off". Connect the position LVDT's, force transducers, and actuator motor to aircraft harness.
- Connect J3 on the EDU to the AFCAS test box provided by Department 071.
- Turn rudder hydraulic power switch "on".
- Manually position rudder actuator, using the rudder surface, until the voltages at the actuator LVDT output E for both LVDT's is within $0 \pm .100$ VDC. The rudder surface position shall be $0 \pm 1/4^\circ$. Adjust bell-crank-to-surface push rod as required to obtain $0 \pm 1/4^\circ$ surface position.
- Measure and record the voltages shown on Table I which are available at terminals on the AFCAS test box.
- Apply sufficient pedal force to produce between 1.0 and 2.0 VDC on the force transducer outputs. Observe the corresponding LED illumination on the AFCAS test box during right and left commands.
- Turn rudder hydraulic power switch "off", then turn #1 inverter "off".
- Reconnect wire No. C240AOT to terminal A1 of the hydraulic pump motor relay KA. Insure terminals of relay are protected against shorting. Reconnect plug to power heat exchanger blower. Turn off 28 VDC ground cart.

5.2 Hydraulic System

- Apply 25 psig air pressure to reservoir. Use nitrogen bottle with pressure regulators.
- CAUTION: Operation of the 8000 psi motor/pump unit without engines running requires external reservoir pressurization. Apply air pressure through a capped Tee located near the reservoir pressure regulator.
- Attach temperature potentiometer to thermocouple in motor/pump suction line (furnished by Department 071).
 - Disconnect power plug from EDU.
 - Insure jumper wires between pins 3 and 5 of relays K 108 and K 109 are in place and secure.
 - Apply 28 VDC to aircraft. Turn on #1 inverter. Heat exchanger blower should be running.

TABLE A-I

NOTE: AFCAS POWER ON
HYDRAULIC POWER OFF
RUDDER AT 0° (NULL POSITION)

PEDAL COMMAND	ACTUATOR LVDT OUTPUT E, VDC		FORCE TRANSDUCER OUTPUT E, VDC		VALVE DRIVER OUTPUT E, VDC						
	REQUIRED	#1	#2	REQUIRED	#1	#2	REQUIRED	#1	#2	#3	#4
NO PEDAL	0 ± .100			0 ± .125			0 ± .500				
RIGHT PEDAL	+0.2 MAX.			+1 TO +2			+9 TO +10				
LEFT PEDAL	-0.2 MAX.			-1 TO -2			-9 TO -10				

- Turn rudder hydraulic power switch "on" in cockpit. Observe that pressure is 8000 psi on cockpit gage. Look for leaks.
- Run at 8000 psi until the suction line fluid temperature reaches 210°F or the fluid temperature stabilizes (.5°F rise/minute). Do not run longer than 20 minutes. Record fluid temperature every minute to estimate stabilization temperature and to establish permissible hangar operating time for the 8000 psi system. During the above test, the "oil hot" light should illuminate when the fluid temperature reaches $205 \pm 5^\circ\text{F}$. Reset suction line thermal switch as necessary. If the fluid temperature did not reach 200°F, block the heat exchanger air inlet (right hand side of aircraft) and observe fluid temperature and oil hot light. Do not exceed 210°F fluid temperature. Reset thermal switch as required.
- Turn off rudder hydraulic power switch, #1 inverter and 28 VDC ground power supply.
- Manually push rudder full right and full left to verify trailing capability with hydraulic power off. Measure breakout force required to make rudder trail. Apply load to trailing edge of rudder (not trim tab) opposite push-rod attach point. Protect rudder skin. Record loads.

5.3 AFCAS Installation

- Connect power plug to EDU.
- Turn on 28 VDC ground power supply, #1 inverter, and rudder hydraulic power switch.
- Operate rudder pedals. Assure that rudder operation is satisfactory. Rapidly oscillate rudder a sufficient number of cycles (at least 25) to remove any trapped air within the rudder actuator. Note sensitivity and dead band.
- Apply full right and left pedals. Measure and record maximum rudder deflection. Maximum right and left rudder should be $12 \pm 1/2$ degree. Determine that rudder returns to $0 \pm 3/4$ degree with no pedal force.
- Measure and record pedal force and rudder deflection to establish force vs. deflection curve.
- Measure and record rudder pedal travel for maximum rudder deflection.
- Measure and record the voltages shown below with no rudder pedal command.

<u>Description</u>	<u>Required Voltage</u>
Actuator LVDT Output E	0 ± 0.100 VDC
Force Transducer Output E	0 ± 0.125 VDC
Valve Driver Output E	0 ± 0.500 VDC

- Hook up instrumentation provided by Department 071 for measuring pressure transients.
- Operate hydraulic system at 8000 psi and check for detrimental pressure oscillations. Operate rudder and measure pressure surges.
- Hook up oscilloscope provided by Department 071 to 28 VDC terminals on aircraft circuit breaker panel.
- Measure "noise" on 28 VDC bus with motor/pump unit running.

Secure Procedure

- Turn rudder hydraulic power switch "off", then turn #1 inverter "off" and remove external electrical power from aircraft.
- Remove jumper wires from relay sockets XK 108 and XK 109.
- Reinstall relays K 108 and K 109.
- Dump reservoir pressure.

Fluid Contamination Check

- Take fluid sample from reservoir using existing procedures for contamination check. Visually examine case drain and return filter bowls for debris.

A.2 GROUND DEMONSTRATION TESTS

1.0 PREPARATIONS

- Open RH hydraulic bay access door. Open fuselage bay access door. Put steps up to door. Remove vertical stabilizer side panel.
- Attach temperature potentiometer to thermocouple in motor/pump suction line (provided by Department 071).
- Hook up oscilloscope (provided by Department 071) to check "noise" on 28 VDC bus.

2.0 SIMULATED FLIGHT TEST

- The test shown on Table A-II simulates a one hour flight from take-off to landing. Both engines shall be run. Instrumentation will be operated the same as during an actual flight. A "data burst" as used on Table A-II is defined as turning the photo recorder "on" for approximately 15 sec., then "off". The TM oscillograph shall run continuously.
- Monitor pump suction line fluid temperature throughout this test using portable potentiometer. If temperature reaches +200°F, turn hydraulic power switch "off" and shut down.
- Measure "noise" on 28 VDC bus at engine idle with hydraulic power switch "on".
- Take fluid sample from reservoir for contamination check within one hour following ground demonstration test.
- Process film and reduce instrumentation data.

3.0 PREPARATIONS FOR FLIGHT TEST

- Remove pump case drain filter bowl. Deliver bowl, element and fluid to Department 071 for patch test.
- Service reservoir in accordance with Specification HA0201-259, paragraph 4.2.7.
- Mark fluid level position on reservoir sight glass.
- Check instrumentation zeros and R Cals.

TABLE A-II

GROUND DEMONSTRATION TEST

<u>SIMULATION</u>	<u>ENGINE SPEED, %</u>	<u>ELAPSED TIME, MIN.</u>	<u>PHOTO RECORDER</u>	<u>RUDDER, AILERON & ELE- VATOR OPERATION</u>
Engine Start	0 to 48%	0 → 1/4	On	No
Hyd. Power Switch "On"			On	
System Check- out & Taxi Out	48%	1 9	DB DB	Periodic
Take-Off	100%	10 14	DB DB	Periodic
Cruise	90%	15 25	DB	Periodic
	100%	26 28	DB	
	90%	29 34	DB	
	100%	35 37	DB	
	90%	38 43	DB	
	80%	44 50	DB	
Landing & Taxi-In	48%	51 59	DB	Periodic (+ Speed Brakes)
Hyd. Power Switch "Off"		60	On	
Engine Shutdown	48% to 0	61	On	No

A.3 PILOT INFORMATION

Changes made to the aircraft to incorporate the Advanced Flight Control Actuation System are described herein. The test installation is a fully powered control-by-wire directional system containing:

- Electric motor driven pump
- Rudder actuator
- Electronic drive unit
- Force transducer

The modified hydraulic system will operate at two pressure levels: 3000 psi and 8000 psi. An 8000 psi motor/pump unit has been added to power the rudder system (only). Both engines drive the normal 3000 psi pumps which power the lateral, horizontal, speed brake, and landing gear systems in the usual manner. The 3000 psi and 8000 psi systems share the T-2C reservoir and have common return lines. The test installation is shown schematically on Figure A-1. The modified system will operate functionally the same as the basic T-2C aircraft except rudder operation will not be manual but hydraulically powered. The variable stability system has been deactivated and the speed brakes may be operated using the normal speed brake switch.

The original cable system between the rudder pedals and rudder has been modified to incorporate the control-by-wire system. The rudder pedal cables operate a sector which is prevented from rotating by a force transducer. The rudder pedals will have very little displacement. Force on the pedals is converted to a proportional electrical signal from the force transducer. This command signal is transmitted to the electronic drive unit which conditions the signal and powers a torque motor on the rudder actuator. The torque motor in turn drives a conventional control valve on the actuator. The electrical system contains redundant circuitry which provides high immunity to component failures.

The rudder actuator has a pressure operated bypass valve which permits the rudder to trail if hydraulic power is lost. In the event of a "hard-over" electronic-type failure, the pilot can cause the rudder to trail by turning the 8000 psi rudder hydraulic power system switch to "off".

The rudder trim system is unchanged. Trim response will be different, however, due to the change from a manual to a fully powered rudder. The yaw damper system has been disconnected.

Maximum rudder displacement is reduced from ± 25 degrees to ± 12 degrees. This reduction will permit the pilot to land safely with a "hard-over" rudder, opposite engine out, and a three knot crosswind. The relationship between rudder displacement and pedal force is approximately 8 lb/deg. of rudder movement (93 lb. for full travel).

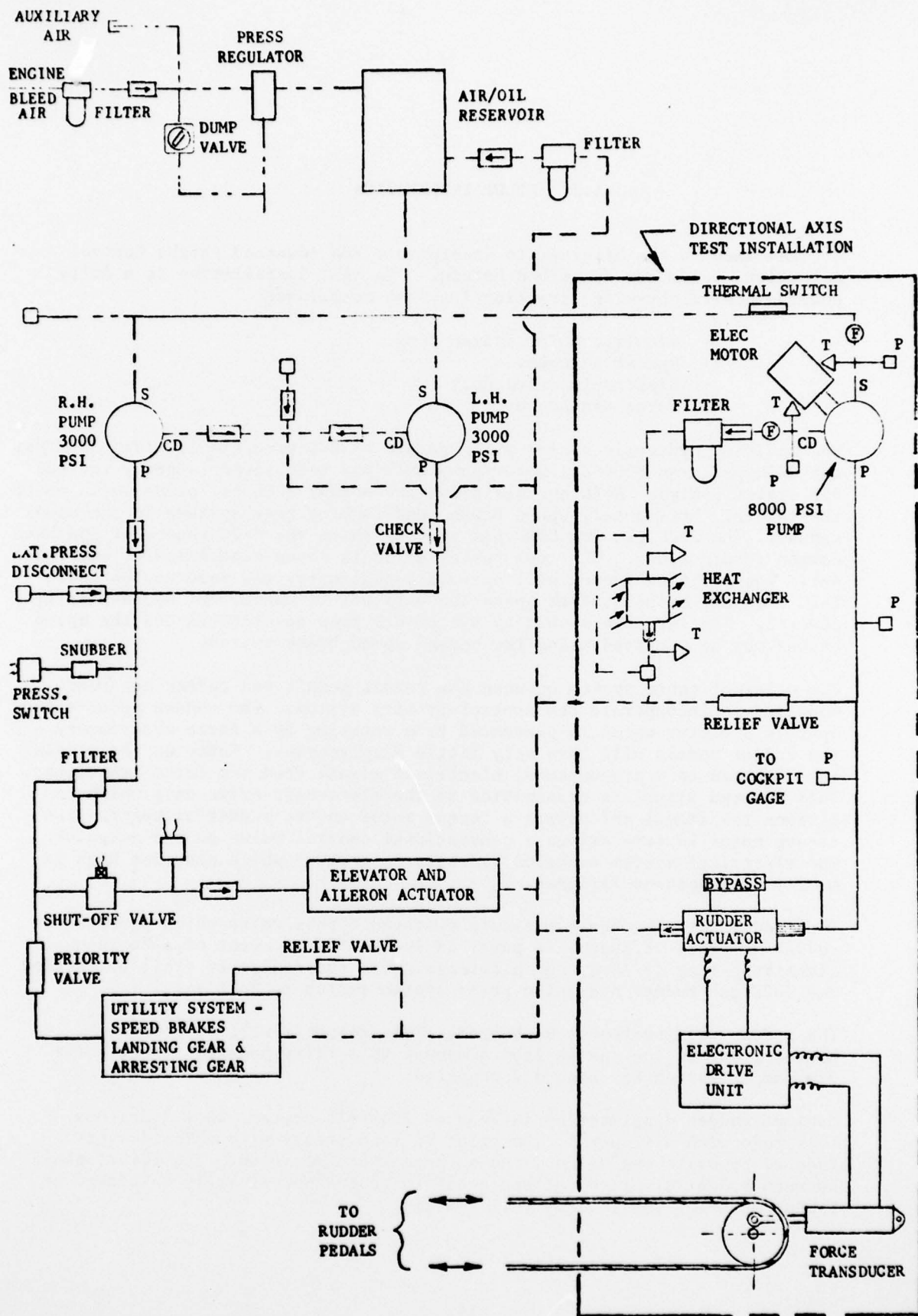


FIGURE A-1 SCHEMATIC DIAGRAM OF MODIFIED HYDRAULIC SYSTEM

Because of the additional load imposed on the 28 VDC generators, the motor/pump unit can be operated only when both engines are running. For this reason, the unit should be turned "on" after both engines have been started and turned "off" before engine shut-down.

Modifications in the cockpit area are as follows:

1. 8000 psi hydraulic pressure on the rudder actuator and electric power to the EDU can be shut off by means of a rudder hydraulic power switch located on the pilot's auxiliary instrumentation control panel (shroud).

NOTE: For total flight control boost shut-off, the above hydraulic power switch and the normal system boost shut-off switch must be moved to "off". The rudder will trail in this situation and cannot be operated.

2. Output from the 8000 psi pump is displayed on the upper right hand side of the pilot's instrument panel.

NOTE: The pressure displayed is in the pump discharge line and will fall to zero when the hydraulic power switch is at "off".

3. An oil hot light is provided on the pilot's auxiliary instrument panel (shroud). This light indicates excessive hydraulic system fluid temperature. Actuation of the light is an indication of system malfunction.

Contingency recommendations are:

1. If the left and right yaw responses become significantly different for equal inputs, a malfunction in the system is indicated. Terminate test. Turn the rudder hydraulic power switch "off". Make return flight.
2. If the rudder should become "hard-over", terminate test. Turn rudder hydraulic power switch "off". Make return flight.
3. If the oil hot light comes on, terminate test. Turn rudder hydraulic power switch "off". Reduce power setting and alternately cycle the speed brakes and landing gear during return flight to lower bulk fluid temperature. Stop cycling when fluid temperatures become normal.
4. If the 8000 psi system pressure drops below 6000 psi, terminate test. Turn the rudder hydraulic power switch "off". Make return flight.
5. If it should become necessary to shut-down one engine, turn the rudder hydraulic power switch "off" before engine shut-down.

A.4 FLIGHT TESTS

NOTE: The principal objective is to log 10 hours of flight time on the AFCAS test installation.

FIRST FLIGHT

Maximum Altitude	20,000 Feet
Maximum Speed	250 KOAS

Perform the following maneuvers at 10,000 and 15,000 feet:

- Directional control check with rudder hydraulic power switch "off", \approx 200 KOAS, PLF (conduct remaining tests with rudder hydraulic power switch "on").
- Level flight MRT 250 K (Max.)
- Idle RPM descent \approx 250 KOAS
- 1/2 directional control, sideslip angle right and left @ 250 KOAS, PLF
- Full directional control, sideslip angle right and left @ 250 KOAS, PLF

SECOND FLIGHT

Maximum Altitude	30,000 feet
Maximum Speed	340 KOAS or .85 MN

*Perform the following maneuvers at 20,000 and 30,000 feet:

- Level flight MRT (no airspeed limit).
- Idle RPM descent \approx 250 KOAS
- 1/2 directional control sideslip angle right and left, @ V_{max} MRT. (This control input may be reduced at the pilot's discretion.)

*Do not operate speed brakes above 20,000 ft.

THIRD AND SUBSEQUENT FLIGHTS

Flight Envelope: Sea level to 30,000 feet
Air Speed: Up to 340 KOAS or 0.85 MN, whichever is less.
Flight Maneuvers: Optional

Flight Data

- Take-offs and Landings:

Record data continuously during first one minute of take-off and climb and continuously during the one minute prior to touchdown (Flights 1 and 2 only).

- Flight: Record a 15 second data burst once every 10 minutes (all flights).

- Maneuver: Record data continuously during maneuver.

Maneuvers for Pilot Comments

NOTE: Pilot to perform these at his discretion. Recorders not on.
Pilot to comment after landing.

- Apply small rudder inputs, note response and dead band.

- Apply pulse inputs, evaluate recentering, left and right.

- Make comparison of CBW "feel" with manual "feel".

- Comment on rudder pedal operation, i.e., no displacement (force only).

- The pilot is encouraged to perform any additional maneuvers that would provide worthwhile data.

Post Flight Checks

- Plot fluid temperature versus time curves from test data. (first two flights and last flight) (performed by Department 071)

- Look for any trends (increase) in pump case flow (Department 071).

- Remove pump case drain filter bowl (first two flights and after final flight). Deliver bowl, element and fluid to Department 071 for patch test.

- Take fluid sample for contamination check (first two flights and after final flight).

- De-brief pilot after each flight.

- Make decisions regarding changes or additional procedures for next flight.

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